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Profile derived fluxes above inhomogeneous terrain:
a numerical approach

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Profile derived fluxes above inhomogeneous terrain: a numerical approach

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WAGENINGEN

"If we knew all the laws of Nature, we should need only one fact, or the description of one actual phenomenon, to infer all the particular results at that point. Now we know only a few laws, and our result is vitiated, not, of course, by any confusion or irregularity in Nature, but by our ignorance of essential elements in the calculation. Our notions of law and harmony are commonly confined to those instances which we detect; but the harmony which results from a far greater number of seemingly conflicting, but really concurring, laws, which we have not detected, is still more wonderful. The particular laws are as our points of view, as, to the traveller, a mountain outline varies with every step, and it has an infinite number of profiles, though absolutely but one form. Even when cleft or bored through it is not comprehended in its entirety."

from: Walden

Henry David Thoreau
(1817-1862)

Aan mijn ouders
en aan Marja en Emily

Voorwoord

Het verrichte onderzoek, waarvan deze dissertatie een van de eindprodukten is, had een aanpak die omschreven wordt door de term modellering. Een dergelijke aanpak brengt met zich mee dat slechts een relatief gering aantal mensen naast de onderzoeker zelf een inbreng in het geheel heeft. Maar die inbreng zelf hoeft op zich niet gering te zijn.

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Een nieuw onderzoek moet door anderen bedacht, aangevraagd en verdedigd worden voordat de onderzoeker zelf nog maar iets heeft gedaan. Voor dit voorbereidende en essentiële werk ben ik vooral Frits Bottemanne zeer dankbaar.

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Bennekom, 15 oktober 1984.

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List of symbols

<u>Symbol</u>	<u>Description</u>	<u>S.I. unit</u>	<u>Introduced in:</u>
a	factor in Elliott's expression for the height of the internal boundary layer	-	3.2.2
a	constant in Mulhearn's model	-	3.3.1
a ₁	constant in Bradshaw's model	-	3.5.1
a ₁ , a ₂	constants used in the modeling of the pressure terms	-	A3.1
a _t	constant used in the modeling of the turbulent transport terms	-	4.1.2
b	constant used in the modeling of the destruction terms	-	4.1.2
c	constant used in the modeling of the dissipation terms	-	4.1.2
c ₁₃	constant used in the modeling of the pressure covariance	-	A2.1
C, C ₁ , C ₂	constants used in the modeling of the pressure covariance	-	A3.1
c _{pd}	specific heat of dry air at constant pressure	J kg ⁻¹ K ⁻¹	2.3
d(x)	height of the internal boundary layer	m	2.1
d ₁ , d ₃	constants used in the modeling of the destruction terms	-	A3.1
e	used in the turbulent kinetic energy defined by: TKE = $\overline{e^2}$	m ² s ⁻²	3.5.1
E	water vapour flux density	kg m ⁻² s ⁻¹	4.3.2
Ei, E ₁	exponential integral functions	-	3.3.1
f, F	universal functions in the models of Townsend and Mulhearn	-	3.3.1
f	Coriolis parameter	s ⁻¹	3.4.1
g _i	acceleration of gravity, g _i = (0,0,g)	ms ⁻²	2.3

<u>Symbol</u>	<u>Description</u>	<u>S.I. unit</u>	<u>Introduced in:</u>
g, G	universal functions in Mulhearn's model	-	3.3.1
G	Geostrophic wind speed	ms^{-1}	3.4.1
G	function in Bradshaw's model	-	3.5.1
G	soil heat flux density	Wm^{-2}	5.2
h_o	average height of roughness elements	m	6.3
h_{BR}	height at which the fluxes determined with the Bowen ratio method deviate 10% from the surface fluxes	m	6.2
H_α	scale height of the atmosphere	m	2.3
H	sensible heat flux	Wm^{-2}	4.3.2
I	vertical flux of horizontal momentum	$\text{kg m}^{-1} \text{s}^{-2}$	5.2
k	von Karman constant ($k = 0.41$)	-	3.2.1
k	thermal conductivity	$\text{Jm}^{-1} \text{s}^{-1} \text{K}^{-1}$	2.3
$K_{m,h,w}$	scalar eddy transfer coefficients for momentum, heat and water vapour	$\text{m}^2 \text{s}^{-1}$	3.2.1/5.2
k	turbulent kinetic energy $k = \overline{e^2}$	$\text{m}^2 \text{s}^{-2}$	3.5.1
ℓ_w	length scale of the vertical velocity	m	2.3
ℓ_α	length scale for the vertical variation of specific volume (α)	m	2.3
ℓ_H	length scale for the horizontal motion	m	2.3
ℓ	mixing length	m	3.2.1
ℓ_o	length scale in the models of Townsend and Mulhearn	m	3.3.1
L	length scale in Bradshaw's model	m	3.5.1
L	Obukhov length	m	5.2
ℓ_1, ℓ_e	length scales in the model of Huang and Nickerson	m	3.5.1

<u>Symbol</u>	<u>Description</u>	<u>S.I. unit</u>	<u>Introduced in:</u>
m	ratio of upstream and downstream roughness length $m = z_{01}/z_{02}$	-	3.2.2
M	$M = \ln(z_{01}/z_{02})$	-	3.2.2
\underline{P}	(instantaneous) pressure $\underline{P} = P_r + \tilde{P}$	Pa	2.3
\tilde{P}	perturbation pressure $\tilde{P} = P - P_r$	Pa	
P	production rate of turbulent kinetic energy	$m^2 s^{-3}$	3.5.1
$P_{1,2,3}$	terms in the transformation of the \overline{uw} equation	-	A5
\tilde{q}	specific humidity $\tilde{q} = Q - \bar{Q}$	-	2.3
q_*	specific humidity scale, $q_* = -\overline{wq}/u_*$	-	4.2.1
Q_s	saturation specific humidity	-	A3.1
R_o	surface relative humidity (defined at $z = z_o$)	-	A3.2
R_d	gas constant for dry air	$J kg^{-1} K^{-1}$	2.3
R_i	Richardson number	-	A3.1
R_n	net radiation	$W m^{-2}$	5.2
$r_{\theta q}$	correlation coefficient	-	5.4.2
r_a	aerodynamic resistance	sm^{-1}	A4
r_s	surface resistance	sm^{-1}	A4
s	slope of the saturation specific humidity curve	K^{-1}	A4
S	specific entropy	$J kg^{-1} K^{-1}$	2.3
t	time	s	2.3
\underline{T}	(instantaneous) temperature $\underline{T} = T_r + \tilde{T}$	K	2.3
\tilde{T}	perturbation temperature	K	2.3
T_*	temperature scale, $T_* = \overline{w\theta}/u_*$	K	4.1.2
\underline{u}_i	(instantaneous) Eulerian velocity with components $(\underline{u}, \underline{v}, \underline{w})$ and $\underline{u}_i = u_{i,r} + \tilde{u}_i$	ms^{-1}	2.3
\tilde{u}_i	perturbation velocity $\tilde{u}_i = U_i + u_i$	ms^{-1}	2.3
u_*	friction velocity, $u_* = (\tau_o/\rho_o)^{1/2}$	ms^{-1}	A2

<u>Symbol</u>	<u>Description</u>	<u>S.I. unit</u>	<u>Introduced in:</u>
u_0	velocity scale in Townsend's model	ms^{-1}	3.3.1
x_i	position vector with components (x, y, z)	m	2.1
x_{\max}	downstream distance where $d(x) = z_{\max}$	m	2.1
z_0	surface roughness length	m	2.1
z_I	PBL height	m	A3.2
z_{\max}	upper grid level	m	A3.2
z_1, z_2	lowest two grid levels	m	A3.2

Greek symbols

α	specific volume	$\text{m}^3 \text{kg}^{-1}$	2.3
β	Bowen ratio (gradient)	-	5.2
β^*	Bowen ratio (flux)	-	5.2
γ	$\gamma = c_p / \lambda$	K^{-1}	A4
Γ_d	dry adiabatic lapse rate	Km^{-1}	2.3
$\delta(x)$	height of the IAL	m	2.1
δ_{ij}	Kronecker delta	-	2.3
ϵ	dissipation rate of turbulent kinetic energy	$\text{m}^2 \text{s}^{-3}$	3.5.1
ϵ^*	ratio of the molecular masses of water vapour and dry air	-	2.3
ϵ_{ijk}	permutation tensor (Levi Civita)	-	2.3
η	dimensionless height, $\eta = z / z_0$	-	3.3.1
θ	(instantaneous) potential temperature, $\theta = \theta_r + \tilde{\theta}$	K	2.3
$\tilde{\theta}$	perturbation potential temperature, $\tilde{\theta} = \theta - \theta_r$	K	2.3
κ	constant in the approximation of τ	-	A5
κ	ratio, $\kappa = R_d / c_{pd}$	-	2.3
κ_H	thermal diffusivity, $\kappa_H = k / \rho c_p$	$\text{m}^2 \text{s}^{-1}$	2.3
κ_v	diffusivity of water vapour	$\text{m}^2 \text{s}^{-1}$	2.3

<u>Symbol</u>	<u>Description</u>	<u>S.I. unit</u>	<u>Introduced in:</u>
λ	length scale used in Taylor's model	m	3.4.1
λ	latent heat of vaporization of water	J kg ⁻¹	4.3.2
μ	molecular viscosity of air	kg m ⁻¹ s ⁻¹	2.3
ν	kinematic viscosity of air	m ² s ⁻¹	2.3
ρ	(instantaneous) density $\rho = \rho_r + \tilde{\rho}$	kg m ⁻³	2.3
$\tilde{\rho}$	perturbation density	kg m ⁻³	2.3
σ_u	variance of the horizontal velocity fluctuation, $\sigma_u^2 = \overline{u^2}$	ms ⁻¹	4.3.1
τ	shear stress	Nm ⁻²	3.2.1
τ	turbulent time scale	s	4.1.2
$\phi_{m,u,w}$	dimensionless functions in the atmosphere surface layer for momentum, heat and water vapour	-	3.2.3/5.2
ω	frequency	s ⁻¹	2.3
Ω_i	earth rotation vector	s ⁻¹	2.3

Abbreviations

ABL	Atmospheric Boundary Layer	2.1
ASL	Atmospheric Surface Layer	2.1
IAL	Internal Adapted Layer	2.1
IBL	Internal Boundary Layer	2.1
PBL	Planetary Boundary Layer	2.1
TKE	Turbulent Kinetic Energy	3.1

1 Introduction

1.1 SCOPE AND GOALS

This study deals with the problems which arise when one tries to measure some of the components of the energy balance at the earth's surface. A simplified picture of what happens at the interface of the two distinctly different media air and earth is the following. The net amount of energy supplied to the earth's surface, after various physical processes (absorption, reflection, scattering etc.) have taken place, is used to heat the ground and the atmosphere just above the surface and is also used to evaporate the water at or near the surface. In this simplification we have disregarded the energy absorbed by vegetation and other, usually small, terms of the energy balance. Thus energy is transported from the surface into the ground and the atmosphere.

The transport of a quantity per unit area and unit time is also referred to as a flux density. Heating of the atmosphere and evaporation are called the sensible and latent vertical heat flux density, respectively. These flux densities can be directed upwards (into the atmosphere) or downwards (to the earth), depending on the stratification of the atmosphere just above the surface. These two fluxes are always accompanied by a vertical flux of horizontal momentum which is directed downwards.

The assessment of the magnitude of either three, or all of these flux densities is of crucial importance to a number of practical questions in meteorology, agriculture, hydrology and related fields. In meteorology the energy supplied to the earth's surface is one of the main processes driving the atmosphere. Because of the large latent heat of vaporization of water, large amounts of energy can be redistributed through the atmosphere. At a much smaller scale the energy balance is an important boundary condition for models of the atmospheric boundary layer for air pollution models and studies of air-sea interaction. In hydrology the

fluxes of sensible and latent heat are important for the estimation of the water balance of inland lakes, the evaporation from vegetation systems etc.. For problems concerned with e.g. the water setup by wind in a network of channels the momentum flux must be known. Finally, in agriculture the estimation of the fluxes of momentum, latent and sensible heat are needed when one is interested in matters concerned with wind erosion, crop yield, plant diseases, pest control and the like.

Proper measurement of the flux densities is beset with many difficulties and uncertainties. It would of course be ideal if one is able to measure them directly at the surface using the conservation of mass principle and other conservation equations. Instruments performing these tasks have indeed been designed (drag plates, heat flux plates, lysimeters) and are in use, but they are not easily installed and cumbersome to use. The only possibility left is the indirect measurement in the atmosphere and in the ground. Net radiation is relatively easily and accurately measured but the other terms of the energy balance still provide some severe difficulties. Measurement of the soil heat flux density is difficult because of the large spatial variation of specific heat, water content and thermal conductivity of the soil. Also distillation processes and other latent heat fluxes in the soil are very difficult to assess quantitatively. Furthermore, the complexities introduced by water transport through the roots of vegetation makes this approach to flux measurement unattractive, thus forcing us to look at the only other alternative: flux measurements in the atmosphere.

In the atmosphere we are dealing with a medium that is highly turbulent most of the time and which will produce large fluctuations of the value of quantities measured in it. The fluctuating output of instruments placed in the turbulent atmosphere introduces the necessity of determining an average of that output over a given space or time interval. After establishing this average, the instantaneous value of a quantity consists of its average value plus the fluctuating part. Estimation of the

flux densities of momentum and those of sensible and latent heat can then roughly be divided into two methods.

The first possibility is the measurement of profiles of the mean values of wind speed, temperature and humidity. Using the concept of molecular diffusion, where the flux density of a quantity and its gradient are connected by means of a molecular diffusion coefficient, an analogy can be drawn between molecular and turbulent diffusion. Defining a so-called eddy diffusivity we relate the turbulent flux density of a quantity with the gradient of its mean value. This enables the estimation of the turbulent flux density, once we have a good approximation of both the eddy diffusivity distribution and the gradient of the mean value.

The second possibility is the direct measurement of turbulent quantities. If the proper turbulent quantities are correlated with each other the turbulent flux can be determined directly (eddy correlation technique) or indirectly (dissipation techniques).

The advantage of the profile method over the correlation and dissipation methods is that averages are relatively easy to obtain, while it is much more difficult to measure the correlation of two fluctuating quantities. Its disadvantage is that an accurate distribution of the eddy diffusivity is not easy to establish, or must be assumed, while it still remains questionable if gradient diffusion is a correct mechanism in all possible situations that may occur in the atmospheric boundary layer (see Corrsin, 1974). Recent and current research reveals that turbulent transport is not a smooth continuous process. It is a process which consists of relatively large periods of time with hardly any transport interspersed with relatively short periods of time with vigorous mixing and very large transport. Going down to molecular time and length scales we find that molecular diffusion is no continuous process either. We remedy this by considering only time and length scales which are several orders of magnitude larger than the molecular scales, in this way ensuring the validity of the continuum hypothesis. Turbulent diffusion in the

atmosphere comprises time and length scales which are very much larger than the molecular ones. In fact they are large enough to make an analogy with molecular diffusion hazardous, especially when large spatial and/or time variations occur.

The direct measurement of correlations of turbulent quantities avoids the use and the problems of the eddy diffusivity concept. But a serious disadvantage of this method is that the fluctuation measurement sensors require a fast response. Because all frequencies occurring in the spectrum of turbulence contribute to the turbulent flux, sensors have to be fast enough to register even the highest of those frequencies. This poses a very high demand on sensor performance, especially when installed close to the surface. Finally, a difficulty connected with both methods is the problem of defining a proper length of time over which the average values should be determined.

All the above mentioned problems connected with the two methods have of course been studied extensively and considerable progress both in theoretical and technical respect has been made. But there are a few basic assumptions which form the cornerstone of both methods, and which also need careful consideration. These assumptions may all be condensed in the following crucial starting point of the theories underlying the two methods just described. We assume that the surface layer of the atmosphere is in equilibrium with the underlying surface. Equilibrium means that all the characteristic variables of the atmospheric surface layer will not change when the flow continues its course over the earth's surface.

Amongst other things, which will be disregarded for the moment, equilibrium implies that fluxes will be constant with height in the atmospheric surface layer and are equal to the fluxes at the surface. This means that instruments may be placed at any height in this layer, excluding the demands on sensor performance from our consideration for the time being. Equilibrium also implies that a relationship between the vertical distribution of the

average value of a quantity and its vertical flux density can be established, which means that the fluxes can be derived from measured profiles. Finally, equilibrium also implies that several terms in the equations for the second moments are small enough to be neglected, which implies that production and dissipation (or destruction) terms balance each other. This is a necessary condition for the application of the dissipation technique.

It is obvious from the definition of equilibrium given above that true equilibrium will never occur in practice because external conditions will change both in time and space. But in some cases a reasonable approximation of the state of equilibrium can be reached. This is possible if weather conditions do not change during the averaging period and if the measuring site is chosen in such a way to ensure a large upstream fetch of uniform terrain. It is not surprising therefore, that for the well-known experiments performed in Kansas (Izumi, 1971) and Minnesota (Izumi and Caughey, 1976) great care was taken in choosing the location for measurements in order to avoid advection effects created by a non-uniform upstream fetch as much as possible. These are the experiments, amongst others, that have provided the data from which numerous empirical flux-profile relations have been determined and from which many empirical constants have been established in the theory of atmospheric turbulence. This means that existing formulae for the determination of turbulent fluxes are, strictly speaking, only valid in equilibrium situations. A proper choice of the measuring period and of the averaging time will ensure almost constant weather conditions, but extensive homogeneous terrains are for most practical situations not easily available. So the question arises: how large are the errors introduced in the flux determination, using standard methods, by the non-uniformity of the upstream terrain, and is there a possibility to make corrections for the deviations generated by the inhomogeneities? These questions form the starting point of all that the present study encompasses.

1.2 METHODS

Any problem in physics can, generally speaking, be approached in two distinctly different ways: theoretically and experimentally. The first option is often beset with serious mathematical difficulties. In the case of atmospheric physics and turbulence these are very severe and drastic simplifications have to be made in order to make the problem theoretically tractable. The question then arises if enough of the essential processes and characteristics of the system under consideration are preserved in order to get sensible and non-trivial solutions. The second option is beset with many practical difficulties such as the quality of equipment, weather conditions, choice of location etc. Furthermore, it will lead to the setup of a measuring campaign which generally is no mean feat, even when measuring only average quantities, let alone turbulence quantities.

A separate branch of the theoretical treatment of problems in science was developed with the introduction and large scale use of fast digital computers and the application of numerical mathematics in the fifties and early sixties. Modeling intricate physical processes and interactions using these computers belongs, in principle, to the theoretical treatment option but is distinctly different from it in some respects. But on the other hand it also has some characteristics which are typical features of the experimental treatment option. It does not altogether belong to the theoretical analysis since the mathematical theory for solving nonlinear partial differential equations is still inadequate and a complete solution of the Navier-Stokes equation which describes the dynamics of turbulence, is a very long way off indeed. The experimental character of the modeling option is given by the fact that every run of the model on a computer looks like the performance of a physical experiment. Thus modeling forms a synthesis of the traditional two options and it soon emerged as a powerful tool in solving problems in many fields in science.

The advantages of this third option are obvious. No expensive measuring campaign is necessary, only a modern computer with its usual facilities is needed. With the theoretical analysis it shares the advantage of one being able to choose, within limits, parameters of, and variables in the atmospheric surface layer such as the Obukhov length, the wind direction, friction velocity, surface roughness length, temperature and humidity profiles. In contrast, for experiments performed in the atmosphere these quantities depend on large scale weather systems, terrain location, time of the season etc. Finally, the degree of simplification of the governing equations will often be less than the one necessary to obtain analytical solutions. Thus one is able to tackle a whole range of problems with numerical modeling techniques, which are far from being treated with analytical mathematical methods. But the third option still shares a number of disadvantages of the first option. It is e.g. not always trivial to see if all physical processes and interactions are properly described by the set of equations in use. Also, these equations have to be simplified in order to arrive at a numerically solvable set. It is difficult to assess how much information is lost by these simplifications. The system of equations also must form a closed set, i.e. the number of equations must equal the number of unknowns. This creates an enormous problem in atmospheric turbulence, generating a whole new field of science devoted to this subject.

There are also disadvantages typical for the third option which are almost absent in the other two options. E.g. the lesser the degree of simplification the larger the number of equations and, consequently the larger the number of closure assumptions. These assumptions each introduce a number of constants which have to be determined one way or another. Also, there will always be processes on scales smaller than the computational grid but which do interact with the larger features of the flow. These subgrid-scale processes have to be taken into account somehow, but more often than not they are simply neglected. Also, a numerical solution will never be able to provide the physical insight

into the problem as given by an analytical one, because it does not produce any functional relationship. This is the most important disadvantage.

The above considerations must be kept in mind when choosing a way to treat the aforementioned questions. But also factors like feasibility must be taken into account. It was decided that the best strategy for this study was formed by the possibilities inherent to the third option.

Chapter 2 gives an analysis of the problem we are facing. It presents an outline of the basic laws of physics which are of interest here, as well as the simplifications which must be made. In Chapter 3 an account is given of the various models and methods which have been used in the past to study the effects created by a change in surface conditions, concentrating mainly on a change in surface roughness. In Chapter 4 a critical examination is made of the numerical model employed in the present study. It contains a summary of all the restrictions and drawbacks inherent to the use of this specific model. In Chapter 5 the standard flux profile methods are compared with flux profile relations derived from the modeled second moment equations for homogeneous situations. Chapter 6 treats the errors introduced by terrain inhomogeneities in the profile methods of flux determination. Chapter 7, finally, contains the conclusions of the present study. In it recommendations are given for further study and possible treatment of the problem with an experiment which, as usual, will have to give an answer to the question if all that is stated in this study has any truth in it.

2 Analysis of the problem

2.1 GENERAL FEATURES AND RELEVANCE FOR FLUX DETERMINATION TECHNIQUES

The thin layer of air (0.1 - 0.3% of the radius of the earth) that envelops the surface of our planet is called the atmosphere. It is kept in motion primarily by the differences in local heating of the earth's surface by the sun. At or near the interface of the atmosphere and the surface of the earth a continuous interaction takes place. This interaction ranges from the large scales (oceans, continents, mountain ranges) to the very small scales (pebbles, leaves of grass, grains of sand), in fact down to the molecular scale. As a consequence, the atmospheric motions comprise a large spectrum of length and time scales (Figure 2.1).

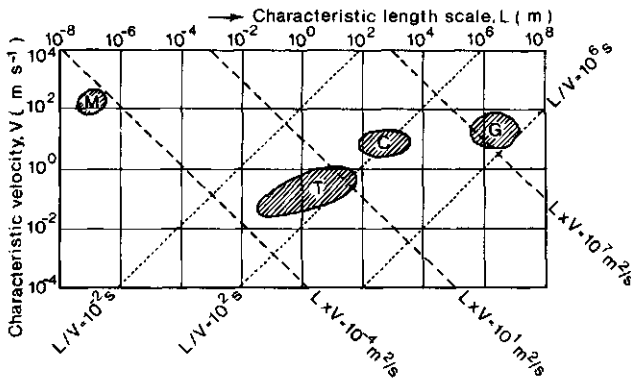


Figure 2.1 Classification of length and time scales of atmospheric motions

The lowest layer of the atmosphere is affected directly by the nature and properties of the surface itself. In this part of the atmosphere which is called the atmospheric boundary layer (ABL) turbulence plays an important role.

In the upper part of the ABL the Coriolis force contributes substantially to the dynamics of it. The lowest part of the ABL is called the atmosphere surface layer (ASL) (Figure 2.2). Within this layer the turbulent fluxes do not change significantly with height if the surface of the earth is horizontally homogeneous.

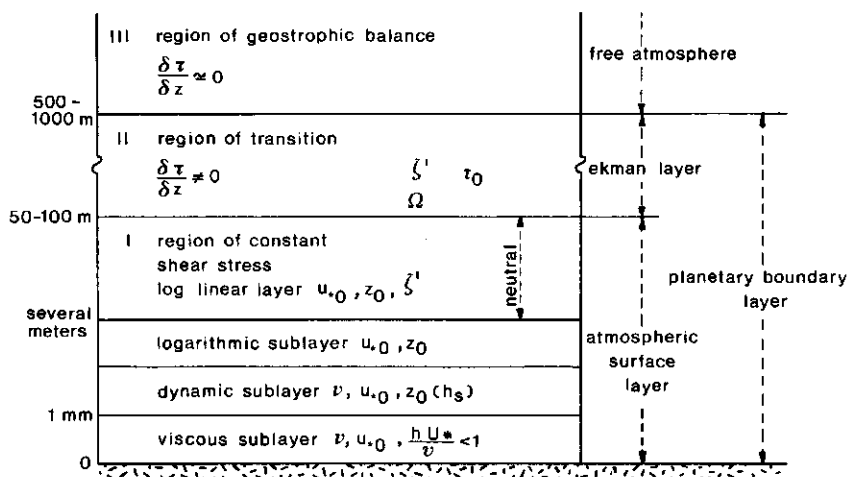


Figure 2.2 Vertical structure of the atmospheric boundary layer (after: Huang and Nickerson, 1972)

Within the ASL also the influence of the Coriolis force is negligible. For that reason the direction of the wind remains approximately constant with height. This means that our analysis of the surface layer is a two-dimensional one. We align the horizontal axis of the coordinate system along the direction of the mean wind vector.

In its course over the surface of the earth the ASL interacts, as was mentioned above, with the underlying surface of the earth. This interaction process is maintained by means of the turbulent fluxes of momentum, heat, water vapour and other, usually negligible, constituents of the atmosphere. If the surface is homo-

geneous over a sufficiently large distance and if external weather conditions (e.g. the overall pressure gradient, the static stability etc.) are constant, a state of equilibrium in the ASL will be reached. Disregarding other possibilities, the state of equilibrium of the ASL will be defined here as that situation where mean turbulent fluxes are constant with height, or very nearly so, and the profiles of mean quantities such as temperature do not change downstream (see also section 1.1).

If an air mass in such an equilibrium encounters a stepwise change in surface conditions it will have to adjust to these new conditions. Eventually, under the same restrictions as mentioned before, the surface layer will reach a new state of equilibrium, determined by the new surface conditions. This problem belongs to a class of problems in turbulent shear flow theory, viz. that of relaxing flows. We state that a relaxing flow is one in which a turbulent shear flow changes from one state of equilibrium to another state of equilibrium by virtue of changes in external conditions. These changes in external conditions are represented by changes in boundary conditions of the theoretical models that describe the physics of the atmosphere subjected to these discontinuities. The change may be limited to a single quantity such as surface temperature (e.g. Johnson, 1955; Vugts and Businger, 1977), surface humidity (Weisman, 1975; McNaughton, 1976), surface heat flux (Antonia et al., 1977), surface roughness (Elliott, 1958; etc.) or surface topography (de Bray, 1973; Bowen and Lindley, 1977; Dawkins and Davies, 1981). But, usually two or more of these quantities will change simultaneously (Rider et al., 1963; Dyer and Crawford 1965; Tieleman and Derrington, 1977).

The ASL which is subjected to a stepwise change in surface conditions will be transformed. As was pointed out in the previous chapter, this transformation need not be a smooth continuous process. In fact, experiments show that turbulent transport for the greater part results from the action of turbulent structures. These turbulent structures represent elongated, spatially coherent, organized flow motions (Schols, 1984). Keeping this in mind we merely hope that with the practice of using averages the effects of the individual turbulent structures will be

smoothed by the averaging procedure. On the average, the surface layer will start to adjust to the new surface conditions in its lowest layers close to the ground. Through turbulent transport the change in surface conditions will affect layers of increasing height as the air travels downstream. Thus at a given horizontal distance (x) downstream of the discontinuity this influence will change the atmospheric surface layer up to a certain height ($d(x)$). This height increases with x . Above this height no change will be noticeable. The region of the atmospheric surface layer between $z = 0$ and $z = d(x)$ is called the Internal Boundary Layer (IBL) (Figure 2.3). Below $z = d(x)$ the profile of every

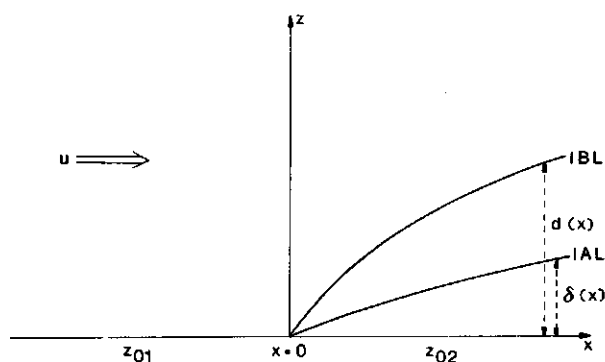


Figure 2.3 Definition sketch of a change in surface conditions

variable will deviate from its original equilibrium distribution. Above $z = d(x)$ every profile remains unchanged. Of course, the definition of these layers is not rigorous for it results from the use of statistical methods. There are large eddies and turbulent structures in the ASL which momentarily are able to transport parcels of air already affected by the new surface to heights far greater than d .

As the air travels further along, the lowest part of the IBL will become locally adapted to the new surface. This means that up to a certain height $z = \delta(x) < d(x)$ fluxes will be constant with height though they are still changing with downstream distance. Thus no new equilibrium has yet been reached, but the local flux-

es in the lowest part of the IBL are constant up to $z = \delta(x)$. We will call the region the Internal Adapted Layer (IAL) (Figure 2.3).

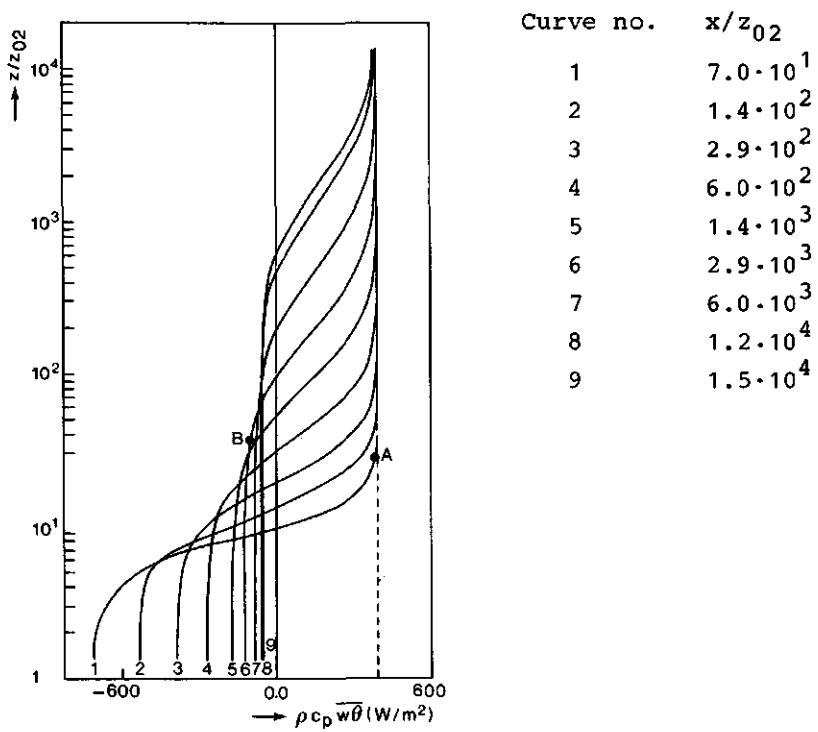


Figure 2.4 Profiles of the vertical heat flux after a change in surface conditions

Figure 2.4 serves to illustrate the above. It presents the stream-wise variation of the turbulent heat flux profile $\rho c_p \overline{w\theta}(x,z)$ after a change from a hot, dry, smooth surface to a cool, wet, rough surface as calculated with the model of Rao et al. (1974b). Starting from a surface layer which is completely in equilibrium with the underlying terrain, we see that in that case (indicated by the dashed line) the vertical turbulent heat flux is constant with height. The value of $\rho c_p \overline{w\theta}$ is positive, denoting an upward heat flux. The other curves represent the heat flux profile for various distances downstream. Just after the step change the temperature of the surface drops, generating a downward, negative

heat flux. Only the lowest region of the surface layer is affected up to e.g. point A of curve 1. This point A corresponds to the local height of the IBL. Gradually a region develops where $\rho c_p \overline{w\theta}$ is constant with height (e.g. up to point B of curve 6) though $\rho c_p \overline{w\theta}$ still depends on the horizontal distance (x). Point B corresponds to the local height of the IAL.

Figure 2.4 clearly shows that the magnitude of the vertical heat flux critically depends on the location. Thus the determination of flux densities above a non-homogeneous terrain is beset with many difficulties and uncertainties. Depending on the method one uses to determine those flux densities, this may lead to considerable errors. In practice, one is usually interested in the flux density from a given stretch of nearly homogeneous field irrespective of the characteristics of the surrounding terrains. As the horizontal dimensions of the field of interest are always finite, problems often arise concerning the use of flux determination formulae, interpretation of the results of measurements, sensor location etc.

2.2 GOAL OF THE PRESENT STUDY

In this study it is our intention to analyse the effects created by one, or more, changes in surface conditions on the structure of the atmospheric surface layer in order to:

1. estimate the error introduced in the standard flux determination techniques when they are applied in these conditions, and
2. provide simple techniques for estimating these errors using a minimum number of data concerning sensor location, surrounding terrain(s) etc.

To achieve these goals we use the data provided by the model described in Chapter 4, as if they were actually determined by experiment. We implicitly assume that the model represents the actual situation fairly well. Using the data from the model we are able to perform the above mentioned analyses and draw our conclusions on the performance of the standard flux determination techniques in non-homogeneous situations.

2.3 GOVERNING EQUATIONS

Whatever happens to the air above the changing surface conditions, it will always have to obey the conservation equations of mass, momentum and specific entropy. These equations have been derived many times (e.g. Hinze, 1959; Lumley and Panofsky, 1964; Monin and Yaglom, 1971; Tennekes and Lumley, 1972), hence we confine ourselves to the main lines. For details we refer to the textbooks mentioned. Notice will be taken, however, of the approximations and assumptions involved in this derivation.

We start with the conservation equations for an ideal, viscous, compressible, Newtonian fluid in a uniform gravitational field and in a rotating system

$$\rho \left(\frac{\partial u_i}{\partial t} + u_j \frac{\partial u_i}{\partial x_j} \right) = - \frac{\partial P}{\partial x_i} - \rho g_i + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) - \frac{2}{3} \mu \frac{\partial u_k}{\partial x_k} \delta_{ij} \right] - 2\Omega_j \epsilon_{ijk} u_k \rho, \quad (2.1)$$

$$\frac{\partial \rho}{\partial t} + \rho \frac{\partial u_i}{\partial x_i} + u_i \frac{\partial \rho}{\partial x_i} = 0, \quad (2.2)$$

$$\rho T \left(\frac{\partial S}{\partial t} + u_j \frac{\partial S}{\partial x_j} \right) = \frac{\partial u_i}{\partial x_j} \left[\mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) - \frac{2}{3} \mu \frac{\partial u_k}{\partial x_k} \delta_{ij} \right] + \frac{\partial}{\partial x_j} (k \frac{\partial T}{\partial x_j}), \quad (2.3)$$

$$P = \rho R_d T. \quad (2.4)$$

These equations represent the conservation equations of momentum, mass and specific entropy, respectively and the equation of state. S is specific entropy, ρ density, T temperature, u_i velocity, μ viscosity, k thermal conductivity, δ_{ij} is Kronecker's delta and ϵ_{ijk} is the alternating tensor (Levi Civita). In addition Einstein's summation convention is used. We assume that the gradients of P , ρ and T are small enough to ensure that μ and k are approximately constant throughout the fluid. We define a refer-

ence state of the atmosphere, denoted by the subscript "r" such that the instantaneous value \tilde{m} of the variable m ($m = \rho, T, P$ or u_i) is decomposed in

$$\tilde{\rho} = \rho_r + \tilde{\rho}, \quad \tilde{P} = P_r + \tilde{P}, \quad \tilde{T} = T_r + \tilde{T}, \quad \tilde{u}_i = u_{i,r} + \tilde{u}_i, \quad (2.5)$$

where \tilde{m} is a perturbation of this reference state.

Following e.g. Dutton and Fichtl (1969) the equation of state is expanded around this reference state, this yields

$$\frac{\tilde{\rho} - \rho_r}{\rho_r} = \frac{\tilde{P} - P_r}{P_r} - \frac{\tilde{T} - T_r}{T_r} \quad \text{or} \quad \frac{\tilde{\rho}}{\rho_r} = \frac{\tilde{P}}{P_r} - \frac{\tilde{T}}{T_r}, \quad (2.6)$$

where higher order derivatives are ignored. This is only permissible if the deviations $\tilde{\rho}$, \tilde{T} and \tilde{P} are small in comparison with their respective reference state value,

$$\text{hence } \left| \frac{\tilde{\rho}}{\rho_r} \right| \ll 1, \quad \left| \frac{\tilde{P}}{P_r} \right| \ll 1 \text{ and } \left| \frac{\tilde{T}}{T_r} \right| \ll 1.$$

If for the reference state, we assume that the effects of radiation, viscosity and heat conduction are not important and that the fluid is in steady motion, hence e.g. $u_{i,r} = U_r(z)$, the equations of motion (2.1) imply that

$$\frac{\partial P_r}{\partial x} + \frac{\partial P_r}{\partial y} = 0 \quad \text{and} \quad \frac{\partial P_r}{\partial z} = -\rho_r g \quad (2.7)$$

which, after some manipulation gives

$$\frac{\partial T_r}{\partial x} + \frac{\partial T_r}{\partial y} = 0. \quad (2.8)$$

This means that the vertical variation of $U_r(z)$, $\rho_r(z)$ and $T_r(z)$ is still free to choose.

In the reference state the atmosphere is supposed to have an

adiabatic temperature profile

$$\frac{\partial T_r}{\partial z} = - \frac{g}{c_{pd}} = - \Gamma_d . \quad (2.9)$$

Now, Eqs. (2.7) and (2.9) together with the equation of state

$$P_r = R_d T_r \rho_r , \quad (2.10)$$

completely determine the vertical distribution of P_r , T_r and ρ_r .

The continuity equation (2.2) may be expanded in the form

$$\frac{\partial \tilde{\rho}}{\partial t} + (\tilde{u} \frac{\partial \tilde{\rho}}{\partial x} + \tilde{v} \frac{\partial \tilde{\rho}}{\partial y}) + \tilde{w} \frac{\partial \tilde{\rho}}{\partial z} + \tilde{w} \frac{\partial \rho_r}{\partial z} = - \frac{1}{\rho_r} (\frac{\partial \tilde{u}}{\partial x} + \frac{\partial \tilde{v}}{\partial y}) - \frac{1}{\rho_r} \frac{\partial \tilde{w}}{\partial z} , \quad (2.11)$$

where $|\tilde{\rho}/\rho_r| \ll 1$ has been used on the right hand side. Replacing all variables by a Fourier representation and estimating the orders of magnitude of all variables and their derivatives Dutton and Fichtl (1969) were able to show that in Eq. (2.11) only the terms on the right hand side remain, hence

$$\frac{\partial \tilde{u}_i}{\partial x_i} = 0 , \quad (2.12)$$

if the following conditions are met

$$\begin{aligned} \text{(i)} \quad & \omega^2 \ll g/\ell_w , \\ \text{(ii)} \quad & |\hat{v}_H/\hat{w}| \leq \ell_H/2\pi\ell_w , \\ \text{(iii)} \quad & \ell_w \ll H_\alpha , \\ \text{(iv)} \quad & \ell_w \sim \ell_\alpha . \end{aligned} \quad (2.13)$$

Here ω is the frequency in the time dependent part of the Fourier representation ℓ_w , ℓ_H and ℓ_α are length scales for the vertical and horizontal motion and the vertical variation of specific volume ($\alpha = \rho^{-1}$) respectively. H_α is a scale height defined by

$H_\alpha^{-1} = \frac{1}{\alpha_r} \frac{\partial \alpha_r}{\partial z}$. Because (2.13) restricts the permissible vertical scales to heights much smaller than H_α this case is known as "shallow convection". For shallow convection, this approximation allows the fluid to be treated as incompressible.

Moreover, using condition (iii) of Eq. (2.13), it is possible to show that the pressure perturbation term in Eq. (2.6) can be ignored in the case of shallow convection, hence this equation reduces to

$$\frac{\tilde{\rho}}{\rho_r} = - \frac{\tilde{T}}{T_r} . \quad (2.14)$$

Next we turn to the equations of motion (2.1). We divide (2.1) by ρ_r and substitute the perturbation forms (2.5). The nonviscous vertical component terms on the right hand side of (2.1) then read

$$- \frac{1}{\rho_r} \frac{\partial \tilde{P}}{\partial z} - g = - \frac{1}{\rho_r} \frac{\partial \tilde{P}}{\partial z} - \frac{\tilde{\rho}}{\rho_r} g ,$$

where $|\frac{\tilde{\rho}}{\rho_r}| \ll 1$ has been used, except in the gravitation term, where it cannot be ignored (Boussinesq approximation). The kinematic viscosity is defined by $\nu = \nu_r = \mu/\rho_r$. If we furthermore use (2.12) and (2.15) the equations of motion become

$$\frac{\partial \tilde{u}_i}{\partial t} + \tilde{u}_j \frac{\partial \tilde{u}_i}{\partial x_j} = - \frac{1}{\rho_r} \frac{\partial \tilde{P}}{\partial x_i} + \frac{\tilde{T}}{T_r} g_i + \nu \frac{\partial^2 \tilde{u}_i}{\partial x_j \partial x_j} - 2\Omega_j \epsilon_{ijk} \tilde{u}_k . \quad (2.15)$$

The entropy equation (2.3) can be transformed in an equation in terms of the temperature (see Appendix 1):

$$\frac{\partial \tilde{T}}{\partial t} + \tilde{u}_j \frac{\partial \tilde{T}}{\partial x_j} = \kappa_H \frac{\partial^2 \tilde{T}}{\partial x_j \partial x_j} , \quad (2.16)$$

where $\kappa_H = k/\rho_r c_p$ is the thermal diffusivity.

As the latent heat flux will also play an important role in the

present study we must supplement the above equations with one representing the variation of specific humidity. To a good approximation, water vapour can be considered as a passive contaminant. We find, analogous to (2.16)

$$\frac{\partial \tilde{q}}{\partial t} + \tilde{u}_j \frac{\partial \tilde{q}}{\partial x_j} = \kappa_v \frac{\partial^2 \tilde{q}}{\partial x_j \partial x_j}, \quad (2.17)$$

where \tilde{q} is the specific humidity and κ_v is the diffusivity of water vapour. To account for the effect of the variable water vapour contents of the atmosphere on the density, we introduce the virtual temperature T_v which is defined by

$$T_v = (1 + q(\frac{1}{\epsilon} - 1))T \approx (1 + 0.608q)T. \quad (2.18)$$

This correction for moisture is of importance only in the buoyancy term in the equations of motion. In the reference state of the atmosphere we assume that no water vapour is present hence

$$T_v = T_{vr} + \tilde{T}_v = T_r + \tilde{T}_v. \quad (2.19)$$

Finally, we introduce the potential temperature which will be used from now on instead of the real temperature

$$\theta = T(P_0/P)^\kappa, \quad (2.20)$$

where P_0 usually is taken equal to $1.0 \cdot 10^5$ Pa and

$\kappa = R_d (1 - 0.23 q)/c_{pd}$ which may be simplified to $\kappa = R_d/c_{pd}$.

We split θ into

$$\theta = \theta_r + \tilde{\theta}, \quad (2.21)$$

where, according to (2.9) $\partial \theta_r / \partial z = 0$. In the atmospheric surface layer we may assume to a good approximation (within 1% at sea level) $\theta \approx T$ and $\tilde{\theta} \approx \tilde{T}$.

The set of equations now consists of Eqs. (2.12), (2.17) and

$$\frac{\partial \tilde{u}_i}{\partial t} + \tilde{u}_j \frac{\partial \tilde{u}_i}{\partial x_j} = - \frac{1}{\rho_r} \frac{\partial \tilde{P}}{\partial x_i} + \frac{\tilde{\theta}}{T_r} g_i + v \frac{\partial^2 \tilde{u}_i}{\partial x_j \partial x_j} - 2\Omega_j \epsilon_{ijk} \tilde{u}_k, \quad (2.22)$$

$$\frac{\partial \tilde{\theta}}{\partial t} + \tilde{u}_j \frac{\partial \tilde{\theta}}{\partial x_j} = \kappa_H \frac{\partial^2 \tilde{\theta}}{\partial x_j \partial x_j}. \quad (2.23)$$

Next we assume that the air motions can be separated into a slowly varying mean flow and a rapidly varying turbulent component. Furthermore we assume that the flow is ergodic, which means that ensemble averages may be replaced by time averages. Thus we arrive at Reynolds' convention,

$$\tilde{u}_i = U_i + u_i, \quad \tilde{\theta} = \theta + \theta, \quad \tilde{P} = P + p, \quad \tilde{q} = Q + q, \quad (2.24a)$$

where $\overline{u_i} = 0$, $\overline{p} = 0$ etc., $\overline{\tilde{u}_i \tilde{\theta}} = \theta U_i + \overline{\theta u_i}$ etc. and

$$\overline{\theta \tilde{u}_i} = \theta U_i \text{ etc.} \quad (2.24b)$$

Applying (2.24a) and (2.24b) to Eqs. (2.12), (2.17), (2.22) and (2.23) and averaging these equations (see Appendix 1) the equations for the mean flow read

$$\frac{\partial U_i}{\partial x_i} = 0, \quad (2.25)$$

$$U_j \frac{\partial U_i}{\partial x_j} = - \frac{1}{\rho_r} \frac{\partial P}{\partial x_i} + g_i \frac{\theta}{T_r} - 2\Omega_j \epsilon_{ijk} U_k + v \frac{\partial^2 U_i}{\partial x_k \partial x_k} +$$

$$- \frac{\partial}{\partial x_j} (\overline{u_i u_j}), \quad (2.26)$$

$$U_j \frac{\partial \theta}{\partial x_j} = \kappa_H \frac{\partial^2 \theta}{\partial x_k \partial x_k} - \frac{\partial}{\partial x_j} (\overline{u_j \theta}), \quad (2.27)$$

$$U_j \frac{\partial Q}{\partial x_j} = \kappa_v \frac{\partial^2 Q}{\partial x_k \partial x_k} - \frac{\partial}{\partial x_j} (\overline{u_j q}). \quad (2.28)$$

As long as the viscous sublayer that coats the surface is excluded from our considerations, the molecular diffusion terms in (2.26)-(2.28) can be neglected. We furthermore consider the situation where a fully turbulent steady air flow is at right angles to the surface discontinuity. This means that a restriction is made to a two-dimensional case. These simplifications of Eqs. (2.25)-(2.28) result in the following set of equations,

$$\frac{\partial U}{\partial x} + \frac{\partial W}{\partial z} = 0, \quad (2.29)$$

$$U \frac{\partial U}{\partial x} + W \frac{\partial U}{\partial z} = - \frac{1}{\rho_r} \frac{\partial P}{\partial x} - \left[\frac{\partial}{\partial x} \overline{u^2} + \frac{\partial}{\partial z} \overline{uw} \right], \quad (2.30)$$

$$U \frac{\partial W}{\partial x} + W \frac{\partial W}{\partial z} = - \frac{1}{\rho_r} \frac{\partial P}{\partial z} - \frac{\theta_v}{T_r} g - \left[\frac{\partial}{\partial x} \overline{uw} + \frac{\partial}{\partial z} \overline{w^2} \right], \quad (2.31)$$

$$U \frac{\partial \theta}{\partial x} + W \frac{\partial \theta}{\partial z} = - \left[\frac{\partial}{\partial x} \overline{u\theta} + \frac{\partial}{\partial z} \overline{w\theta} \right], \quad (2.32)$$

$$U \frac{\partial Q}{\partial x} + W \frac{\partial Q}{\partial z} = - \left[\frac{\partial}{\partial x} \overline{uq} + \frac{\partial}{\partial z} \overline{wq} \right], \quad (2.33)$$

where the terms containing the effect of the Coriolis force have been neglected.

Careful consideration of the terms on the right hand side of (2.30)-(2.33) (Yeh and Brutsaert, 1970; Plate, 1971; Peterson, 1972) led to the conclusion that the first term between the brackets, as well as the pressure term in (2.30), are negligible in comparison with the second term between the brackets, at least for fetches larger than, say, 1 m. This implies that Eq. (2.31) can be omitted, while Eqs. (2.30) (2.32) and (2.33) reduce to

$$U \frac{\partial U}{\partial x} + W \frac{\partial U}{\partial z} = - \frac{\partial}{\partial z} \overline{uw} , \quad (2.34)$$

$$U \frac{\partial \theta}{\partial x} + W \frac{\partial \theta}{\partial z} = - \frac{\partial}{\partial z} \overline{w\theta} , \quad (2.35)$$

$$U \frac{\partial Q}{\partial x} + W \frac{\partial Q}{\partial z} = - \frac{\partial}{\partial z} \overline{wq} . \quad (2.36)$$

The four equations (2.29) and (2.34)-(2.36) contain seven unknowns: U , W , θ , Q , \overline{uw} , $\overline{w\theta}$ and \overline{wq} , hence they do not form a closed system and cannot be solved. This is the well-known closure problem. This problem can only be tackled if a closed system of equations is obtained. Two possibilities now arise:

- (i) decrease the number of unknowns by expressing the quantities on the right hand side of Eqs. (2.34)-(2.36) as a combination of the other dependent variables on the left hand side. This is called first order modeling.
- (ii) increase the number of equations by introducing conservation equations for the quantities on the right hand side of Eqs. (2.34)-(2.36). This is called second order modeling.

If the first possibility is used a relatively simple model will emerge which can often be treated analytically. If we use the second possibility the problems turn out to be aggravated. By devising additional equations for the variables on the right hand side new unknowns appear which make the difference between the number of equations and the number of unknowns even larger. So in this case one is again forced to use the first option: reducing the amount of unknowns. The modeling of the unknowns in the second moment equations often cannot be done rigorously on sound physical grounds, and in that case it must be constructed rather artificially.

In the next Chapter models will be encountered which have been constructed using either one of the two mentioned closure techniques. There, the benefits and drawbacks of both types of models will be analysed.

3 Models for the description of the internal boundary layer

3.1 INTRODUCTION

In this Chapter a review will be given of existing models concerned with turbulent shear flows adjusting to suddenly altered lower boundary conditions. In this review of theoretical models we will focus our attention primarily on models dealing with a change in surface roughness.

Ever since 1958 (Elliott) and 1952 (Gandin) (for the English, respectively Russian literature) a lot of work has been dedicated to the problem under consideration. A tremendous number of articles concerning models of ever increasing sophistication has appeared. In some of these articles the growth rate of the IBL is predicted, while in others a more detailed description of the flow within this region is given. The results of the first group of articles can be applied directly to estimate the constraints posed upon the maximum height at which instruments may be located when measuring e.g. profile derived fluxes. The second group of papers allows the prediction of the velocity-and shear stress profiles within the IBL as well as the variation of the surface shear stress downstream of the discontinuity. Not until the more advanced second order models appeared in the late sixties was it possible to give a more complete description in terms of the behaviour of other turbulent quantities which were neglected or parameterized in the older models.

Apart from some early methods in which the roughness change problem is treated as a diffusion problem (Philip, 1959; Dyer, 1963) the bulk of theoretical models will be divided into three classes of increasing order of complexity:

1. First order closure
 - (i) Integral models
 - (ii) Models based on self preservation
 - (iii) Numerical models
2. "One and a half" order closure (also called $k - \epsilon$ or turbulent

kinetic energy (TKE) models)

3. Second order closure.

In the following we will discuss these types of models.

3.2 INTEGRAL MODELS

3.2.1 Description

One of the first attempts to treat the effect of a step change in roughness length theoretically was carried out by Elliott (1958).

Elliott postulated that there is a region in which the changes of the flow occur and he called this the internal boundary layer (IBL). Within the IBL he assumed that the shear stress is independent of the height (z), while the surface shear stress is a function of the downstream distance (x). All other models in this class make the same assumptions but only differ in the assumed form of the shear stress profile within the IBL (Panofsky and Townsend 1964, Taylor 1969).

To close the set of equations formed by (2.29) and (2.34), Elliott used the relationship between the turbulent flux of momentum \overline{uw} and the gradient of the mean wind U , which is valid in horizontally homogeneous situations, assuming that it also holds in advective situations:

$$-\overline{uw} = \ell^2 \left(\frac{\partial U}{\partial z} \right)^2, \quad (3.1)$$

or

$$\tau = -\rho \overline{uw} = \rho K_m \frac{\partial U}{\partial z}, \quad (3.2)$$

which means that

$$K_m = \ell^2 \left| \frac{\partial U}{\partial z} \right|, \quad (3.3)$$

where ℓ is the mixing length, usually taken as

$$\ell = kz. \quad (3.4)$$

The equations (2.29), (2.34), (3.1) and (3.4) form a closed system which can be solved. In accordance with the von Kármán-Pohlhausen technique the equations are integrated over the height of the IBL resulting in an equation relating U , d and u_{*2} . By inserting a prescribed velocity profile in that equation and combining it with the necessary condition of velocity continuity at $z = d(x)$, Elliott obtained a differential equation which can be solved to give $d(x)$. After this quantity has been established it is possible to obtain $u_{*2}(x)$, $U(x,z)$ and $\overline{uw}(x,z)$ respectively by the scheme outlined in Appendix 2.

After Elliott's publication a number of authors have sought to improve his model. Panofsky and Townsend (1964) removed the discontinuity in shear stress in Elliott's model by assuming a linear variation of the shear stress with height within the IBL, instead of a constant stress. A few years later Taylor (1969) constructed a model in which even the shear stress gradient was continuous at the interface at $z = d$. For a comparison of these shear stress profiles the reader is referred to Taylor (1969). Both theories are refinements of the model of Elliott and their results consequently do not considerably differ from the original model. A somewhat deviating approach was presented by Plate and Hidy (1967) who integrated the momentum equation between $x = 0$ and a given value of x , while they also incorporated a streamline displacement in their model. The advantage of this model, as pointed out by Plate (1971), is the possibility of incorporating pressure gradients and a gradual change in roughness downstream of $x = 0$. Plate and Hidy developed this model to predict the behaviour of a wind blowing from a smooth solid surface to a water surface with small waves of increasing waveheight.

3.2.2 Results

Because in the models of Elliott, Panofsky and Townsend and Taylor the shear stress- and velocity profiles are prescribed, their main merit lies in the prediction of the growth rate of the IBL height $d(x)$. Elliott approximated this rate by

$$d(x) = a \cdot z_{02} \left(\frac{x}{z_{01}} \right)^{0.8}, \text{ where } a = 0.75 - 0.03 \ln \left(\frac{z_{02}}{z_{01}} \right) \quad (3.5a,b)$$

The $d \propto x^{0.8}$ growth rate appears to be a very good estimate. Panofsky and Townsend (P&T) used a somewhat different definition of d resulting in d -values that are larger than those predicted by (3.5), but the growth rate is essentially the same. Note that in this type of model also the variation of the surface shear stress for $x > 0$ can be computed. By equating the horizontal velocity at $z=d$, an equation is obtained that gives u_{*2} as a function of x, z_{02} and m , where

$$m = z_{01}/z_{02} \quad \text{and} \quad M = \ln(m) \quad (3.6a,b)$$

This is not mentioned by the authors themselves. For both the Elliott and the Panofsky and Townsend models these curves are given in Figure 3.1 for rough-to-smooth (RS) as well as smooth-to-rough (SR) changes. A typical feature of these curves is the over (under)-shoot followed by a gradual return to a new equilibrium value. This behaviour of the surface shear stress was confirmed by experiments (e.g. Bradley, 1968).

3.2.3 Discussion

The main shortcoming of integral models is the a priori assumption of the profiles of velocity and shear stress within the IBL, which pose a severe restriction to the possible types of flow for $x > 0$. Just downstream of $x = 0$ the flow is developing towards a new equilibrium, hence it is very doubtful that an equilibrium logarithmic profile will exist there.

Furthermore the concept of the mixing length is assumed, where only depends on local, averaged quantities. This assumption ignores all history of the turbulent flow upstream, which is not realistic.

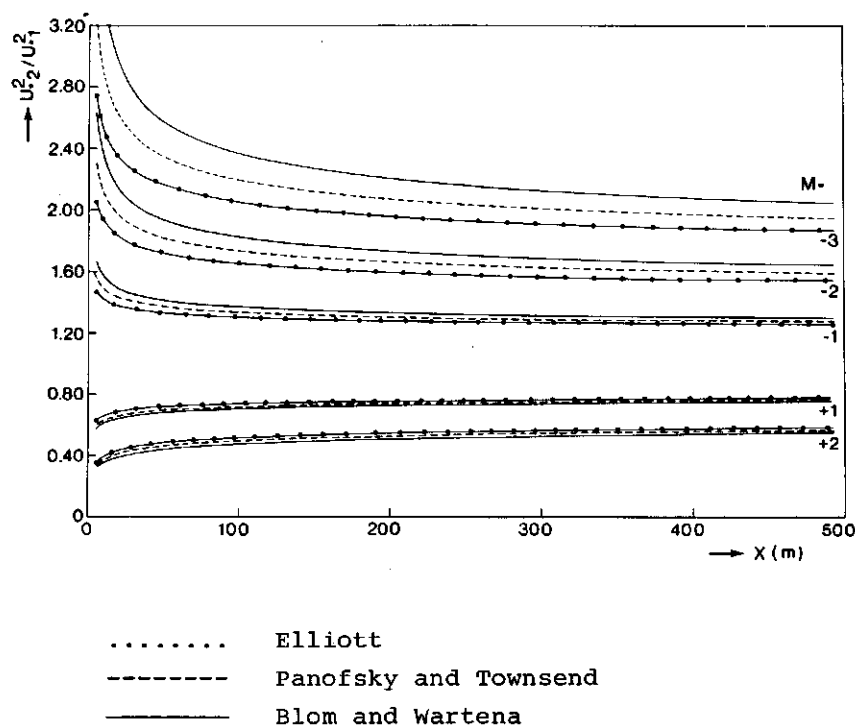


Figure 3.1 Variation of surface shear stress with downstream distance for various models

As was mentioned by Antonia and Luxton (1972) and also by Plate and Hidy (1967) the integral models require a limited depth of the disturbed boundary layer compared with the whole of the PBL. This means that integral models are only applicable within a limited distance downstream of $x = 0$.

Finally, Peterson (1969) states that the prescribed form of the shear stress profile is only true if the nondimensional windshear

$\phi_m = \frac{kz}{u_*} \frac{\partial U}{\partial z}$ equals unity in the transition region in nonequilibrium flow conditions, which is often not the case. The prescription of this profile is in fact a fourth equation which creates a model where the number of equations exceeds the number of unknowns. It should be possible to calculate shear stress distributions by solving the horizontal equation of motion, given the velocity profile and the equation of continuity. But, as Taylor (1969) noticed, the variation of the wind profile downstream and the behaviour of the surface shear stress might be obtained without the use of the mixing length hypothesis. The vertical distribution of shear stress, however, can not be obtained without this hypothesis.

3.3 MODELS BASED ON SELF PRESERVATION OF PROFILES

3.3.1 Description

The first and most important attempt to treat the problem of a step change in roughness based on arguments of self preservation was undertaken by Townsend (1965a,b; 1966).

Assuming the profile of a given quantity to be "self preserving" is equivalent to saying that the functional form for the vertical distribution of this quantity is invariant with fetch and that the scale is a function of fetch only.

Townsend assumed that upstream of the step the velocity is given by the usual logarithmic profile, i.e. $\frac{kz}{u_*} \frac{\partial U}{\partial z} = 1$, while downstream of $x = 0$ he distinguished three regions:

- region I : only vertical displacement of streamlines
- region II : transition layer
- region III : velocity and shear stress locally adapted to the new roughness.

The velocity within the IBL (region II and III) consists of three parts:

- 1/ the original velocity profile for $x < 0$
- 2/ a contribution due to flow acceleration

3/ the velocity change due to the vertical displacement of the streamlines.

Using the condition of incompressibility, Townsend showed that the change in velocity due to the streamline displacement is smaller than the change due to the acceleration of the flow, so part 3/ was neglected within the IBL. For a change of roughness Townsend assumed self preservation of the shear stress and of the acceleration term of the velocity change:

$$v(z) = \frac{u_0}{k} f\left(\frac{z}{\ell_0}\right) \quad (3.7)$$

$$\tau_2(z) = u_*^2 + (u_{*2}^2 - u_{*1}^2) F\left(\frac{z}{\ell_0}\right) \quad (3.8)$$

where v is the contribution part 2/.

$\ell_0(x)$ is a length scale denoting the thickness of the accelerated region

$u_0(x)$ is the scale of the change in mean velocity

f, F are universal functions, independent of x because of the assumed self preservation.

Substituting these expressions in the equation of motion (2.34) gives a relation between f and F :

$$-\eta \left(\frac{df}{d\eta}\right) = \frac{dF}{d\eta} \quad (3.9)$$

where $\eta = z/\ell_0$

A second relation between f and F is obtained by using the mixing length hypothesis (Eq.(3.2)):

$$\frac{df}{d\eta} = \eta^{-1} F. \quad (3.10)$$

Solving (3.9) and (3.10) for f and F yields

$$F(\eta) = e^{-\eta} \quad (3.11)$$

$$f(\eta) = - \int_{\eta}^{\infty} \frac{e^{-x}}{x} dx = \text{Ei}(-\eta) = -E_1(\eta) \quad (3.12)$$

To obtain expressions for $u_0(x)$ and $\ell_0(x)$ Townsend used the assumption that for small values of z the velocity profile within the IBL is adapted to the new roughness length and friction velocity i.e.

$$U = \frac{u_*}{k} 2.3 \ln\left(\frac{z}{z_0}\right) \quad (3.13)$$

This is the so-called inner boundary condition. Together with the forms derived for $f(\eta)$ and $F(\eta)$, Townsend arrived at expressions for $u_0(x)$ and $\ell_0(x)$.

It was shown by Blom and Wartena (1969) that for small values of z the expression for u_0 did not approach the inner boundary condition. They were able to remove this inconsistency of Townsend's theory, giving a slightly different expression for $u_0(x)$. Blom and Wartena also applied Townsend's theory to two subsequent changes in surface roughness. They found expressions analogous to those calculated by Townsend. A consequence of multiple roughness changes, however, is an increasing number of layers generated by every new roughness change, which are all influenced by different regions of the upstream surface. The number of analytical solutions then grows rapidly and soon becomes unattractive.

Self preservation of wind and shear stress profiles was used by Logan and Fichtl (1975) who assumed that the velocity defect and shear stress difference functions were self preserving. They derived analytical forms for these functions, which agree reasonably well with the experimental results of Bradley (1968).

The latest attempt, using arguments of self preservation of profiles has been made by Mulhearn (1977), who considered the changes in surface fluxes and mean profiles of velocity, temperature and concentration of a scalar quantity after a roughness change. Mulhearn notices that z_0 is a quantity primarily connected with the flow and not with the underlying surface. Only in equilibrium flows can a connection be made between z_0 and the surface bounding the flow. Thus near the transition we may not, strictly

speaking, use an equilibrium value of z_0 to describe the flow. Mulhearn's model is very similar to the one of Townsend: he assumed that the velocity within the IBL consists of three parts identical to the expressions used by Townsend, hence he also used the relations (3.7) and (3.8). Mulhearn showed that self preservation is possible if $u_0 \sim (\ell_0)^a$ which leads to a relation between f and F resembling eq. (3.9):

$$af - \eta \left(\frac{df}{d\eta} \right) = \frac{dF}{d\eta} \quad (3.14)$$

A second relation between f and F is derived from a perturbed form of the upstream mixing length equation, resulting in eq. (3.10). In practice

$$u_0^{-1} \frac{du_0}{dx} \ll \ell_0^{-1} \frac{d\ell_0}{dx}$$

which implies that $|a| \ll 1$, then eq. (3.14) transforms to eq. (3.9) and the same solutions eqs. (3.11) and (3.12) are obtained. Mulhearn distinguished two regions in his analysis:

1. the region just downstream of the roughness change, where z_0 can not be used to specify the underlying surface
2. the region where the fetch is large enough to use z_0 .

Using the argument of self preservation and imposing the 'law of the wall' on the velocity profile close to the surface, he was able to derive expressions for both the velocity and length scales u_0 and ℓ_0 in these regions. Finally, Mulhearn extended his analysis to changes in a scalar quantity (concentration of a passive contaminant or temperature) but excluded stability effects. He obtained expressions for g and G which describe the change in scalar concentration and scalar flux respectively, and are identical to those for f and F .

3.3.2 Results

The self preserving profiles for the velocity and shear stress viz. eqs. (2.11) and (2.12) are identical in the theories of

Townsend, Blom and Wartena and Mulhearn, only Logan and Fichtl obtained different forms.

A comparison of the velocity profiles of Blom and Wartena (1969) with the corresponding ones of Elliott (1958) and Panofsky and Townsend (1964) is presented in Figure 3.2. It shows that the different models give very similar results as far as velocity changes are concerned, a fact already noticed by Taylor (1969). This means that the horizontal velocity U is not a proper quantity to distinguish between the performance of different models or to compare theoretical results with experimental data. Because the wind profile within the IBL is prescribed or subjected to severe restrictions, it is more useful to consider e.g. $d(x)$ for which these models were originally devised. Nevertheless some authors

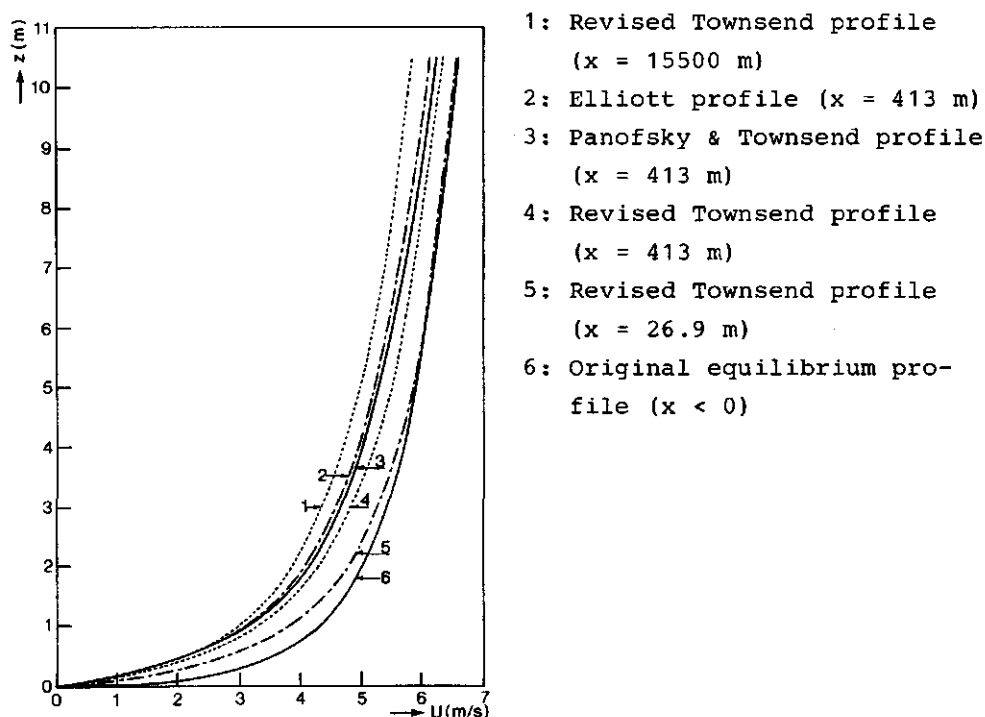


Figure 3.2 Velocity profiles after a change in surface roughness. (After Blom and Wartena, 1969).

(e.g. Panofsky and Townsend, 1964; Blackadar et al., 1967) do not use this quantity to examine the results of certain models with experiments. This is understandable because mean velocity measurements are relatively easy to make, while turbulent quantities such as shear stress are much harder to measure. As a consequence the number of mean velocity measurements after a change in surface roughness cited in the literature far exceeds that of all other quantities.

In Figure 3.1 the downstream variation of the surface shear stress from the model of Blom and Wartena is compared with the variation predicted by some integral models. Apparently there is a close agreement between these sets of curves. The difference in the surface shear stress values of the various models is larger for the SR than for the RS change. Furthermore this difference decreases with increasing downstream distance and, especially for the SR case, it also decreases with decreasing roughness difference. Both phenomena are a consequence of the fact that firstly, u_{*2} approaches u_{*1} when $x \rightarrow \infty$, though only theoretically for an infinitely deep boundary layer, and secondly, a smaller difference $z_{02}-z_{01}$ naturally will cause a smaller disturbance of the flow field. The growth rate of ℓ_0 is similar to the one of d predicted by Elliott, exponents ranging between 0.71 and 0.88 for $-3 < M < 3$.

A puzzling fact remains the independence of ℓ_0 on the downstream surface roughness (z_{02}), which can be detected on inspecting Blom and Wartena's eq. (12), which can be written as

$$\ell_0 \left[\ln \left(\frac{\ell_0}{z_{01}} \right) - 1 \right] = 2k^2 x.$$

In Figure 3.3 shear stress profiles within the IBL of several models are compared. Figure 3.3b uses the true vertical distance and hence it also shows the height of the IBL of the different models. If we nondimensionalize the vertical coordinate by means of the appropriate IBL height of the model Figure 3.3a is obtained. It shows that for the three models in question, the profiles of the

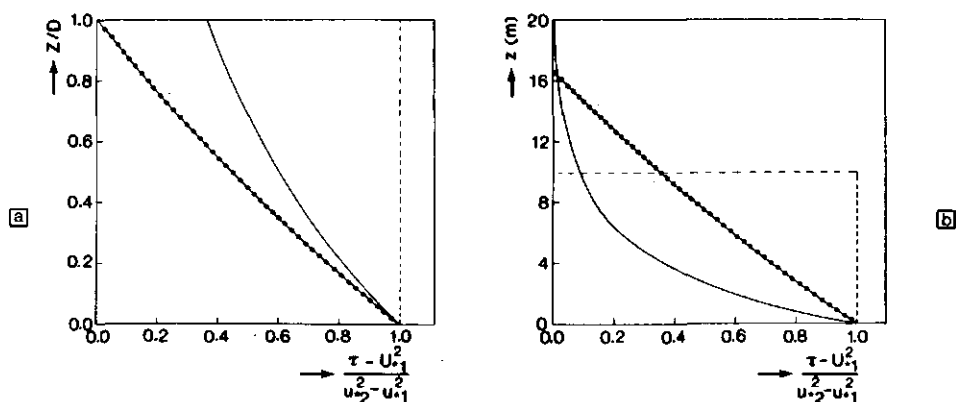


Figure 3.3 Shear stress profiles for various models for a smooth to rough transition ($M = -3$). ---- Elliott, Panofsky and Townsend, — Blom and Wartena.

shear stress do not resemble each other, in contrast with the close agreement between the velocity profiles and the downstream variation of the surface shear stress.

3.3.3 Discussion

One of the main shortcomings of the self preservation models is the mixing length assumption (also discussed in 3.2.3) which they have in common with all other first order closure models. All self preservation models neglect the influence of the induced pressure field which makes the analysis only applicable to distances relatively far from the step change i.e. $d/z_{O2} \gg 1$, (see e.g. Tani (1968) and Antonia and Luxton (1972)).

The models of Townsend and Blom and Wartena result in a small value of the adapted layer height $\delta \approx 0.1 \cdot \ell_0$ giving a height to fetch ratio of 1/300. This is considerably smaller than predicted by other theories.

Finally, another limitation of the self preservation models is that the prediction of the surface shear stress and the IBL

thickness is only justified for small changes of the roughness length. This limits their applicability.

3.4 NUMERICAL MODELS WITH FIRST ORDER CLOSURE

3.4.1 Description

Further development in the analysis of the effects occurring after a change in surface characteristics is made primarily with the aid of numerical models. After writing the governing equations in a finite difference form and specifying the boundary conditions, the spatial distribution of each dependent variable can be computed. The set of equations always include the continuity equation and the horizontal momentum equation. To these equations one may add one or more terms to incorporate additional effects e.g. buoyancy. Several authors include the vertical momentum equation in their model, and hence are able to calculate the pressure field generated by the surface discontinuity. Also, other equations may be added to analyse more than one change in surface conditions simultaneously.

All models mentioned in this section still use the mixing length assumption to close the set of equations, hence they all have the same deficiencies connected with this particular assumption. Most of them consider flows in neutral stability only. It is not necessary to give a detailed description of each and every first order model ever constructed, rather a survey of the most important characteristics of them is worth mentioning. These details are summarized in Table 3.1.

3.4.2 Results

To present the results of every first order model would be rather exhaustive, so only the most interesting features are briefly reviewed here, and comparisons are made with the results of earlier models.

The governing equations of Taylor's first model (1969a) are

Table 3.1 Summary of first order models

MODEL	ADDITIONAL TERMS IN EQ. (2.34)	ADDITIONAL EQS. TO (2.29) and (2.34)	MIXING LENGTH DISTRIBUTION	REMARKS
Taylor (1969 ^a)	none	none	$l = k(z + z_{oi})$	abrupt change in mixing length throughout IBL
Taylor (1969 ^b)	pressure and Coriolis force	Y-momentum	$l = \frac{k(z + z_{oi})}{1 + kz/\lambda}$ $\lambda = \frac{G}{f}(0.0004)$	description of Ekman layer
Wagner (1966)	time derivative and Coriolis force	Y-momentum (2.29) and (2.34) combined, incl. energy, eq. of state	$l = \frac{k(z + z_o)}{1 + k(z + z_o)/\lambda}$ $\lambda = 0.00027(u_g^2 + v_g^2)/f$	see above + non- steady state
Nickerson (1968)	none	none	$l = kz$	change in u_* instead of z_o
Onishi and Estoque (1968)	pressure	Z-momentum ⁽¹⁾	$l = kz$	gradual change of z_o coincident with change in l
Huang and Nickerson (1974)	pressure	Z-momentum ⁽¹⁾	$l = kz\phi$ $\phi = \text{stability}$ factor	also for unstable situations and change in surface temperature
Taylor (1970)/(1971)	none	heat flow eq.	$l = \frac{k(z + z_o)}{\phi}$ $\phi = \text{stability}$ factor	also for change in surface temperature or heat flux. For (un)stable situation

(1) The z-momentum equation is transformed into an elliptic equation for the pressure, by taking the divergence of the vector momentum equation.

identical with the ones used by Panofsky and Townsend (1964). It is therefore not surprising that the results of his numerical model show good agreement with their results. But where P&T assumed a linear variation of $\tau^{1/2} - u_{*1}$ with z/d , Taylor found the distributions given in Figure 3.4. This result indicates that (i) shear stress profiles are a much better quantity for comparing different models than the velocity profiles and (ii) these shear stress profiles are almost self preserving as assumed by Townsend (1965, 1966). On the other hand self preservation of shear stress profiles was not found by Wagner (1966), Onishi and Estoque (1968) and Huang and Nickerson (1974). The shear stress profiles of these last three models closely resemble each other, which for the latter two is not surprising since Huang and Nickerson's model comprises only a slight extension of Onishi and Estoque's model. The upstream effect of the roughness change, as a conse-

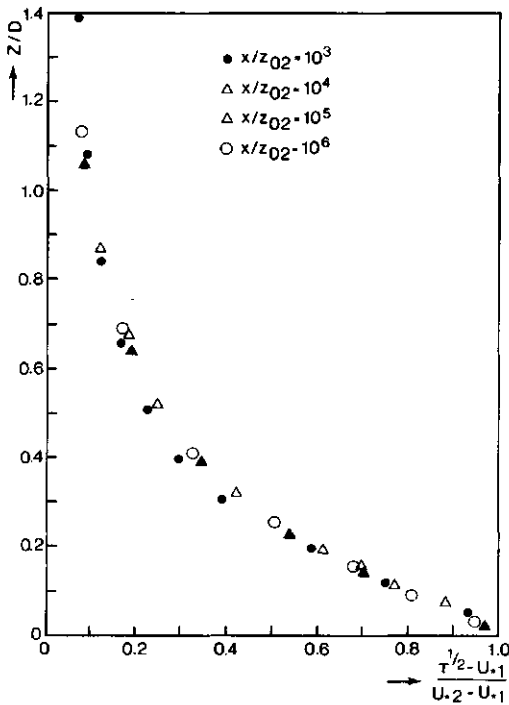


Figure 3.4

Shear stress profiles after a change in surface roughness ($M = +2$).

(After: Taylor, 1969a)

quence of the introduction of a pressure term in the horizontal momentum equation, is clearly visible in the results of Onishi and Estoque (1968) and Huang and Nickerson (1974). It is absent in Wagner's results, although Wagner did detect a slight effect in the upstream horizontal velocity distribution.

The inclusion of a pressure term and, consequently, the vertical momentum equation gives an opportunity to study the generated pressure field (Figure 3.5). As observed by Onishi and Estoque

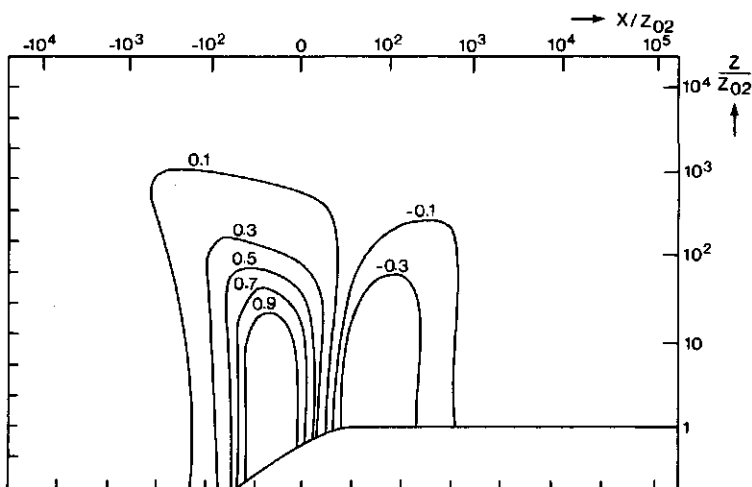


Figure 3.5 Nondimensional pressure field (deviation from the hydrostatic pressure) for a smooth to rough transition with $M = -5$. (After Huang and Nickerson, 1974a).

this pressure field "... acts as if it were a smoothing function, causing the change in wind speed to occur over a larger horizontal distance". At larger downstream distances no net pressure effect is noticeable. The effect of the pressure is thus confined to a small distance before and after the roughness change. Here it also affects the distribution of the vertical velocity, shifting the place of maximum velocity and decreasing absolute values compared to the vertical velocity field obtained without the pressure term extension.

There is a close agreement in the growth rate of the IBL height between the different models, all confirming Elliott's power law. However, the absolute value of the IBL height is just a matter of definition. Taylor (1969) defined it as the height where $\tau^{1/2} - u_{*1}$ has been reduced to 10% of its surface value of $u_{*2} - u_{*1}$, whereas Wagner (1966) defines this height by a 0.2% change in horizontal velocity. A slight difference in the growth rate between the RS and SR changes was noticed by Huang and Nickerson (1974). They found the growth rate for a RS change to be approximately $\propto x^{0.7}$ instead of $\propto x^{0.8}$. This was attributed to the fact that the adjustment of the flow to the new z_0 value is faster above a relatively rough surface than above a smooth surface, a feature also noticed by Wagner (1966).

The effects of stratification were incorporated in the models of Huang and Nickerson (1974) and Taylor (1970, 1971). The former authors also considered a change in surface temperature coincident with a change in surface roughness. The presence of an unstable upstream flow, they concluded, only results in small deviations from the neutral upstream flow case. Unfortunately, the effects created by the instability of the upstream flow and those created by the change in surface temperature are difficult to distinguish. Taylor (1970, 1971), in an extension of his model of 1969, also considered the effect of a change in roughness with or without either a change in surface heat flux or temperature. When only a step change in surface temperature without a change in roughness was considered, solutions became unstable for large distances downstream. Taylor attributed this to the particular form used for the mixing length. In the case of a step change in heat flux similar problems arised. The extension of his model to include the stable case Taylor (1971) resulted in the peaking of the shear stress profiles near the outer edge of the IBL in the case of a step in surface heat flux. Taylor's model results indicate that roughness change effects are greatest near the leading edge, while further downstream thermal stability effects will play a dominant role.

Finally, the effects of a roughness change in a deeper layer

than the constant flux layer were examined in the models by Taylor (1969b,c) and Wagner (1966). Both considered these changes in the Ekman layer at neutral stability. This extension could be made by incorporating the Coriolis force effect in the equations of motion. At the same time the length scale distribution has to be specified in the Ekman layer.

Taylor was able to show that a roughness change has an effect on the wind direction noticeable up to a rather large value of $x/z_{02} \approx 10^7$.

3.4.3 Discussion

The first advantage of numerical models over analytical methods is the possibility of deriving the profiles of the velocity and the shear stress instead of prescribing them (integral models) or making restricting assumptions (self preservation models). Furthermore, numerical models do not need an a priori assumption about an internal boundary layer required for earlier models. The second advantage of numerical models is the relative ease with which they can be modified or extended to include various additional terms in the governing equations to account for additional effects e.g. pressure, buoyancy etc.

In general, the disadvantage of numerical models is that in order to get computationally stable solutions sometimes additional simplifications in the terms of the governing equations have to be introduced which may affect the final solutions. The effects of these simplifications are difficult to estimate. Moreover, any numerical model usually contains a large number of numerical constants, the values of which are subject to limitations posed by modeling assumptions. These values may range quite arbitrarily between those limits. A fundamental objection against the use of first order models (numerical as well as analytical) is, again, the closure using the mixing length hypothesis (Plate, 1971).

Peterson (1969) points out that in some models the value of the

nondimensional wind shear is put equal to unity (e.g. Onishi and Estoque (1968)) while this is in disagreement with experimental data and the results of higher order models. This simplification, Peterson asserts, results in smooth velocity profiles without a point of inflection. Huang and Nickerson (1974b) notice that mixing length models do not contain advection and diffusion terms of the turbulent kinetic energy. This manifests itself in an overestimation of the surface shear stress in case of a SR change. The most essential difference, they maintain, is the much faster increase of the surface shear stress in higher order models after a RS change. As indicated by their results and those of others (Peterson, 1969) a mixing length model reasonably describes the flow structure only when the production of turbulent kinetic energy (TKE) equals its dissipation, which is not the case in the non-equilibrium part of the flow just downstream of the surface roughness change, and in the transition region between the adapted layer and the edge of the IBL.

3.5 NUMERICAL MODELS WITH A HIGHER ORDER CLOSURE

3.5.1 Description

As was pointed out before, closing the horizontal momentum equation by means of correlating the Reynolds stress with the gradient of the mean velocity is beset with many theoretical and practical objections. To overcome these objections, many other ways have been tried to model the Reynolds stress terms. Reviews of several methods to obtain the desired closure are presented e.g. by Launder and Spalding (1972), Mellor and Herring (1973), Harsha (1977), Rodi (1979) and Zeman (1981). For the roughness change problem, comparisons between higher order closure models and integral and first order closure models were presented by Taylor (1973) and Wood (1978).

The models which use a higher order closure scheme to analyse the effects of a change of surface conditions can be divided into two groups:

(i) Turbulent kinetic energy models (one and a half order closure).

This group contains the models of Bradshaw, Ferriss and Atwell (1967), Peterson (1969), Shir (1972) and Huang and Nickerson (1974b). The basic assumption is: $\tau_{ij} \propto e^2$, hence the Reynolds stress depends on the turbulent kinetic energy (TKE):

$k = e^2 = \overline{u_i u_i}^{(*)}$. Hence a transport equation for e^2 must be added.

This type of model can be extended to include also a transport equation for the mixing length (k- ϵ models). Although k- ϵ models appear to be very promising for boundary layer analysis, little use of them for roughness change modelling has so far been made, only one attempt was briefly described by Wood (1978).

(ii) Second order closure models.

This group consists mainly of different versions of the model of Wyngaard, Coté and Rao (1974): viz. those of Rao et al. (1974a) for a change in surface roughness, Rao (1975) the same model but extended for diabatic situations, Rao et al. (1974b) for a change in surface roughness, temperature and humidity, and Wyngaard and Coté (1974) for the evolution of a convective planetary boundary layer. In 2nd order modeling an explicit transport equation is written for every second moment. For the two-dimensional case this requires equations for: \overline{uw} , $\overline{u^2}$, $\overline{v^2}$, $\overline{w^2}$ and $\overline{\epsilon}$. For diabatic conditions those are completed with equations for: $\overline{\theta}$, $\overline{w\theta}$, $\overline{u\theta}$ and $\overline{\theta^2}$. Finally these can also be extended with equations for: \overline{Q} , \overline{wq} , \overline{uq} and $\overline{q^2}$ if a change of surface humidity is coincident with the roughness change. However, the problem of closure is getting more difficult with every equation which is added to the list.

ad (i) Kinetic energy models

An attempt to model turbulence was undertaken in 1967 by Bradshaw, Ferriss and Atwell (1967). Apart from the equations (2.29) and

(*) Some authors define the TKE to be equal to $\frac{1}{2} \overline{e^2}$.

(2.34) with the momentum equation modified with the boundary layer approximation to account for the pressure gradient force, an equation is added which accounts for the transport of TKE:

$$\frac{1}{2}\rho \left(U \frac{\partial \overline{e^2}}{\partial x} + W \frac{\partial \overline{e^2}}{\partial z} \right) - \tau \frac{\partial U}{\partial z} + \frac{\partial}{\partial z} (\overline{pw} + \frac{1}{2}\rho \overline{e^2 w}) + \rho \overline{\epsilon} = 0 \quad (3.15)$$

where $\overline{e^2} = \overline{u_1 u_1}$ is the TKE

$\overline{\epsilon} = \nu (\partial u_1 / \partial x_j)^2$ is the viscous dissipation rate.

Closure of this equation is obtained by defining:

$$a_1 = \tau / \rho \overline{e^2} \quad (3.16a)$$

$$L = \frac{(\tau / \rho)^{3/2}}{\overline{\epsilon}} \quad (3.16b)$$

$$G = (\overline{pw} / \rho + \frac{1}{2} \overline{e^2 w}) / (\frac{\tau_{\max}}{\rho})^{1/2} \cdot \frac{\tau}{\rho} \quad (3.16c)$$

where $\tau_{\max} = \tau(z = \frac{1}{2}d)$

Bradshaw et al. used $a_1 = 0.15$ and assumed that L and G are functions of z/d and depend on the shear stress profile. With the assumptions (3.16) Equation (3.15) is transformed into a transport equation for the Reynolds stress \overline{uw} . After defining suitable boundary conditions, Bradshaw et al. (1967) were able to show that their model proved to be satisfactory in correlating model predictions with experimental results. However, the authors did not apply their model to the roughness change problem. This extension of their model was made by Schols (1979) and Douwes (1980).

Peterson (1969) and Shir (1972) also used an equation similar to Eq. (3.15) to close their set of equations, the latter making use of a vorticity equation instead of the usual horizontal momentum equation. A slight modification to these Bradshaw-type models was introduced by Huang and Nickerson (1974b). Referring

to the experiments performed by Yeh and Nickerson (1970) the relationship between τ and $\overline{e^2}$ was transformed into:

$$\tau = -\overline{uw} = (\overline{e^2})^{\frac{1}{2}} \ell_1 \frac{\partial U}{\partial z} \quad (3.17)$$

The diffusion term and the dissipation term were approximated by respectively:

$$\overline{pw} + \frac{\overline{we^2}}{2} = -K_m \frac{\partial}{\partial z} (\frac{\overline{e^2}}{2}), \quad (3.18a)$$

$$\text{where } K_m = (\overline{e^2})^{\frac{1}{2}} \ell_1$$

$$\text{and } \overline{\epsilon} = (\frac{\overline{e^2}}{2})^{3/2} \frac{1}{\ell_\epsilon} \quad (3.18b)$$

After deriving equations for the length scales ℓ_1 and ℓ_ϵ , the TKE equation, the continuity equation and the two momentum equations (including the pressure term) form a closed set.

A discussion on the relative importance of the terms in the TKE equation and the momentum equation was given by Peterson (1972). He applied his analysis to a more extended TKE equation than the one used in his model of 1969. In his analysis he used the same approximations for the diffusion and dissipation terms as in his earlier model. With the given set of equations 7 different models are constructed depending on the inclusion or rejection of one or more terms of interest in the model and comparisons are made between the predictive qualities of each model.

ad (ii) Second order closure models

As is discussed e.g. by Zeman (1981) many attempts have been made in turbulence modeling by means of 2nd order closure techniques. But the application of these techniques to the roughness change problem is restricted to a small number of cases.

The only models described so far, with explicit equations for the second moments are the ones developed by Rao, Wyngaard and Coté. They were summarized at the beginning of this section. The horizontal momentum equation (2.30) was used without the pressure term.

The equation for the Reynolds stress tensor $\overline{u_i u_j}$ is derived from the Navier-Stokes equation resulting in:

$$U_j \frac{\partial}{\partial x_j} \overline{u_i u_k} + \overline{u_j u_k} \frac{\partial U_i}{\partial x_j} + \overline{u_j u_i} \frac{\partial U_k}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_i u_k u_j}) +$$

$$- (\overline{u_k \frac{\partial p}{\partial x_i}} + \overline{u_i \frac{\partial p}{\partial x_k}}) - 2 \overline{\varepsilon} \frac{\delta_{ik}}{3} \quad (3.19)$$

where $\overline{\varepsilon}$ is the viscous dissipation rate of TKE, and the summation convention applies. The third order terms on the right hand side of Eq. (3.19) have to be modeled. To achieve this, use has been made of the closure techniques devised by Lumley and Khajeh-Nouri (1974). This procedure is described by Wyngaard, Côté and Rao (1974). The resulting equations for \overline{uw} , $\overline{u^2}$, $\overline{v^2}$ and $\overline{w^2}$ ($\overline{uv} = \overline{vw} = 0$) together with the modified Eq. (2.30), the continuity equation (2.29) and an equation for $\overline{\varepsilon}$:

$$U_j \frac{\partial \overline{\varepsilon}}{\partial x_j} + \overline{\frac{\partial}{\partial x_j} (u_j \varepsilon)} = - 4 \frac{\overline{\varepsilon}^2}{\overline{e}} + 4a \overline{\varepsilon} \frac{P}{\overline{e}^2} \quad (3.20)$$

where $a = 0.5$ and P is the production rate of TKE, form the basic set of equations. After modeling the third order terms in Eqs. (3.19) and (3.20) the problem is solved numerically. In Chapter 4 we will give a more detailed description of the extended form of this model, which also contains temperature and humidity variables.

3.5.2 Results

It is interesting to know how much is gained by extending the numerical models with higher moment equations. Therefore we will concentrate our attention in this section on the differences in the results of 1st and higher order models.

The velocity profile downstream of $x = 0$, which in 1st order theories was a monotonic function of $\ln(z)$ demonstrates a different behaviour in the higher order models. Peterson (1969) was the first to notice that velocity profiles within the IBL, as

predicted by a higher order model, contain a point of inflection both for the RS and SR changes i.e. a local minimum, respectively maximum of the velocity gradient. This observation was confirmed by the results of Shir's model (1972) as well as those of Huang and Nickerson (1974b) and Rao et al. (1974a). If the nondimensional wind shear is defined by:

$$\phi_m = \frac{kz}{u_{*2}} \cdot \frac{\partial U}{\partial z} \quad (3.21)$$

then it follows from Eqs. (3.1)-(3.4) and

$$\tau = \rho u_*^2 \quad (3.22)$$

that ϕ_m was taken equal to unity in all models which use the mixing length assumption. Figure 3.6 shows the variation of ϕ_m in higher order models. It is clearly visible that $\phi_m > 1$ (< 1), for RS (SR) changes, over a major part of the IBL. For large

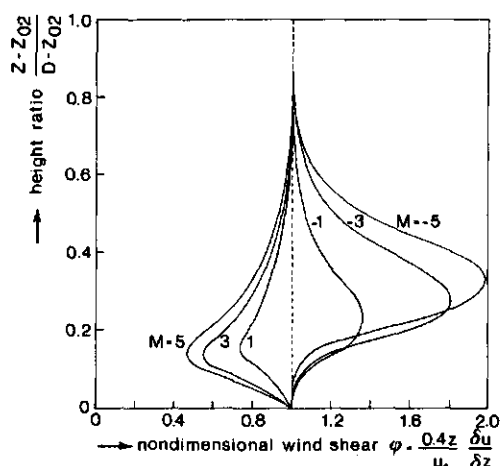


Figure 3.6 Comparative magnitudes of deviations of the nondimensional wind shear from unity for various roughness changes.

$$\begin{aligned} \text{For } M < 0, \quad x/z_{02} &= 10^2 \\ &= +1, &= 10^3 \\ &= +2, &= 10^4 \\ &= +3, &= 10^5 \end{aligned}$$

(After Peterson, 1969).

distances downstream the curves of ϕ_m tend to unity, indicating that mixing length models will give more satisfactory results there.

The shear stress profiles predicted by Peterson, Shir and Rao et al. do not agree with each other. Both Peterson and Shir found that, especially for the SR change, the shear stress profiles are not self preserving. For the RS change their profiles are only approximately self preserving. The model of Rao et al. on the other hand produced shear stress profiles that possess self preserving properties to a high degree, again especially for the RS change and to a slightly lesser extend for the SR change (see Figure 3.7). These last results agree very well with those of Taylor's numerical model (see 3.4.2).

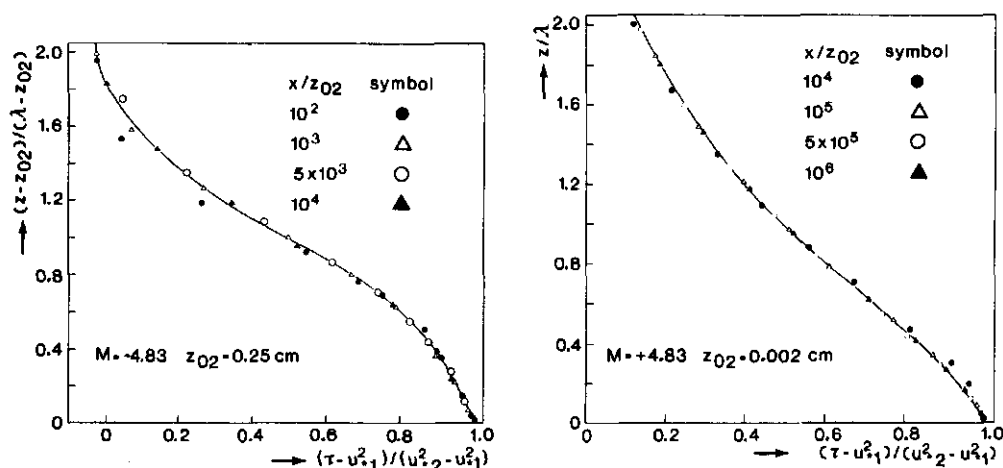


Figure 3.7 Nondimensional shear stress profiles for a smooth to rough transition (left) and for a rough to smooth transition (right). (After Rao, Wyngaard and Coté, 1974a).

The gain in accuracy of the prediction of the surface shear stress variation as compared with 1st order and older models is practically zero. A slightly faster response to the new surface was detected by Huang and Nickerson (1974b), while Shir found a

peculiar behaviour in the SR change. Seemingly in agreement with the data of Bradley (1968) two oscillations occur close to $x = 0$ before the surface shear stress levels off to its asymptotic value. This pattern was not discernible in the RS case nor in any other model.

The prediction of the value of the height of the IBL is ambiguously because of the different definitions of $d(x)$. Shir and Peterson introduce two IBL heights d_τ and d_u , defined with regard to the shear stress and velocity profiles respectively. All authors find, again, the same growth rate: $d(x) \propto x^{0.8}$, only Shir detects a small difference between the RS and SR changes. The influence of stratification on the growth rate was investigated by Rao (1975). He found that the growth rate exponent depends on the Obukhov length L but is approximately independent of M . He found that for neutral stability $d \propto x^{0.77}$ (both for RS and SR changes) while for unstable conditions exponents ranged from 0.9 (near neutral) to 1.4 (strongly unstable).

Finally, Rao et al. questioned the assumption of proportionality between shear stress and TKE as used by Bradshaw et al. (1967) and in other models in the nonequilibrium part of the flow. According to Rao's model results, the value of a_1 in Eq. (3.16a) varies considerably in this region. It approaches its equilibrium value only near the edge of the IBL. This is supported by the experimental results of Antonia and Luxton (1971).

3.5.3 Discussion

The need for second order models arose when "K-theory" or first order models appeared to fail in situations which are rapidly changing in space or time. The philosophy behind the application of second order models is: "if a crude assumption for second moments predicts first moments adequately, perhaps a crude assumption for third moments will predict second moments adequately". One implicitly assumes that in the latter case also the prediction of the first moments will be improved.

Hence second order models should provide a better tool for describing the effects after a change in surface conditions, in theory at least. In practice the results of second order models are only slightly better than the ones achieved by first order models as far as first moments are concerned. This is not surprising since there was hardly any room for improvement of the first moments anyway. The gain of second order models is largely confined to the description of the second moments, which are only crudely approximated by first order models or not at all. Unfortunately, measuring these second moments especially in the atmosphere, is at least very difficult (Reynolds stress components, turbulent fluxes) or even almost impossible (pressure strain correlations), so comparisons between theory and experiment are difficult to make. Also, it is important to notice that the reliability of a second order model depends on the modeling of the third moments as well as on the specification of the boundary conditions. Since the physical basis for this is weak, it is sometimes called an art (Wyngaard, 1982). This art of devising correct models for the higher order moments is still evolving rapidly, but a general basis to do so has already been constructed. Rotta (1951) and Lumley and his co-workers (Lumley, 1979), have developed the general lines for the modeling of the higher order terms, but no general agreement has yet been established. This is one of the disadvantages of the use of a second order model. Another one is the large number of modeling assumptions needed to close the system of equations. This unavoidably will bring along a great number of numerical constants the values of which are difficult to determine.

Second order models also have important advantages. First of all, they do not require the use of the mixing length assumption, which is the weak point of simpler models (Peterson, 1972; Huang and Nickerson, 1974b).

Finally, with higher order modeling one can easily incorporate advection and diffusion of TKE into the model as well as many other effects which can not be considered in first order models.

4 Examination of model performance

In this Chapter we will subject the model employed in this study to a close examination.

A physical model of nature helps us to understand the phenomena we are studying. It equips us with a framework of concepts and equations which will have to account for all the characteristics of the subject observed so far. With a model we should be able to interpret the information which experiments have given us, because it provides us with the interrelationships between the various properties of the observed phenomena.

Naturally, for our own sake, we try to keep a model as simple as possible, elegance lying in a simple model explaining a myriad of properties and even predicting phenomena heretofore not observed or even guessed at. However, as science progresses and observations are refined and extended, new matters may be brought to light which do not fit into the existing framework. We then either have to abandon the model altogether, devising a new one, or we have to alter and improve the existing one to incorporate the new facts.

An example of the latter course of science is the gradual sophistication of turbulence models presented in Chapter 3. But even the second order models mentioned there do possess a fair amount of simplifications and do neglect minor effects, just to keep things manageable. Some fundamental simplifications were already encountered in the derivation of Eqs. (2.26)-(2.28). The assumptions mentioned in that Chapter are commonly accepted and need not bother us here. We will concentrate therefore to the specific simplifications incorporated in the present model, some of which were discussed already in Chapter 3.

In section 4.1 a brief outline of the model is presented. By taking notice of the simplifications made beforehand one should be

able to appreciate the limitations inherent in the present model. In sections 4.2 and 4.3 the model's functioning will be analysed in homogeneous and inhomogeneous situations respectively.

4.1 DESCRIPTION OF THE MODEL

4.1.1 General

As we mentioned in Chapter 3 the model of Wyngaard, Coté and Rao (1974) was first used in 1974 (Rao et al., 1974a) to describe the structure of the IBL after a sudden change in surface roughness without any simultaneous change in surface temperature, heat flux etc. In the same year (Rao et al., 1974b) the extension in order to include these simultaneous changes was made. Model results compared favourably with the measurements of Rider et al. (1963). With a further extension (Wyngaard and Coté, 1974) comprising the whole of the ABL, results were obtained which agreed very well with the ones given by the 3-dimensional model of Deardorff (Deardorff, 1972). A detailed description of the model of Rao, Wyngaard and Coté was given by Wyngaard et al. (1974), who discussed the techniques used to close the equations for the second moments (Lumley and Khajeh-Nouri, 1974) as well as the boundary conditions and the numerical techniques. A brief summary of this model is presented in section 4.1.2.

In order to apply a model to a particular situation one usually has to modify certain parts of it. It is of course possible to change the closure assumptions using recently introduced ideas, but this is beyond the scope of the present study. Furthermore, as there is still no general agreement on the modeling of all the third order and pressure-strain terms, the modeling expressions of the original model were not fundamentally changed. The only changes were some buoyancy corrections as well as some boundary conditions and the constants of the model which are determined by these boundary values.

4.1.2 Model equations

Equations (2.29) and (2.34)-(2.36) are the equations of the steady state 2-dimensional mean field in the IBL. As was noticed in Chapter 2 they do not comprise a closed set of equations. In the second order modeling employed in the model of Wyngaard, Coté and Rao equations must be generated for the second moments: \overline{uw} , $\overline{w\theta}$ and \overline{wq} . The procedure to obtain these equations has been presented many times (e.g. Monin and Yaglom, 1971; Tennekes and Lumley, 1972) and hence an abridged version of this derivation will be presented here. We refer to the above mentioned literature for details.

Equations (2.25)-(2.28) for the mean field variables U_i , θ and Q were obtained after subjecting the respective equations for the corresponding, instantaneous variables \tilde{u}_i , $\tilde{\theta}$ and \tilde{q} (2.12), (2.17), (2.22) and (2.23) to the Reynolds' decomposition and averaging. Now if (2.25)-(2.28) are subtracted from (2.12), (2.17), (2.22) and (2.23) the equations for the turbulent field variables u_i , θ and q are obtained. We will not present these equations in this text for brevity, but they are derived and presented in Appendix 1. The equation for the Reynolds' stress tensor $\overline{u_i u_k}$ can be generated by multiplying the equation for u_i with u_k , adding the equation for u_k multiplied with u_i , and averaging the result. This procedure is outlined in Appendix 1. The resulting equation for steady state conditions reads

$$\begin{aligned}
 & \underbrace{U_j \frac{\partial}{\partial x_j} \overline{u_i u_k}}_{\text{I}} + \underbrace{\overline{u_j u_k} \frac{\partial U_i}{\partial x_j}}_{\text{II}} + \underbrace{\overline{u_j u_i} \frac{\partial U_k}{\partial x_j}}_{\text{II}} = - \underbrace{\frac{\partial}{\partial x_j} (\overline{u_i u_j u_k})}_{\text{IV}} + \\
 & - \underbrace{\frac{1}{\rho_r} (\overline{u_k \frac{\partial p}{\partial x_i}} + \overline{u_i \frac{\partial p}{\partial x_k}})}_{\text{V}} + \underbrace{\frac{1}{T_r} (g_i \overline{u_k \theta_v} + g_k \overline{u_i \theta_v})}_{\text{III}} - \underbrace{\frac{2}{3} \bar{\epsilon} \delta_{ik}}_{\text{VI}} + \\
 & - 2\Omega_j (\epsilon_{ijl} \overline{u_l u_k} + \epsilon_{kjl} \overline{u_l u_i}).
 \end{aligned} \tag{4.1}$$

This equation can also be obtained by first generating the equation for the instantaneous Reynolds' stress $\tilde{u}_i \tilde{u}_k$, subtracting

Table 4.1. Scheme of interactions

variable equation	W	U	\emptyset	Q	\overline{uw}	$\overline{w\theta}$	\overline{wq}	$\overline{u^2}$	$\overline{v^2}$	$\overline{w^2}$	$\overline{u\theta}$	\overline{uq}	$\overline{\theta^2}$	$\overline{q^2}$	$\overline{q\theta}$	$\overline{\epsilon}$
(2.29) Cont.	x	x														
(2.34) U	x	\emptyset			x											
(2.35) \emptyset	x	x	\emptyset			x										
(2.36) Q	x	x		\emptyset			x									
(A3.15) \overline{uw}		x			\emptyset			x								x
(A3.12) $\overline{u^2}$		x			x			\otimes		x						x
(A3.13) $\overline{v^2}$								x	\emptyset	x						x
(A3.14) $\overline{w^2}$								x	x	\emptyset						x
(A3.16) $\overline{u\theta}$		x			x			x	x	x	\emptyset					x
(A3.17) $\overline{w\theta}$								x	x	x			x			x
(A3.18) $\blacktriangleright \overline{uq}$		x			x			x	x	x		\otimes				x
(A3.19) \overline{wq}								x	x	x				\otimes		x
(A3.21) $\overline{\theta^2}$								x	x	x			\emptyset			x
(A3.22) $\blacktriangleright \overline{q^2}$								x	x	x				\otimes		x
(A3.20) $\overline{q\theta}$								x	x	x					\otimes	x
(A3.23) $\overline{\epsilon}$		x			x			x	x	x						\otimes

0 : variable in own equation

\square : introduced through moisture correction

\blacktriangleright : equations that will vanish if no moisture correction is used (weak coupling).

from it the equation for $U_i U_k$, and subsequently averaging the result (see Monin and Yaglom, 1971).

The equation for $\overline{u_i \theta}$ can be derived by first multiplying the equation for θ with u_i , then multiplying the equation for u_i with θ , adding the two equations and averaging. Assuming stationarity again we obtain (see Appendix 1)

$$\begin{aligned} U_j \frac{\partial}{\partial x_j} \overline{u_i \theta} + \overline{u_i u_j} \frac{\partial \theta}{\partial x_j} + \overline{u_j \theta} \frac{\partial U_i}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_i u_j \theta}) - \frac{1}{\rho_r} \overline{\theta \frac{\partial p}{\partial x_i}} + \\ \text{I} \qquad \qquad \text{II} \qquad \qquad \text{II} \qquad \qquad \text{IV} \qquad \qquad \text{V} \\ g_i \frac{\overline{\theta \theta_v}}{T_r} + -2\Omega_j \varepsilon_{ijk} \overline{u_k \theta} . \end{aligned} \quad (4.2)$$

III

An analogous procedure using the equation for q instead of θ yields the equation for $\overline{u_i q}$

$$\begin{aligned} U_j \frac{\partial}{\partial x_j} \overline{u_i q} + \overline{u_i u_j} \frac{\partial Q}{\partial x_j} + \overline{u_j q} \frac{\partial U_i}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_i u_j q}) - \frac{1}{\rho_r} \overline{q \frac{\partial p}{\partial x_i}} + \\ \text{I} \qquad \qquad \text{II} \qquad \qquad \text{II} \qquad \qquad \text{IV} \qquad \qquad \text{V} \\ g_i \frac{q \theta_v}{T_r} + -2\Omega_j \varepsilon_{ijk} \overline{u_k q} . \end{aligned} \quad (4.3)$$

III

Equations (4.2) and (4.3) contain the second moments $\overline{\theta \theta_v}$ and $\overline{q \theta_v}$. Realizing that $\theta_v = (1 + 0.608q)\theta$ in accordance with Eq. (2.18) we see that we need three additional equations viz. those for the second moments $\overline{\theta^2}$, $\overline{q^2}$ and $\overline{q\theta}$ (see Table 4.1). They can be derived from the equations for q and θ (A1.12) and (A1.14) with the same method as employed before (see Appendix 1).

$$U_j \frac{\partial}{\partial x_j} \overline{\theta^2} + 2 \overline{u_j \theta} \frac{\partial \theta}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_j \theta^2}) - 2\overline{\varepsilon_\theta} , \quad (4.4)$$

I \qquad \qquad \text{II} \qquad \qquad \text{IV} \qquad \qquad \text{VI}

$$U_j \frac{\partial}{\partial x_j} \overline{q^2} + 2 \overline{u_j q} \frac{\partial Q}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_j q^2}) - 2\overline{\varepsilon_q} , \quad (4.5)$$

I \qquad \qquad \text{II} \qquad \qquad \text{IV} \qquad \qquad \text{VI}

$$\begin{array}{ccccccc}
 U_j \frac{\partial}{\partial x_j} \overline{q\theta} & + & \overline{u_j q} \frac{\partial \theta}{\partial x_j} & + & \overline{u_j \theta} \frac{\partial Q}{\partial x_j} & = & - \frac{\partial}{\partial x_j} (\overline{u_j q \theta}) - \overline{\varepsilon} q \theta \\
 \text{I} & & \text{II} & & \text{II} & & \text{IV} \quad \text{VI}
 \end{array} \quad (4.6)$$

As was mentioned in the previous chapter new unknowns appear in Eqs. (4.1)-(4.6) which prevent us from obtaining a closed system of equations. We will have to order all the terms of (4.1)-(4.6) systematically and devise modeling assumptions for those terms which contain unknown dependent variables. Inspection of (4.1)-(4.6) shows that several corresponding terms can be distinguished. They will be briefly discussed.

Advection terms

All terms in (4.1)-(4.6) denoted by I read

$$U_j \frac{\partial}{\partial x_j} M = U \frac{\partial M}{\partial x} + V \frac{\partial M}{\partial y} + W \frac{\partial M}{\partial z}, \quad (4.7)$$

where M is $\overline{u_i u_k}$, $\overline{u_i \theta}$, $\overline{u_i q}$, $\overline{\theta^2}$, $\overline{q^2}$ and $\overline{q\theta}$ respectively. These terms represent the divergence of the transport $U_j M$ and thus the net increase or decrease of the quantity M per unit volume and unit time in stationary conditions for a fluid particle following the flow. In that case the substantial derivative reads

$$\frac{dM}{dt} = \frac{\partial M}{\partial t} + U_j \frac{\partial M}{\partial x_j} = U_j \frac{\partial M}{\partial x_j},$$

because $\partial M / \partial t = 0$.

Shear production terms

In a manner analogous to the one we used to obtain (4.1)-(4.6) for $\overline{u_i u_k}$, $\overline{u_i \theta}$ etc. equations for $u_i u_k$, $U_i \theta$ etc. can be derived. No derivation is given here, only the resulting equation for $U_i U_k$ will be presented (see Monin and Yaglom, 1965)

$$\begin{aligned}
& U_j \frac{\partial}{\partial x_j} U_i U_k + \frac{\partial}{\partial x_j} \left[U_k \overline{u_i u_j} + U_i \overline{u_k u_j} \right] - \left[\overline{u_j u_k} \frac{\partial U_i}{\partial x_j} + \overline{u_j u_i} \frac{\partial U_k}{\partial x_j} \right] = \\
& - \frac{1}{\rho_r} \left[U_k \frac{\partial P}{\partial x_i} + U_i \frac{\partial P}{\partial x_k} \right] + \frac{T_v}{T_{vr}} \left[g_i U_k + g_k U_i \right] - 2\Omega_j \left[\epsilon_{ijk} U_l U_k + \right. \\
& \left. + \epsilon_{kjl} U_l U_i \right] + \nu \frac{\partial^2}{\partial x_j \partial x_j} U_i U_k - 2\nu \frac{\partial U_i}{\partial x_j} \frac{\partial U_k}{\partial x_j} . \quad (4.8)
\end{aligned}$$

If the equations for $\overline{u_i u_k}$ and $U_i U_k$ are compared we observe that the terms

$$\overline{u_j u_k} \frac{\partial U_i}{\partial x_j} + \overline{u_j u_i} \frac{\partial U_k}{\partial x_j}$$

appear in both equations but with opposite signs. This leads to the conclusion that these terms describe the exchange of $\tilde{u}_i \tilde{u}_k$ between the mean and fluctuating motion. In a turbulent surface layer these terms usually represent a gain for the turbulent motion and consequently a loss for the mean motion. Hence they are called the shear production terms. For the remaining terms labelled II in the other equations, similar conclusions can be drawn.

Buoyant production terms

When the atmosphere is neutrally stratified the term in the u_i -equation (A2.7) containing g_i is equal to zero and hence the terms in (4.1)-(4.3) labelled III are equal to zero. When stratification is present however, these terms represent the production or destruction (depending on the sign of the stability) of a second moment.

Turbulent transport terms

Following e.g. Tennekes and Lumley (1972) we notice that the terms labelled IV in (4.1)-(4.6) all have the form $\partial \overline{Mu_j} / \partial x_j$, where M again is $\overline{u_i u_k}$, $\overline{u_i \theta}$, $\overline{u_i q}$ etc. Gauss' integral theorem states that

$$\int \frac{\partial}{\partial x_j} (\overline{Mu_j}) dV = \oint (\overline{Mu_j}) \cdot dS ,$$

where $\overline{Mu_j}$ is a vector field. This implies that the turbulent transport terms disappear when they are integrated over a volume on which surface the third order moments $\overline{Mu_j}$ are zero. Apparently these terms merely tend to redistribute the quantity M from one point inside the flow to another.

Modeling these terms with the technique of Lumley and Khajeh-Nouri (1974) would greatly complicate the present model, for Lumley's expressions relate several third order moments to each other. Hence we would be forced to calculate the third order terms, using a simple expression containing only second and lower order moments, and subsequently calculate all third order terms iteratively. To avoid an excessive growth of computer time only the original simple "ad hoc gradient diffusion" model as Wyngaard (1975) calls it was used i.e.

$$\overline{Mu_i} = -a_t \frac{\partial M}{\partial x_j} \overline{u_i u_j} \tau , \quad (4.9)$$

where a_t is a constant and $\tau = \overline{u_i u_i} / \epsilon$ is a turbulent relaxation time. For $M = u_i u_k$ Eq. (4.9) is replaced by a more complex expression given in Appendix 3.

Pressure terms

Term V in (4.1) can be transformed as follows

$$\overline{u_k \frac{\partial p}{\partial x_i}} + \overline{u_i \frac{\partial p}{\partial x_k}} = \frac{\partial}{\partial x_i} \overline{pu_k} + \frac{\partial}{\partial x_k} \overline{pu_i} - \left(\overline{p \frac{\partial u_k}{\partial x_i}} + \overline{p \frac{\partial u_i}{\partial x_k}} \right) . \quad (4.10)$$

Its contribution to the turbulent kinetic energy budget can be obtained by letting $i = k$ and summarizing. This yields

$$\text{pressure term} = 2 \frac{\partial}{\partial x_i} \overline{pu_i} - 2 \overline{p \frac{\partial u_i}{\partial x_i}} .$$

The second term on the right hand side of the last relation

equals zero by virtue of (A1.3). The first term, like the turbulent transport terms, is the divergence of a vector field and hence only redistributes the turbulent kinetic energy. No kinetic energy can be generated or destroyed by this pressure terms and its contribution to the turbulent kinetic energy equation is not important.

Next however we turn to the separate components $\overline{u_i^2}$ of the turbulent kinetic energy. Analysis of homogeneous two-dimensional flows shows that in that case the pressure terms tend to force the $\overline{u_i^2}$ - distributions towards isotropy and hence it plays an important role in the $\overline{u_i^2}$ - equations. Moreover it can be shown that the pressure terms destroy the off-diagonal components of the $\overline{u_i u_k}$ tensor (Hinze, 1959). We also have to consider the pressure terms in (4.2) and (4.3). Following Wyngaard (1982) (4.2) reduces to approximately

$$\overline{w^2} \frac{\partial \theta}{\partial z} = - \frac{1}{\rho_r} \overline{\theta \frac{\partial p}{\partial z}}, \quad (4.11)$$

in a horizontally homogeneous, near neutral surface layer in quasi steady state conditions. It shows that the pressure term must destroy the temperature flux at the same rate as it is produced by the temperature gradient. A similar behaviour of the pressure term in (4.3) can be expected.

The state of modeling the pressure terms in (4.1)-(4.3) is converging somewhat to generally accepted expressions, but no general agreement exists as yet. By the time the present model was developed no buoyancy and mean strain effects were incorporated in the pressure term model. Realization that these effects are necessary, especially in moderate to strong diabatic boundary layers, soon emerged (Wyngaard and Coté, 1974; Wyngaard, 1975). The expressions used by Wyngaard (1975) are presented in Appendix 3 along with their forms used in the present model.

Dissipation terms

The dissipation terms labeled VI indicate the molecular smoothing of the structure of the turbulent correlations. Even in high-Re-

number flow, we expect viscous dissipation to be the major loss term for turbulent kinetic energy. When the Reynolds number is large one may assume that these terms are independent of the mean flow geometry. In Appendix 1 it was demonstrated that the dissipation terms only affect the variances $\overline{u^2}$, $\overline{v^2}$, $\overline{w^2}$, $\overline{q^2}$ and $\overline{\theta^2}$ and also the covariance $\overline{q\theta}$. The off-diagonal elements of the Reynolds stress tensor $\overline{u_i u_k}$ as well as the turbulent fluxes $\overline{u_i \theta}$ and $\overline{u_i q}$ are not affected at all. The dissipation terms play an important role in equations (4.1) and (4.4)-(4.6).

In the atmospheric turbulent flow, the Re-number is always high enough to justify that the breakup rate of the large eddies should be independent of ν . Hence it appears dimensionally correct to have

$$\overline{\epsilon} = c \cdot \frac{\overline{u_i u_k}}{\tau} \delta_{ik} , \quad (4.12)$$

where the dissipation term is modeled as an isotropic term, because the viscous dissipation process is an isotropic process. The corresponding models for the destruction terms $\overline{\epsilon_\theta}$, $\overline{\epsilon_q}$ and $\overline{\epsilon_{q\theta}}$ are

$$\overline{\epsilon_\theta} = b \frac{\overline{\theta^2}}{\tau}, \quad \overline{\epsilon_q} = b \frac{\overline{q^2}}{\tau}, \quad \overline{\epsilon_{q\theta}} = b \frac{\overline{q\theta}}{\tau}, \quad (4.13a,b,c)$$

where $b = 1.9$.

In the present model $\overline{\epsilon}$ is not approximated by (4.12) but it is calculated from a dynamic equation obtained by modeling its governing equation. We will omit the derivation of this equation (Lumley and Khajeh-Nouri, 1974) and simply present their equation

$$u_j \frac{\partial \overline{\epsilon}}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_j \epsilon}) - 4 \frac{\overline{\epsilon}}{\overline{u_i u_i}} (\overline{\epsilon} - aP) , \quad (4.14)$$

where P is the production rate of turbulent kinetic energy.

The turbulent transport term in (4.14) is modeled by the approxi-

mation given in equation (4.9).

Summarizing, we can say that the present model consists of four mean flow equations (2.29) and (2.34)-(2.36), the equations for the second moments (4.1)-(4.6) and one equation for the dissipation rate $\bar{\epsilon}$ viz. (4.14), with all the modeling assumptions mentioned throughout the text. In all we have 16 coupled parabolic partial differential equations for the 16 dependent variables (see also Table 4.1): U , W , θ , Q , $\overline{u^2}$, $\overline{v^2}$, $\overline{w^2}$, \overline{uw} , $\overline{w\theta}$, $\overline{u\theta}$, \overline{wq} , \overline{uq} , $\overline{\theta^2}$, $\overline{q^2}$, $\overline{q\theta}$ and $\bar{\epsilon}$. These equations are numerically integrated using a Dufort-Frankel explicit finite-difference scheme (Dufort and Frankel, 1953), with forward marching in the x-direction.

The model is divided in two parts. In part I the initial distribution of all variables in the surface layer upstream of the discontinuity is calculated, using predetermined values of u_{*1} , T_{*1} , q_{*1} , z_{01} , θ_{01} , Q_{01} and z_I (height of the ABL).

In part II the final distributions of all variables as calculated in Part I are used as the initial values for the calculation of these distributions downstream of the discontinuity. In part II the surface value of every variable is a function of the downstream distance (x) and the only restriction at the surface is imposed by the energy and water balance (see Appendix 3). Contrary to this the surface value of every variable in Part I remains fixed.

4.1.3 Drawbacks

Because the present model contains a great number of equations which describe many variables in detail, it necessarily must contain a large number of modeling constants which are needed to produce the desired characteristics of turbulence is the ASL. The larger the number of constants in a given model the larger the amount of empirical input needed to determine the values of these "constants". This poses a limitation on the general applicability of the model i.e. the values of the constants

have to be tuned to any change in characteristics which the user of the model deems necessary. Recently (Bradley, Antonia and Chambers, 1982) doubts arised for instance on the model constant involved in the approximation of the pressure term in the modeled $u_{1\theta}$ equation. Bradley et al. (1982) showed that the value of this constant depends on the stability parameter z/L .

There are other simplifications in the present model which are not mentioned by the designers, but which are important. The first simplification incorporated in the model is that for a SR change the lowest level of the numerical grid is placed at the largest z_0 -height. This means that the airflow below this level for $x < 0$ does not influence the calculations for $x > 0$. Consequently it implies that the momentum, energy and humidity contents between z_{01} and z_{02} are neglected for the downstream region. As the original model was applied to a SR change from $z_{01} = 2 \cdot 10^{-5}$ m to $z_{02} = 1.4 \cdot 10^{-3}$ m, this approximation could easily be made, but it certainly limits the applicability of the model for roughness changes when $z_{02} - z_{01}$ is larger.

The second simplification is the omission of the pressure gradient term in the equation of motion (2.34). This omission results in an important numerical simplification for it turns an elliptical partial differential equation into a much easier to handle parabolic one. This was pointed out by Rao et al. (1974a) who based their decision to neglect the pressure field on results obtained by Onishi and Estoque (1968). They demonstrated that the omission of the pressure gradient term has no significant consequences for great downstream distances. Only in the immediate vicinity of the roughness change ($\frac{x}{z_{02}} < 10^3$) will it have any effect. For greater distances downstream no pressure effect is noticeable.

The lower boundary condition after the step change assumes the independence of both the surface relative humidity R_{02} and the difference $R_n - G$ (Eqs. (A3.27) and (A3.25) of downstream dis-

tance. Rao et al. (1974a) commented on the latter assumption arguing that this simplification is only a crude approximation which in practice seems to be quite reasonable. They did not comment on the former assumption of the constant value of R_{02} however. This parameter is, together with z_{02} , the only externally imposed quantity which does not change after the step in surface conditions. The assumption of a constant value of R_{02} can lead to a non-physical behaviour of the solution when a change occurs from a cold and humid surface to a warm and dry one, as will be shown in section 4.3.2. For the reverse change this problem does not arise (see also Appendix 4).

Also, it should be mentioned that in order to define the mean horizontal wind speed throughout the entire flow it is necessary to impose the logarithmic wind profile condition down to $z = z_0$, which is very doubtful to say the least. On the other hand, it is possible that this approximation will only give a minor deviation limited to the lowest layers near $z = z_0$.

With regard to the various modeling assumptions employed in the present model, we must realize that there is much that still has to be learned about the distribution and dynamics of the third moment-and pressure-strain terms. Until more insight is gained in this matter we must accept the sometimes drastic simplifications and crude assumptions which need to be made in order to arrive at manageable expressions. However, the closure approximation of the third moment (4.9) remains unsatisfactory.

Wyngaard et al. (1974) use "a rather simpler, ad hoc gradient diffusion model", although Wyngaard (1975, 1982) does not fail to notice that in the unstable surface layer this approximation probably will fail. Even in Wyngaard's study of the convective PBL (Wyngaard and Coté, 1974) and in a similar study of the stable PBL (Wyngaard, 1975) he stresses this obvious deficiency of the modeling approximation, but no attempt has been made to overcome this imperfection.

Finally, the arguments that are used by Rao et al. to compute

the step in surface heat flux at $x = 0$ remain rather vague. The authors refer to a comment made by Sibbons (Rider et al., 1965) in which he proposes a method to calculate the surface heat flux just after $x = 0$. It is unfortunate that Sibbons also does not clearly point out how he calculated this value. However, calculations made in the present study revealed that the value of the surface heat flux at $x = 0$ only determines a parameter which is used to nondimensionalize every variable.

4.2 HOMOGENEOUS SITUATIONS

Before applying the present model to inhomogeneous situations its capability to handle homogeneous situations should be tested. This means that results should be compared with experimental data and existing views on the structure of the atmospheric surface layer above homogeneous terrain. Only if the model is able to handle such a less complex situation reasonably well, it will be possible to have any faith in its performance in inhomogeneous situations.

In this section the performance of the model in homogeneous situations will be analysed in two ways. First the equilibrium distribution of the variables calculated in Part I of the model will be checked with experimental data. Second the contribution of every term in the equations for the second moments will be calculated and compared with experimentally determined budgets of the second moment equations.

4.2.1 Equilibrium distributions

Numerous experiments have been conducted in the past to establish the distribution of first and second moments in the atmospheric surface layer above homogeneous terrain. In Chapter 1 we mentioned the best efforts known today. These experiments provided the data which were needed to determine many of the constants in the formulae describing the equilibrium distributions in the Monin-

Obukhov similarity framework (Businger et al., 1971; Yaglom, 1977; Viswanadham, 1982).

Through the years, experimental methods and instrumentation improved, and a gradual convergence to the range of values of the empirical constants accepted today has taken place. These values and formulae were used to define the lower and upper boundary conditions in the present model (see Appendix 3). When the values of all variables at the upper and lower boundaries are fixed, it is expected that the calculated equilibrium profiles must agree with empirical curves. However, deviations from the empirical curves between the fixed boundary values might still occur. To avoid an exhaustive discussion of the equilibrium distribution of all the dependent variables in the present model, a restriction is made to those quantities which are commonly used to measure flux densities.

In part I of the model the vertical turbulent flux densities of momentum, heat and water vapour are calculated. According to equation (A3.24) their dimensionless value at the surface equals -1, while the value at the upper boundary (A3.28) agrees with their linear decrease with height to zero at z_T (i.e. the height of the ABL). As is evident from Figure 4.1 the distributions of the three dimensionless flux densities are identical and indeed vary linearly with height to a very good degree.

Figure 4.2 presents the variation of the dimensionless gradient of the horizontal wind speed ($\frac{kz}{u_{*1}} \frac{\partial U}{\partial z}$) and the temperature ($\frac{kz}{T_{*1}} \frac{\partial \theta}{\partial z}$) with height. The dimensionless gradient of the absolute humidity is omitted since it coincides with $\frac{kz}{T_{*1}} \partial \theta / \partial z$. This can be inferred from (A3.17) and (A3.19) and the equality of the $\overline{\theta \theta_v} / T_{*1}^2$ and $\overline{q \theta_v} / q_{*1} T_{*1}$ distributions. Figure 4.2 shows that in a moderately unstable atmosphere the curve of the dimensionless temperature gradient coincides with the empirical ϕ_h -curve to a very good degree. The dimensionless gradient of the horizontal

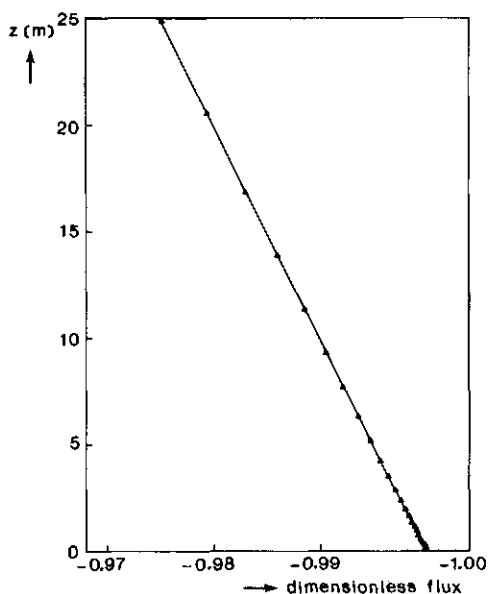


Figure 4.1 Vertical equilibrium distribution of the nondimensional turbulent fluxes of momentum, sensible and latent heat.

wind speed, however, deviates from the expected ϕ_m -curve, especially for large heights hence for large values of $|z/L|$. This deviation points to two possible causes. Firstly, it indicates that at least one of the modeling assumptions lacks a correction for buoyancy effects. Indeed, if we look at the modeling expressions used by Wyngaard and Coté (1974) and by Wyngaard (1975), for an unstable and stable atmosphere respectively, we see that they also had to add buoyancy and mean strain corrections in order to arrive at physically realistic results. Though their studies concentrated on the whole of the PBL, instead of only the atmospheric surface layer, a fair amount of their grid levels were situated in the latter layer because their vertical grid spacings increased logarithmically with height. Efforts in the present study to add similar corrections to the modeling expressions employed in the present model have not yet produced

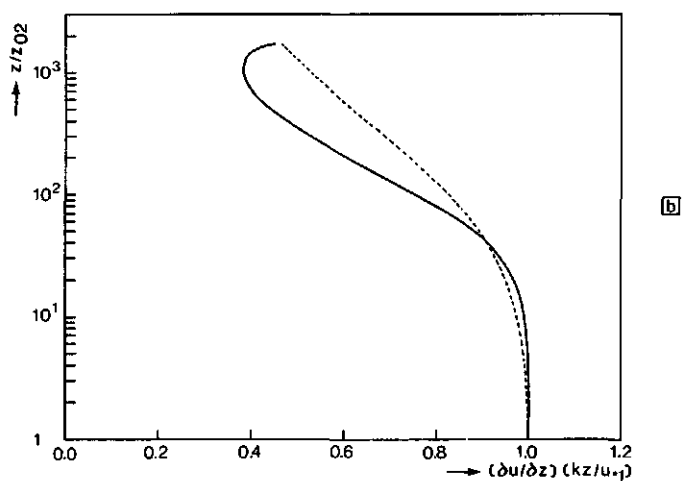
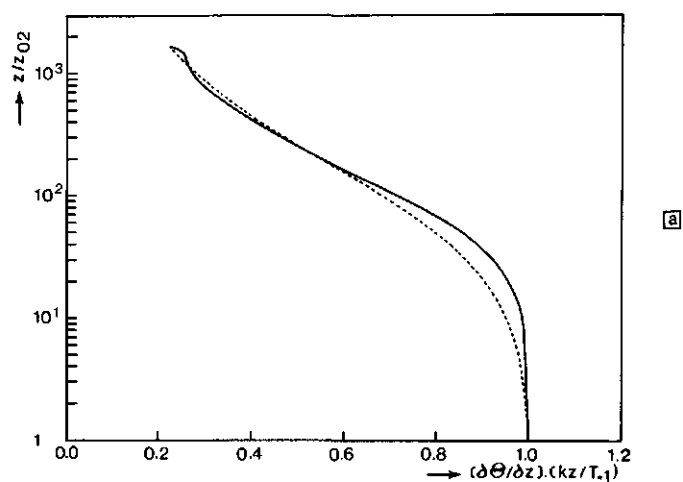


Figure 4.2 Distribution of the dimensionless gradient of temperature (a) and horizontal wind speed (b).

a: ---- ϕ_h , — $(\partial\theta/\partial z) \cdot (kz/T_{*1})$

b: ---- ϕ_m , — $(\partial U/\partial z) \cdot (kz/u_{*1})$

entirely satisfactory results.

The second possible cause is the definition of the upper boundary conditions. The model constants that appear as a factor in front of the dissipation and destruction terms have been determined by invoking the equality of both sides of the equations at the lower boundary. At the upper boundary, however, the equality of both sides of the equations need no longer be present, for the buoyancy terms are also important there. This inequality might force the gradient $\partial U/\partial z$ from its expected $\phi_m(z/L)$ distribution.

4.2.2 Assessment of budgets

To transform the set of equations (2.34)-(2.36) and (4.1)-(4.6) and (4.14) to the homogeneous case, it will simply suffice to put $\partial/\partial x = 0$, and by virtue of the equation of continuity (2.29) also $W = 0$. As a result the equations (2.34)-(2.36) read

$$\overline{\partial u w}/\partial z = 0, \quad \overline{\partial w \theta}/\partial z = 0 \quad \text{and} \quad \overline{\partial w q}/\partial z = 0. \quad (4.15)$$

This indicates the existence of a constant flux layer. Applying these conditions to Eqs. (4.1)-(4.6) and (4.14), or rather to the same equations after modeling, the equations for the second moments and for $\bar{\epsilon}$ in the homogeneous case are obtained. These equations are summarized in Appendix 3. Using the equilibrium profiles evaluated in the initialization program (Part I) we calculated the contribution of each term to the equation in which it is incorporated, thus enabling us to analyse the budgets of Eqs. (4.1)-(4.6) and (4.14) in homogeneous situations.

On inspecting (4.1)-(4.6) and (4.14) it appears that production, pressure gradient interaction, dissipation and/or molecular destruction terms are largest near the surface where all gradients are steepest and turbulent interaction is strongest. Going upwards into the atmosphere, most terms quickly diminish in magnitude as gradients are decreasing rapidly. This makes it difficult

to analyse them forthwith. When drawn on a linear height scale, it is next to impossible to gain any further information from the rapidly decreasing values of almost every term. To reveal the details of the budget also at higher levels, we multiply every term of a given equation with a properly chosen height dependent scale. We will adopt the approach of e.g. Wyngaard and Coté (1970), Wyngaard et al. (1971, 1978) and Bradley et al. (1981) who used kz divided by the proper combination of u_* , T_* and q_* .

Experimental determination of the budget of second moment equations is restricted to those of $\overline{\theta^2}$, $\overline{q\theta}$, \overline{uw} , $\overline{w\theta}$, $\overline{u\theta}$ and $\overline{e^2} = \overline{u^2} + \overline{v^2} + \overline{w^2}$. This means that we can only examine the budgets of the corresponding equations from Eqs. (A2.19)-(A2.30). We will restrict ourselves to respectively: $\overline{\theta^2}$, $\overline{u\theta}$, $\overline{e^2}$ and \overline{uw} .

Temperature variance budget

Nondimensionalizing Eq. (A3.21) by multiplication with $kz/u_*T_*^2$ produces

$$\phi_m - \frac{kz}{u_*T_*^2} a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{\theta^2}}{\partial z}) + \frac{kz}{u_*T_*^2} b \frac{\overline{\theta^2}}{\tau} = 0. \quad (4.16)$$

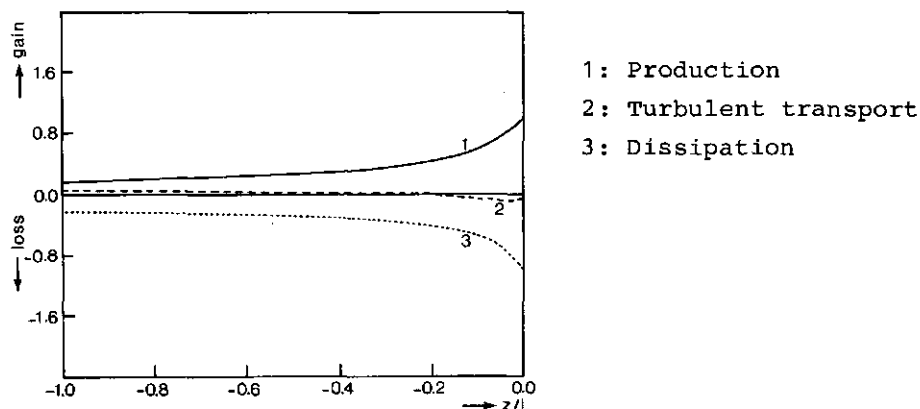


Figure 4.3 Budget of the temperature variance equation.

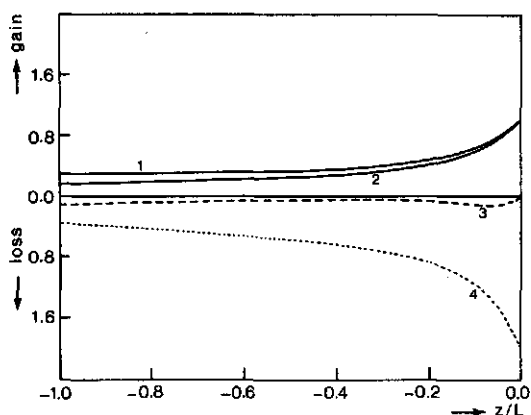
In Figure 4.3 the magnitude of every term of this equation is given as a function of z/L . It is evident that there is an almost perfect balance between the production and dissipation term in moderately unstable and near neutral conditions. It means that the turbulent transport term is at least an order of magnitude smaller than both the production and the dissipation term. This agrees with experimental data from Bradley et al. (1981), Champagne et al. (1977) and Wyngaard and Coté (1971). If it has any significance at all, turbulent transport is a loss in near neutral situations and a gain in moderately unstable conditions. This was also noticed by Wyngaard and Coté (1971), although their data show much scatter and their observation might be considered tentative. For moderately to strong unstable conditions ($-z/L \approx 1$) the ratio production/dissipation attains a value of 0.8 which is in accordance with the data of Champagne et al. (1977) but not with the data of Bradley et al. (1981).

Horizontal heat flux budget

Equation (A3.16) is nondimensionalized with $kz/u_*^2 T_*$ and reads

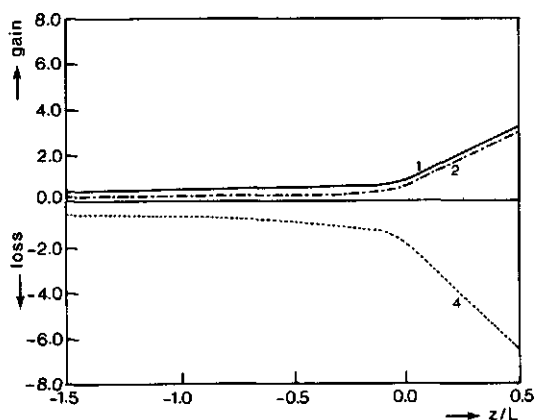
$$\phi_m + \phi_m + \frac{kz}{u_*^2 T_*} a_t \frac{\partial}{\partial z} (\overline{w^2} \frac{\partial \overline{u\theta}}{\partial z}) + \frac{kz}{u_*^2 T_*} d_1 \frac{\overline{u\theta}}{\tau} = 0. \quad (4.17)$$

Figure 4.4a shows the variation of every term of this equation with z/L . The turbulent transport is a small term representing a loss over the entire range of $-z/L$ values considered here. This does not completely agree with the results of Bradley et al. (1981) and Wyngaard et al (1971). Although they also concluded that the turbulent transport term is very small in comparison with the remaining terms of the $\overline{u\theta}$ -equation, they found a different behaviour of this term with $-z/L$. According to Bradley et al. turbulent transport is a loss for near neutral conditions ($-z/L < 0.1$) while beyond this value it represents a gain. Wyngaard et al. found a value of $-z/L \approx 0.3$ for which this change in sign occurs. Figure 4.4 presents the $\overline{u\theta}$ -budget found by Wyngaard et al. (1971) by comparison.



[a]

- 1: Shear production
- 2: Stratification production
- 3: Turbulent transport
- 4: Pressure gradient interaction



[b]

Figure 4.4 Budget of the horizontal heat flux equation;
a. present model, b. Wyngaard et al. (1971).

Turbulent kinetic energy budget

Adding Eqs. (A3.12)-(A3.14) and nondimensionalizing the result with kz/u_*^3 yields

$$\phi_m - \frac{z}{L} + \frac{kz}{u_*^3} a_t \frac{\partial}{\partial z} (\overline{w^2}) - \frac{\partial \overline{e^2}}{\partial z} - \frac{kz}{u_*^3} \overline{\epsilon} = 0. \quad (4.18)$$

As a consequence of the modeling technique applied to the turbulent transport and pressure transport terms (see Appendix 3) it is not possible to separate the contribution of both terms to the TKE budget. This means that the third term in Equation (4.18)

represents the contribution of both terms. The dependence of all terms on z/L is given in Figure 4.5a. An experimental determination of this budget was performed by Wyngaard and Coté (1971) and by Champagne et al. (1977). The results of Wyngaard and Coté are summarized in Figure 4.5b. To facilitate comparison, we have added the contributions of the imbalance term (= pressure transport term) and the turbulent transport term of their measurements. The sum of these contributions represents a gain for the given range of $-z/L$ values. This implies that dissipation exceeds the total production in near neutral to moderately unstable situations. For increasing instability a balance between the total production and

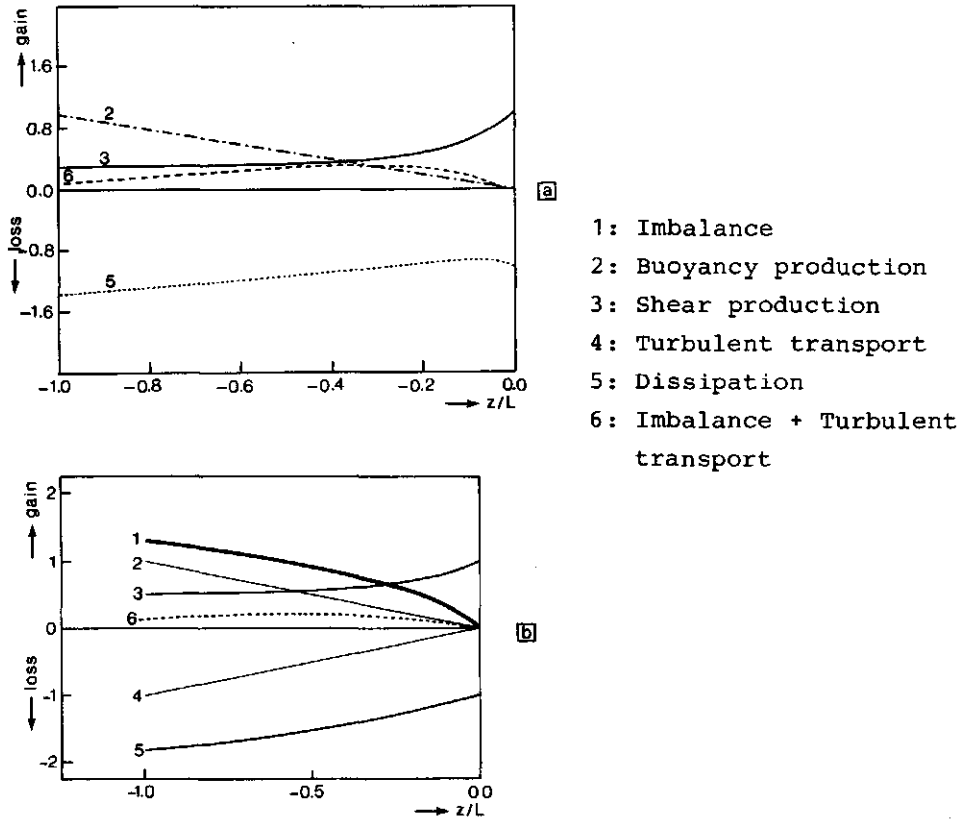


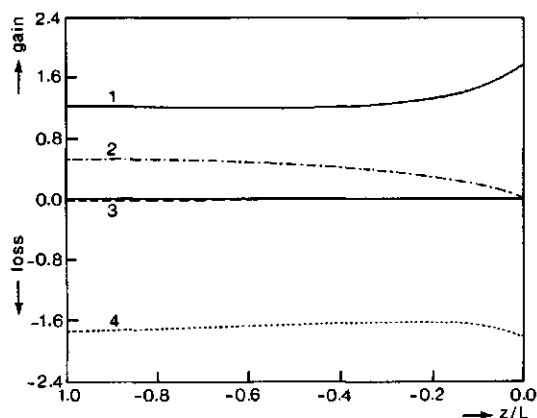
Figure 4.5 Budget of the turbulent kinetic energy equations;
a. present model, b. Wyngaard and Coté (1971).

dissipation is approached for the total transport term diminishes in magnitude. This is consistent with the data from Wyngaard and Coté. Shear production deviates from $\phi_m(z/L)$ by the reasons mentioned before, while buoyancy production follows the expected $-z/L$ behaviour very well.

Shear stress budget

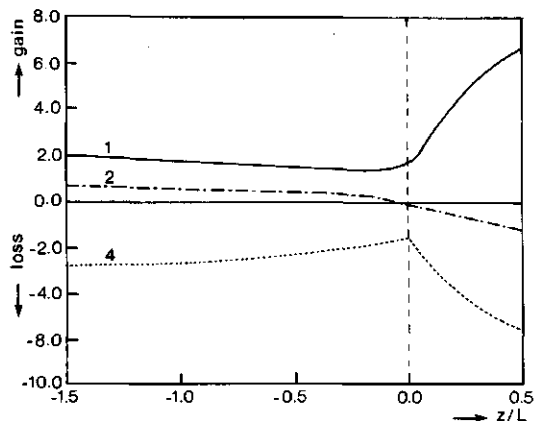
Multiplication of Equation (A3.15) with kz/u_*^3 yields

$$\frac{\overline{w^2}}{u_*^2} \phi_m + \frac{\overline{u\theta}}{w\theta} \frac{z}{L} + \overline{uw} \frac{c}{\tau} \frac{kz}{u_*^2} = 0. \quad (4.19)$$



[a]

- 1: Shear production
- 2: Buoyancy production
- 3: Turbulent transport
- 4: Pressure gradient interaction



[b]

Figure 4.6 Budget of the shear stress equation;
a. present model, b. Wyngaard et al. (1971).

The first two terms represent shear production and buoyancy production respectively. The third term is the approximation for the pressure gradient interaction term, which always represents a loss in the atmospheric surface layer. Buoyancy production is a loss (gain) in stable (unstable) stratification. Shear production will always be a gain. Figure 4.6a presents the magnitude of these terms for unstable situations as calculated with the present model. It compares favourably with the experimental data obtained by Wyngaard et al. (1971) which are summarized in Figure 4.6b. The factor $\overline{u\theta}/\overline{w\theta}$ in the buoyancy production term reduces the importance of this term for increasing values of $-z/L$. Values of the factor $\overline{u\theta}/\overline{w\theta}$ were determined by Wyngaard et al. They found that the magnitude of this factor ranges from 3.4 to 0.6 for $-z/L$ ranging from 0 to 1.0. This agrees with the values calculated with the model viz. they range from 3.0 to 0.4 for the same $-z/L$ interval.

The factor $\overline{w^2}/u_*^2$ in the shear production term is always greater than unity. This implies that the shear production rate of shear stress exceeds the shear production rate of energy (eq. 4.18) over the entire stability range. Moreover, as $\overline{w^2}/u_*^2$ increases with increasing instability, it is obvious that the importance of this production term will not diminish as $-z/L$ increases. Wyngaard et al. actually found an increase of the shear production term for increasing instability, which does not agree with the results of our model, but this is a consequence, once again, of the deviations mentioned in 4.2.1.

4.2.3 Conclusions

The equilibrium profiles in the atmospheric surface layer generated by the present model agree with experimentally determined distributions. This is not surprising since both the upper and lower boundary conditions are chosen to ensure this. However, deviations still occur in moderate to strong unstably stratified boundary layers, where non-monotonic curves appear. These deviations are ascribed to the exclusion of buoyancy corrections in

the closure approximations for the higher order moments and to the inequality of the model equations when the upper boundary conditions are substituted.

The agreement of the budgets of the second moment equations with experimental data is satisfactory, although the scatter in some of the data is rather large and some terms are measured only indirectly. This makes several of our comparisons rather uncertain.

4.3 ANALYSIS OF INHOMOGENEOUS SITUATIONS

4.3.1 Single step in surface conditions

In order to get an appreciation of the quality of performance of the present theoretical model in inhomogeneous situations, a comparison should be made with experimentally obtained data. Model and experiment must of course be carefully tuned to each other. This means that external conditions (two-dimensionality, upstream and downstream roughness lengths, atmospheric stratification) should match as good as possible.

Many comparisons between the theory of relaxing flows and first moments such as the mean horizontal wind speed or the mean temperature have already been made in the past. Thus many times the usefulness of existing models that predict first moments has been demonstrated. We will not repeat such an analysis for the present model, as this would merely lead to the same conclusions drawn by Rao et al. (1974a) and the remarks made in Chapter 3. Furthermore, the gain in accuracy achieved by a second order closure model instead of a less complex first order closure model is primarily to be found in the prediction of the behaviour of the second moments (turbulent fluxes and other covariances). The first moments are already predicted with good accuracy even by the first order closure models.

The above considerations lead inevitably to the conclusion,

already made in section 3.5.3, that we must concentrate our attention on the second moments when analyzing the performance of the present model. In section 3.5.3 it was noticed that reliable measurements of second order properties of turbulence in the atmosphere above non-homogeneous terrain are very hard to come by. In fact, the only data of second moments published till today are the measurements performed by Peterson et al. (1979), Højstrup (1981) and Lang et al. (1983). Figure 4.7 e.g. presents the variation of the horizontal wind variance profile after a SR change in surface roughness. The turbulent kinetic energy model of Peterson (1969) seems to predict the behaviour of

$$\sigma_u = \overline{(u^2)}^{\frac{1}{2}}$$

reasonably well, but this is one of the very few examples where a direct comparison has ever been made.

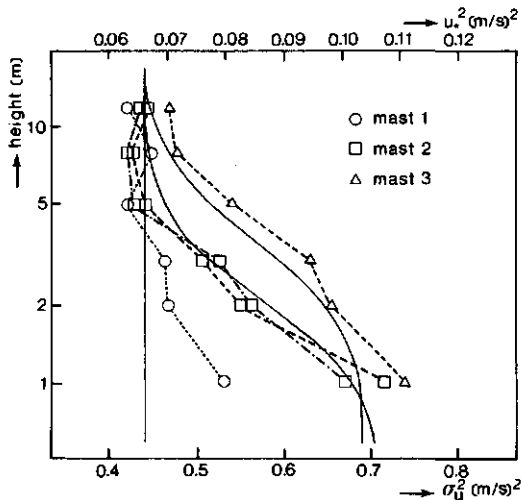


Figure 4.7 Variation of the horizontal wind variance profile after a smooth to rough transition.
 — Peterson's 1969 model. (After Peterson et al., 1979).

Obviously, the scarcity of data of second moments in nonhomogeneous conditions, obtained by measurements in the atmosphere is such that the comparison with model results will not lead to a

definite qualification of the performance of 2nd order models. This is what Bradshaw (1972) called the "fact gap": too many computers chasing too few facts.

This situation can partially be remedied by including windtunnel measurements in our consideration. The carefully designed experiments in a windtunnel often provide us with an abundance of mean and turbulence quantities suitable for model performance evaluation. But the difficulties and deficiencies inherent to the use of wind tunnel data for modeling atmospheric situations are not at all trivial, especially not when using these data for such goals as assessing the predictive qualities of numerical models designed for atmospheric situations.

4.3.2 Consistency of the present model

As experimental verification of the model results is not satisfactorily feasible and awaits the progress and development of experimental methods and instruments, we turn to another prerequisite of any model: it should be consistent with itself. This means that if a given initial equilibrium state of the atmospheric surface layer is subjected to two consecutive changes in surface conditions: the first change to a new set of surface characteristics and the second change back to the original set of surface characteristics, the atmosphere should ultimately return to its original equilibrium. It means that if an equilibrium surface boundary layer encounters a temporary disturbance, the deviations generated by this disturbance should die out as the flow progresses across its original surface.

To this end the distribution of every variable of the model was calculated when it was subjected to the set of boundary conditions presented in Figure 4.8. For $x < 0$ the equilibrium initial profiles were calculated using the predetermined surface parameters as discussed in section 4.1. At $x = 0$ the airflow encounters a surface which is rougher, warmer and dryer than the surface for $x < 0$. After an arbitrary distance of 21 meters the surface con-

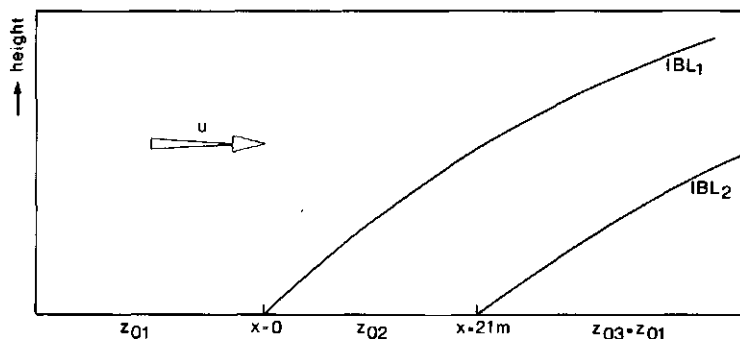


Figure 4.8 Definition sketch of two subsequent changes in surface conditions.

ditions change for the second time and regain their original value. This applies to the surface roughness and the surface relative humidity, the other surface parameters depend on the downstream distance (x).

Two questions are interesting in the given situation. Firstly, do the variable surface parameters (u_* , T_* and q_*) return to the values they possessed at $x < 0$. And if so, how rapidly do these values return. Secondly, what happens to the disturbances introduced into the profile of every variable after the second step change?

Figure 4.9a presents the variation of u_* , T_* and q_* with the downstream distance (x) after the first step at $x = 0$ and after the second step at $x = 21$ m. Along the vertical axis the ratios $u_*(x)/u_{*1}$, $T_*(x)/T_{*1}$ and $q_*(x)/q_{*1}$ are plotted, where u_{*1} , T_{*1} and q_{*1} are the original equilibrium values of the upstream surface ($x < 0$). The three variables u_* , T_* and q_* determine the surface fluxes of momentum, sensible heat and latent heat respectively by virtue of

$$\tau(x) = -\rho \{u_*(x)\}^2 \quad (4.20)$$

$$H_0(x) = -\rho c_p u_*(x) T_*(x) \quad (4.21)$$

$$\lambda E_0(x) = -\rho \lambda u_*(x) q_*(x) \quad (4.22)$$

Starting with $u_*(x)/u_{*1} = 1$, $T_*(x)/T_{*1} = 1$ and $q_*(x)/q_{*1} = 1$ at $x = 0$, these variables are subjected to a change in surface conditions. After the first step at $x \approx 0$ the variables over(under)shoot after which they gradually approach a new equilibrium value, which is different from the original one. This new equilibrium value is not reached however, because at $x \approx 21$ m the second step generates a new over(under)shoot and the variables at the surface gradually adjust once again to the new surface.

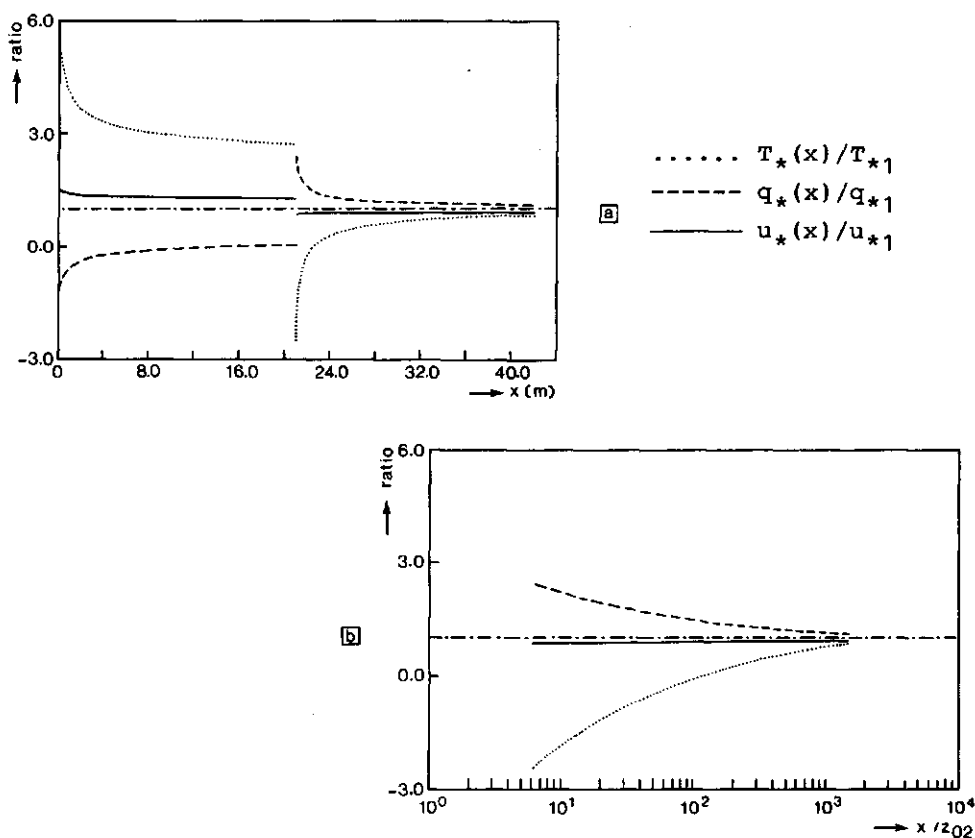


Figure 4.9 Variation of surface parameters for two subsequent changes in surface conditions.

The present model is consistent with itself if, after the second step change the dimensionless surface variables approach their equilibrium value of unity again. This happens indeed as can be seen in Figure 4.9b for $x > 21$ m. The logarithmic scale corresponds with the second part of Figure 4.9a.

The multitude of first and second moment variables in the current model makes a full description of every variable unattractive for reasons of brevity and readability. Therefore, a restriction is made in this section to the behaviour of the profiles of the mean temperature, absolute humidity and those of the vertical fluxes of humidity and heat.

Figures 4.10 and 4.11 present the profiles of the above mentioned variables for (i): the relaxation after the first surface discontinuity and (ii): the relaxation after the second surface discontinuity and the approach of the original equilibrium situation. Figure 4.10 gives the variation of the absolute humidity and the humidity flux for the two different situations. After the first transition the absolute humidity profile adjusts itself in the manner as shown in Figure 4.10a. Especially in the lowest layers the absolute humidity drops because the surface flux of water vapour has suddenly been reduced by the less humid surface. This decrease in absolute humidity gradually affects layers of greater height, and for $x/z_{02} = 1.5 \cdot 10^3$ the change in surface humidity is noticeable up to a height of 0.8 m. At this downstream distance no new equilibrium has yet been reached.

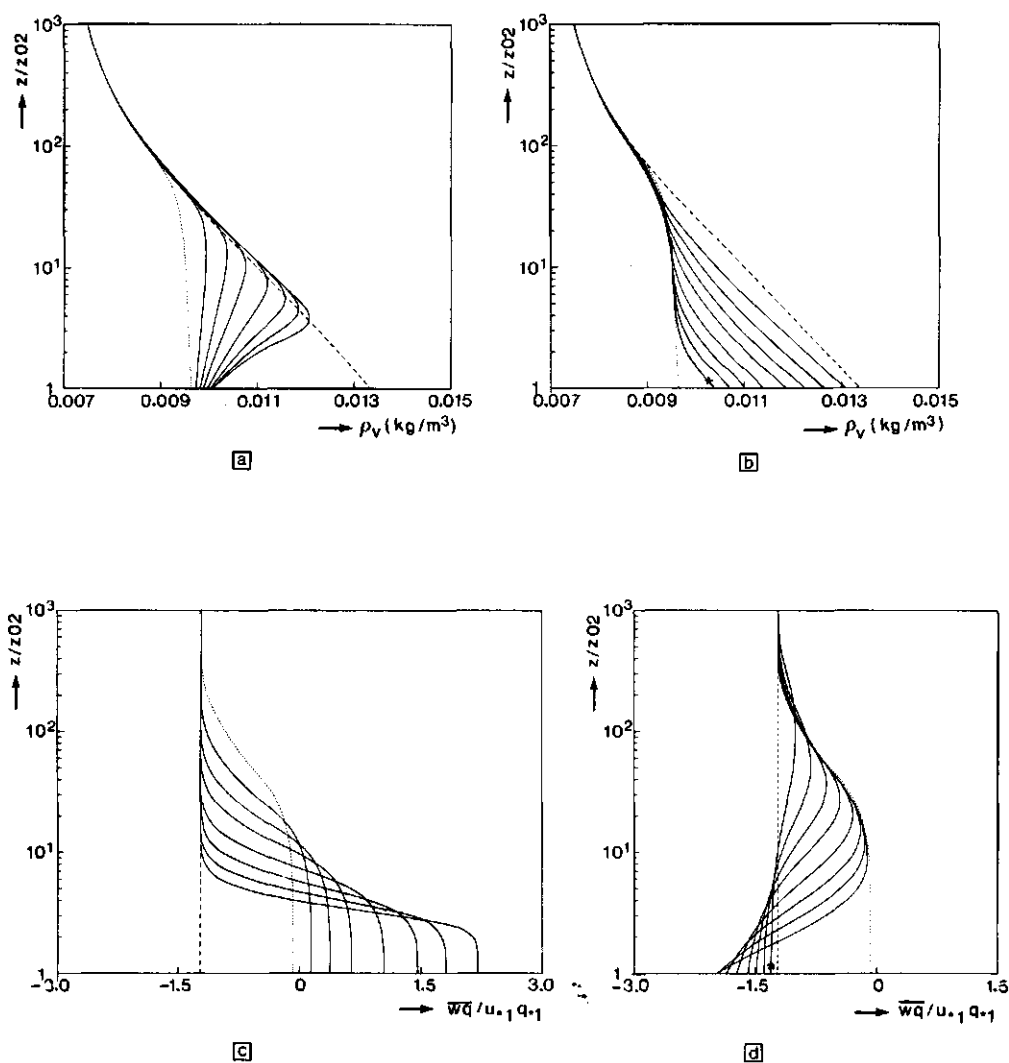
The last profile (curve no.8) in Figure 4.10a serves as the initial profile in Figure 4.10b, which gives the absolute humidity profiles after the second surface transition. At the lowest layers the absolute humidity increases again because of the humid surface. Again this increase diffuses upwards, though not to the highest layers where the decrease due to the first transition is still continuing. No new equilibrium is reached at $x/z_{02} = 3 \cdot 10^3$ but the original profile is approached and the disturbance damps

out as x increases.

From a physical point of view it is not possible to have a positive value of $\partial Q / \partial z$, indicating a negative (i.e. downward) turbulent humidity flux adjacent to the surface. This would imply that water vapour is transported downwards and is absorbed at the surface. This is clearly impossible, for it means that condensation is taking place at the surface, while at the same time the surface relative humidity remains unchanged at $R_{02} < 100\%$. This obvious model deficiency is attributed to the boundary conditions (A3.25) and (A3.27) and was also discussed in section 4.1.3. A more realistic boundary condition was suggested by McNaughton (1984) on the basis of the Penman-Monteith equation, discarding with Eq. (A3.27). This is discussed in Appendix 4.

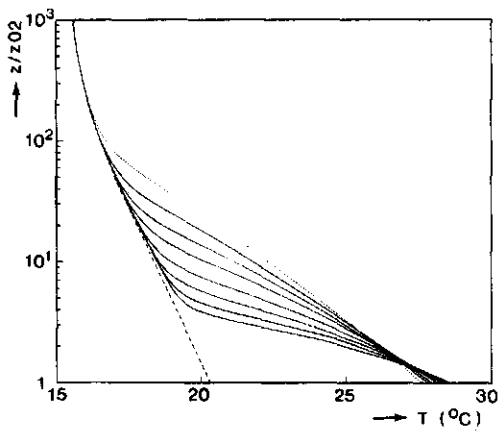
The features encountered in the profiles of the absolute humidity can also be understood in view of the profiles of the vertical humidity flux (Figures 4.10c,d). The factor $u_{*1} q_{*2}$ is used to nondimensionalize this flux. As $q_{*2} < 0$ and $u_{*1} > 0$ the dimensional value of \overline{wq} has a sign opposite from its dimensionless value. Just after the first transition \overline{wq} decreases from its originally positive value to a negative, indicating a reversal of the direction of the flux in the atmosphere just above the surface, depleting these layers of water vapour. As \overline{wq} increases monotonically with z , a decrease of the absolute humidity at every height occurs, in accordance with Figure 4.10a. After some distance downstream the surface value of \overline{wq} becomes positive again (curve no.8), which is reflected in the corresponding curve in Figure 4.10a, where the vertical gradient of the absolute humidity is negative at all heights.

After the second transition the \overline{wq} curve is no longer monotonically increasing but exhibits a minimum value. The part of the curve under this minimum represents an upward flux which decreases with height. At these levels the absolute humidity increases with x , in accordance with Figure 4.10b. The part of the curve above the minimum represents an upward flux which increases with

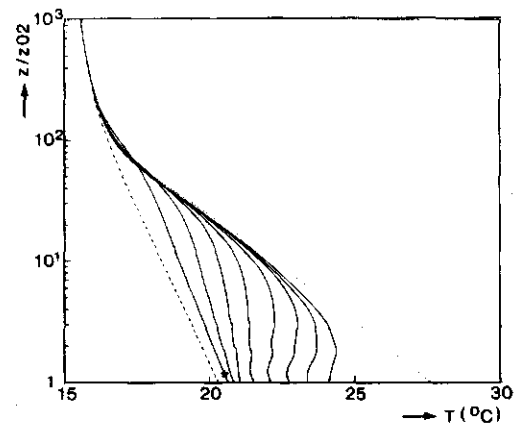


- Profile at $x = 21$ m.
 ----- Initial profile ($x < 0$).
 * Profile at $x = 42$ m.

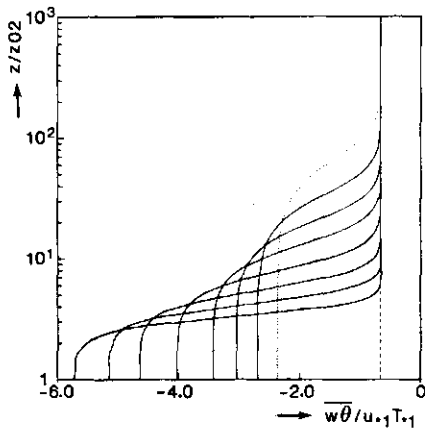
Figure 4.10 Profiles of the absolute humidity and the turbulent flux of water vapour after two subsequent changes in surface conditions for various distances downstream.



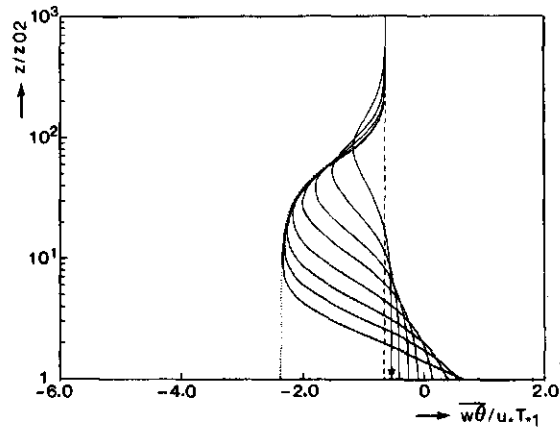
[a]



[b]



[c]



[d]

- Profile at $x = 21$ m.
- Initial profile ($x < 0$).
- * Profile at $x = 42$ m.

Figure 4.11 Profiles of the temperature and the turbulent flux of sensible heat after two subsequent changes in surface conditions for various distances downstream.

height. At these levels the absolute humidity is still decreasing due to the former dryer surface. Further downstream, the \overline{wq} profile returns to its equilibrium distribution where it is constant with height. The disturbance gradually damps out.

Observations, analogous to the ones just made on the absolute humidity and vertical humidity flux profiles can be made with respect to the mean temperature and vertical sensible heat flux profiles which are presented in Figure 4.11. We present this figure for completeness only and will not discuss it.

4.4 SUMMARY

Summarizing the results of the examination of the performance of the present model we can say that in its present state the model contains several imperfections which can be roughly divided into two classes:

- (i) imperfect modeling expressions and/or imperfect upper boundary conditions
- (ii) unrealistic lower boundary conditions after the step change.

The flaws belonging to the first class betray their existence only in diabatic conditions when they produce deviations which are notably marked in the vertical gradient of the mean horizontal wind speed. Adjusting the modeling expressions for buoyancy and mean strain effects introduces a whole new set of numerical constants which must be tuned to obtain the desired results.

The imperfection which solely comprises the whole of the second class is the rather unrealistic assumption of a constant surface relative humidity after $x = 0$. This assumption is able to generate physically unrealistic results, and it certainly is subject to improvement.

5 Fluxes derived from the initial profiles

5.1 INTRODUCTION

Apart from the bulk methods the flux-profile methods, used to determine the surface flux densities, require the (relatively) least effort and (relatively) cheapest instruments. This was discussed in Chapter 1. It is therefore not surprising that this method, or rather this class of methods, has been used extensively and that it is still the most widely adopted method in many disciplines. It is for this reason that literature on this subject has proliferated enormously (for reviews see e.g. Brutsaert, 1982, de Bruin, 1982).

In Chapter 2 it was demonstrated that this method may only be applied if the upstream terrain is homogeneous for a considerable distance. In practice, this condition often is difficult to satisfy, which means that the flux-profile methods are often used in non-homogeneous situations.

As this method has been, and still is, applied so often, it is justifiable to concentrate the main effort in this study on the behaviour of the flux-profile method in non-homogeneous situations.

However, before that analysis can be made it must be known if, in the homogeneous case, the fluxes determined with the various standard methods are consistent with each other. Hence the standard flux-profile methods will be studied when they are applied to the equilibrium distributions of U , θ and Q generated by Part I of the model. In section 5.2 a brief review of the standard flux-profile relations is presented. Using the modeled equations for the turbulent fluxes \overline{uw} , $\overline{w\theta}$ and \overline{wq} a set of so-called "second order flux-profile relations" is derived in section 5.3. This new set of relations is needed for the analysis of the flux profile relations above non-homogeneous terrains,

performed in Chapter 6. The second order flux-profile relations are identical with the standard flux-profile relations if some specific conditions are met. This will be discussed in section 5.4. In that section the results will be considered of the standard flux profile relations when these are applied to the equilibrium profiles.

5.2 SUMMARY AND COMPARISON OF THE FLUX-PROFILE RELATIONS

Through the application of dimensional analysis, invoking similarity, Obukhov (1946, 1971) has shown that the vertical fluxes of momentum, sensible heat and water vapour are related to the profiles of windspeed, temperature and humidity by:

$$I = - \rho k u_* z \frac{\partial U}{\partial z} / \phi_m(z/L) , \quad (5.1)$$

$$H = - \rho c_p k u_* z \frac{\partial \theta}{\partial z} / \phi_h(z/L) , \quad (5.2)$$

$$E = - \rho k u_* z \frac{\partial Q}{\partial z} / \phi_w(z/L) , \quad (5.3)$$

where ϕ_m , ϕ_h and ϕ_w are the dimensionless gradients of U , θ and Q respectively:

$$\phi_m(z/L) = \frac{\partial U}{\partial z} \cdot \frac{kz}{u_*} , \quad (5.4)$$

$$\phi_h(z/L) = \frac{\partial \theta}{\partial z} \cdot \frac{kz}{T_*} , \quad (5.5)$$

$$\phi_w(z/L) = \frac{\partial Q}{\partial z} \cdot \frac{kz}{q_*} . \quad (5.6)$$

These dimensionless gradients are universal functions of the stability parameter z/L , where the Obukhov length (L) is defined by:

$$L = - \frac{T_r}{gk} \cdot \frac{u_*^3}{(-u_* T_* - 0.61 T_r u_* q_*)} . \quad (5.7)$$

The respective fluxes after substituting (5.4)-(5.6) into

(5.1)-(5.3) read:

$$I = - \rho u_*^2, \quad (5.8)$$

$$H = - \rho c_p u_* T_* , \quad (5.9)$$

$$E = - \rho u_* q_* . \quad (5.10)$$

Equations (5.4)-(5.6) are referred to as the Monin-Obukhov similarity theory. These equations form the starting point of every flux-profile method applicable in the atmospheric surface layer. Equations (5.4)-(5.6) may only be employed when considering a horizontal, uniform surface under stationary conditions, and when we restrict ourselves to the atmospheric surface layer. As the functional form of the ϕ 's is not predicted by the Monin-Obukhov similarity theory they have to be determined experimentally. Any experiment performed to achieve this objective must necessarily fulfill the above mentioned conditions. Thus the best sets of data were gathered above extensive, flat, homogeneous, featureless terrains (Kansas, Minnesota, Wangara). Many functional relationships were proposed during the last two decades, they were reviewed by Dyer (1974), Yaglom (1977) and Visnawadham (1982). In the present study the relations proposed by Dyer (1974) will be used:

$$\phi_m = \phi_h^{\frac{1}{2}} = (1 - 16 z/L)^{-\frac{1}{4}}; \text{ for } z/L < 0 \text{ (unstable case)} \quad (5.11)$$

$$\phi_m = \phi_h = 1 + 5 z/L \quad ; \text{ for } 0 < z/L < 1 \text{ (stable case)} \quad (5.12)$$

$$\phi_w = \phi_h \quad ; \text{ for all } z/L \quad (5.13)$$

where $k = 0.41$.

If these relations are accepted, and $\partial U/\partial z$, $\partial \theta/\partial z$ and $\partial Q/\partial z$ are measured, Eqs. (5.4)-(5.6) constitute three coupled equations in three unknowns u_* , T_* and q_* . Usually Eqs. (5.4)-(5.6) are solved iteratively in their integrated form (Brutsaert, 1982;

de Bruin, 1982) but we will not elaborate on this point. The three gradients are determined by measuring U , θ and Q at at least two levels. Or either three at only one level, if respectively the surface roughness length, the surface temperature or the surface specific humidity is known. This standard flux profile method is also known as the Aerodynamic Method (see Table 5.1).

Table 5.1. Summary of input parameters for the application of the various standard flux profile methods.

Method	$R_n - G$	U	θ	Q	Remarks
1. Aerodynamic method	- -	+ + + z_0	+ + + +	+ + + +	Iteratively; coupling by L
2. Bowen ratio method	+	- (+) - (+)	+ +	+ +	If I wanted: add U measurements
3. Combination method	+ + + +	+ + + z_0 + + + z_0	+ + + + - - - -	- - - - + + + +	λE from energy balance (iteratively) H from energy balance (iteratively)

*N.B. ++ measurements at 1 (2) level(s) required.

If the fluxes of heat and water vapour are coupled by means of the energy balance equation, other possibilities arise. In its simplest form (ignoring the smallest terms) the energy balance equation reads:

$$R_n = H + \lambda E + G, \quad (5.14)$$

where λ is the latent heat of vaporization.

Eq. (5.14) states that the net flux density of incoming radiation energy (R_n) is distributed over the soil heat flux density (G), the sensible heat flux density (H) and the latent

heat flux density (λE), counting all fluxes, except R_n , directed away from the surface positive. The quantities R_n and G are readily measured, which means that Eqs. (5.4)-(5.6) and (5.14) constitute four equations in three unknowns. The three possibilities that are presented to us are the three combinations consisting of Eq. (5.14) and two out of three equations from (5.4)-(5.6).

If Eq. (5.4) is excluded from the set of four equations we arrive at the "Bowen Ratio Method" (see Table 5.1). As $\partial U / \partial z$ is not determined no windspeed measurements need to be made. If the Bowen ratio is defined by

$$\beta = \frac{H}{\lambda E}, \quad (5.15)$$

and Eqs. (5.2) and (5.3) are substituted, we obtain

$$\beta = \frac{c_p}{\lambda} \cdot \frac{\partial \theta / \partial z}{\partial Q / \partial z}, \quad (5.16)$$

where use has been made of Eq. (5.13).

Eq. (5.16) shows that the Bowen ratio can be determined by measuring temperature and humidity at two levels. The fluxes follow from

$$\lambda E = \frac{R_n - G}{1 + \beta}, \quad (5.17)$$

and

$$H = \beta \cdot \frac{R_n - G}{1 + \beta}. \quad (5.18)$$

The advantage of the Bowen ratio method is that it is independent of the functional form of ϕ_h and ϕ_w , as long as Eq. (5.13) applies. This method is also independent of the value of k , except for the determination of u_* , but it is not likely that u_* will be determined this way. Another advantage of this method is that it is not necessary to measure the exact height of the sen-

sors above the ground, as long as the two sets of sensors are mounted at the same heights. The disadvantage of the Bowen ratio method is that it does not apply when $R_n - G = 0$ (near sunrise and sunset) (Fuchs and Tanner, 1970; Sinclair et al., 1975; Revfeim and Jordan, 1976).

If either Eq. (5.5) or (5.6) is excluded from the set of equations (5.4)-(5.6) and (5.14), we refrain from temperature or humidity measurements respectively. As humidity measurements in the atmosphere are more difficult to perform than temperature measurements Eq. (5.6) usually will be the one that is excluded (see Table 5.1). This method, known as the "Combination Method", shares with the aerodynamic method the inherent uncertainties connected with the definition of the ϕ 's and the value of k . The advantage is that no humidity measurements are needed if Eq. (5.6) is excluded.

For comparison with the flux profile relations which will be derived from the modeled second order transport equations in 5.3 the flux densities are redefined in terms of eddy covariances:

$$I = \rho \overline{uw}, \quad (5.19)$$

$$H = \rho c_p \overline{w\theta}, \quad (5.20)$$

$$\lambda E = \rho \lambda \overline{wq}. \quad (5.21)$$

If Eqs. (5.1)-(5.3) are substituted into (5.19)-(5.21) respectively, we obtain:

$$\overline{uw} = -ku_*z \frac{\partial U}{\partial z} / \phi_m(z/L) = -K_m \frac{\partial U}{\partial z}, \quad (5.22)$$

$$\overline{w\theta} = -ku_*z \frac{\partial \theta}{\partial z} / \phi_h(z/L) = -K_h \frac{\partial \theta}{\partial z}, \quad (5.23)$$

$$\overline{wq} = -ku_*z \frac{\partial Q}{\partial z} / \phi_w(z/L) = -K_w \frac{\partial Q}{\partial z}. \quad (5.24)$$

The eddy diffusivity K of a property, introduced in these equations, is defined as the ratio of the property flux density to its concentration gradient, viz.

$$K_m = - I / (\rho \frac{\partial U}{\partial z}) = ku_* z / \phi_m(z/L) , \quad (5.25)$$

$$K_h = - H / (\rho c_p \frac{\partial \theta}{\partial z}) = ku_* z / \phi_h(z/L) , \quad (5.26)$$

$$K_w = - \lambda E / (\rho \lambda \frac{\partial Q}{\partial z}) = ku_* z / \phi_w(z/L) . \quad (5.27)$$

5.3 FLUX PROFILE RELATIONS DERIVED FROM THE MODELED TRANSPORT EQUATIONS

The modeled second order equations in the homogeneous case (A3.12)-(A3.23) provide us with another possibility of deriving flux profile relations. Rewriting the equations for the vertical turbulent fluxes (A3.15), (A3.17) and (A3.19) for \overline{uw} , $\overline{w\theta}$ and \overline{wq} respectively, we find after neglecting the turbulent transport terms

$$\overline{uw} = - \frac{\overline{w^2} \tau}{c_{13}} \left(\frac{\partial U}{\partial z} - \frac{g}{T_r} \frac{\overline{u\theta}}{\overline{w^2}} v \right) , \quad (5.28)$$

$$\overline{w\theta} = - \frac{\overline{w^2} \tau}{d_3} \left(\frac{\partial \theta}{\partial z} - \frac{g}{T_r} \cdot \frac{\overline{\theta\theta}}{\overline{w^2}} v \right) , \quad (5.29)$$

$$\overline{wq} = - \frac{\overline{w^2} \tau}{d_3} \left(\frac{\partial Q}{\partial z} - \frac{g}{T_r} \cdot \frac{\overline{q\theta}}{\overline{w^2}} v \right) , \quad (5.30)$$

where $c_{13} = d_3 = 13.1$

These relations were already suggested by Deardorff (1966) and were briefly discussed by Wyngaard (1982). If the turbulent transport terms in Eqs. (A3.15), (A3.17) and (A3.19) are neglected, the production (shear and buoyancy) of the turbulent fluxes is equal to the destruction. These counteracting processes do balance each other to a very good degree. This can be inferred

e.g. from Fig. 4.6 which displays the balance of the terms in the \overline{uw} -equation. The balance of terms in the $\overline{w\theta}$ and \overline{wq} equation, although not shown here, also indicates that the turbulent transport terms in those equations may be neglected.

5.4 COMPARISON IN HOMOGENEOUS SITUATIONS

Next, we could proceed with a direct comparison of the fluxes calculated with the standard flux-profile relations with the fluxes generated with the model. However, a better understanding will be obtained if a comparison is made of the standard flux profile relations and the "second order flux profile relations" (5.28)-(5.30). In 5.1 it was demonstrated that this is allowed in homogeneous situations.

5.4.1 Neutral stratification

In order to compare the standard flux-profile relations derived in 5.2 with Eqs. (5.28)-(5.30) the relatively simple case of a neutral atmosphere will be considered first. Equations (5.22) and (5.28) then reduce to respectively

$$\overline{uw} = -ku_*z \frac{\partial U}{\partial z}, \quad (5.31)$$

and

$$\overline{uw} = -\frac{\overline{w^2\tau}}{c_{13}} \cdot \frac{\partial U}{\partial z}. \quad (5.32)$$

These equations suggest that the factor $\overline{w^2\tau}/c_{13}$ may be identified with the eddy diffusivity $K_m = ku_*z$. This can also be demonstrated by substituting the lower- and upper boundary conditions for $\overline{w^2}$ and τ in $\overline{w^2\tau}/c_{13}$. We obtain then respectively:

$$\frac{\overline{w^2\tau}}{c_{13}} = ku_*z_{01} \quad \text{at } z = z_{01} \quad (\text{lower boundary}) \quad (5.33)$$

and

$$\frac{\overline{w^2 \tau}}{c_{13}} = ku_{*1} z_{\max} \left(1 - \frac{z_{\max} - z_{01}}{z_I}\right)^{-1} \quad \text{at } z=z_{\max} \text{ (upper boundary) (5.34)}$$

The factor $\left(1 - \frac{z_{\max} - z_{01}}{z_I}\right)^{-1}$ in equation (5.34) results from the imposed linear decrease of $-\overline{uw}/u_{*1}^2$ from unity near the surface (lower boundary) to zero at the PBL-height (z_I). The value of the factor $\left(1 - \frac{z_{\max} - z_{01}}{z_I}\right)$ ranges from 0.95 to 0.99 for typical values of z_I and z_{\max} for a well developed PBL. The small variation of \overline{uw}/u_{*1}^2 with height thus introduced, results in a gradually increasing deviation of $\overline{w^2 \tau}/c_{13}$ from $K_m = ku_{*1} z$ (Fig. 5.1). Similar remarks can be made on the water vapour flux density \overline{wq} and the gradient $\partial Q/\partial z$. In neutral conditions these yield essentially the same results regarding the eddy diffusivity of water vapour.

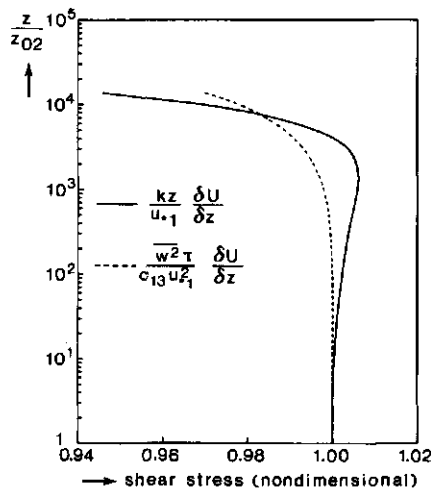


Figure 5.1 Nondimensional gradient of the windspeed in the neutral case ($\frac{z}{L} = 0$).

5.4.2 (Un)stable stratification

In the case of a diabatic atmosphere matters are more complicated. At first sight the term $\frac{g}{T_r} \overline{m\theta}_v / \overline{w^2}$ ($m = u, \theta$ or q) in Eqs. (5.28)-(5.30) seems to serve the same purpose as the inclusion of either ϕ_m , ϕ_h or ϕ_w in Eqs. (5.22)-(5.24) viz. a correction term for buoyancy. Why this apparently straightforward observation is not correct can be seen by inspecting the boundary conditions for the "eddy diffusivity" $\overline{w^2}\tau/c_{13}$. At the lower boundary minor changes occur in comparison with the neutral case, because buoyancy effects are of no importance there. Thus Eq. (5.28) e.g. again reduces to

$$\overline{uw} = - \frac{\overline{w^2}\tau}{c_{13}} \frac{\partial U}{\partial z},$$

which, after substituting the lower boundary conditions leads to (5.33). Analogous expressions apply for the $\overline{w\theta}$ - and \overline{wq} equation.

At the upper boundary, however, buoyancy does have a noticeable effect. If the upper boundary conditions (A3.28) are substituted into $\overline{w^2}\tau/c_{13}$ it is at once obvious that the value of $\overline{w^2}\tau/c_{13}$ at the upper boundary and hence, also the value of $\overline{w^2}\tau/c_{13}$ at lower heights is influenced by stability. This means that buoyancy effects enter Eqs. (5.28)-(5.30) in two ways:

- (i) through the generation of the term $\frac{g}{T_r} \overline{m\theta}_v / \overline{w^2}$ ($m = u, \theta$ or q) and
- (ii) through the upper boundary conditions of $\overline{w^2}\tau/c_{13}$.

The combination of these two effects should equal the buoyancy correction introduced with ϕ_m , ϕ_h and ϕ_w in Eqs. (5.1)-(5.3). To investigate this assumption two possibilities can be explored. The first possibility is to transform Eqs. (5.28)-(5.30), re-writing terms containing $\frac{g}{T_r} \overline{m\theta}_v / \overline{w^2}$ as far as possible towards Eqs. (5.22)-(5.24). In doing so, we are able to show that the two sets of equations are very similar indeed. The second possibility lies in using both methods on the profiles of U , θ and Q generated with Part I of the model (homogeneous case).

The first possibility is explored in Appendix 5. The problem encountered there is the fact that the variables $\overline{w^2}$ and τ ($\tau = \overline{u_1 u_1} / \epsilon$) cannot be rewritten in terms of the first moments because calculation of the balance of the $\overline{u^2}$, $\overline{v^2}$ and $\overline{w^2}$ equations (A3.12)-(A3.14) shows that the turbulent transport terms occurring in those equations cannot be neglected. This deters us from obtaining a simple relation for $\overline{e^2}$ (U, θ).

Figures (5.2) and (5.3) show the results of the calculations of the second possibility. Figures (5.2) and (5.3) represent the distribution of the momentum flux and the sensible heat flux, respectively. The latent heat flux curves are not presented as they coincide with the sensible heat flux curves if the horizontal axis is properly scaled.

In Fig. 5.2 the three curves represent the momentum flux obtained from: the aerodynamic method (curve 1), equation (5.28) (curve 2) and the model (curve 3). Comparison of curves 2 and 3

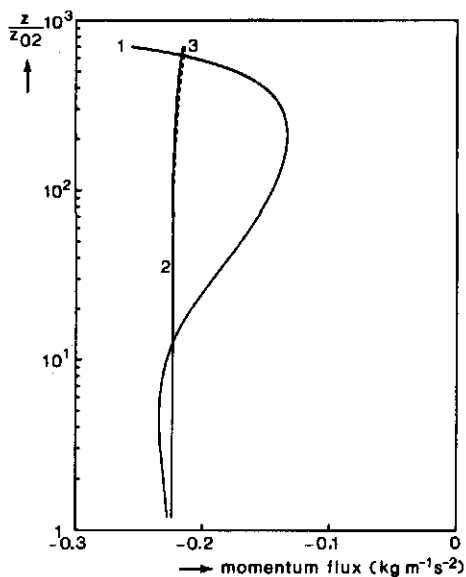


Figure 5.2 Profiles of the momentum flux calculated with: the aerodynamic method (curve 1), Eq. (5.28) (curve 2) and the present model (curve 3).

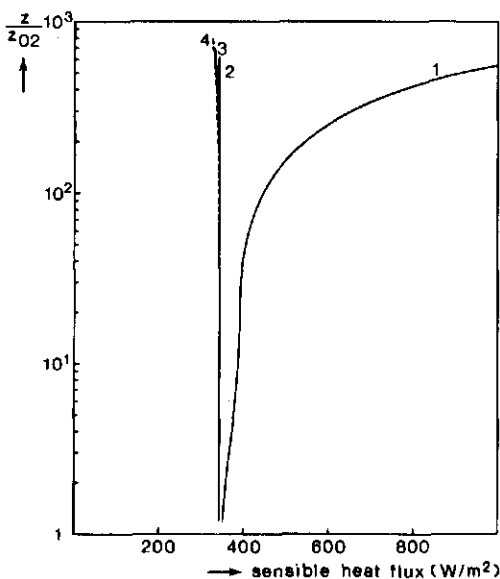


Figure 5.3 Profiles of the sensible heat flux calculated with: the aerodynamic method (curve 1), the Bowen ratio method (curve 2), Eq. (5.29) (curve 3) and the present model (curve 4).

learns, once again, that the omission of the turbulent transport term in Eq. (5.28) has no significant effect on the flux calculation. The deviation of curve 1 from the other two curves is of more concern. This will be discussed further on.

The remarks made above, can be made on the curves in Fig. 5.3. Here the four curves represent the sensible heat flux obtained from: the aerodynamic method (curve 1), the Bowen ratio method (curve 2), Eq. (5.29) (curve 3) and the model (curve 4). For the sensible (and latent) heat flux the deviation of curve 1 from the other curves is caused (just as in Fig. 5.2) by a deficiency of the model. The problem in this case is the fact that Part I of the model is not capable to produce profiles of the gradients of U , θ and Q that obey Eqs. (5.22)-(5.24) exactly. This deficiency of the model stems from the fact that the value of c_{13} and d_3 in Eqs. (A3.15), (A3.17) and (A3.19) for \overline{uw} , $\overline{w\theta}$ and \overline{wq} respectively, no longer suffices to ensure equality of both sides of these equations when the upper boundary conditions are substituted. This situation is notably prominent in very

(un) stable conditions when the buoyancy terms, which do not influence the lower boundary conditions, enter the upper boundary conditions. Calculations show that in the \overline{uw} -equation (A3.15) the discrepancy between both sides of the equation may be as large as 10%, while for the $\overline{w\theta}$ and \overline{wq} equations the discrepancy is about 5%. This discrepancy at the upper boundary forces the gradient $\partial U/\partial z$ to deviate from the expected $\phi_m(z/L)$ distribution, and likewise for the $\partial\theta/\partial z$ and $\partial Q/\partial z$ distributions. This phenomenon is also visible in Fig. (4.2).

There is one other major assumption, though, in one of the standard flux-profile techniques which bypasses the model's deficiency mentioned above. This assumption, made in the Bowen ratio method is

$$K_h = K_w . \quad (5.35)$$

Using the definition of the various K's (5.25)-(5.27) this assumption reduces to Eq. (5.13):

$$\phi_w = \phi_h \quad ; \quad \text{for all } z/L.$$

To verify the validity of this assumption in the present model we consider the correlation coefficient

$$r_{\theta q} = \frac{\overline{q\theta}}{(\overline{q^2})^{\frac{1}{2}} (\overline{\theta^2})^{\frac{1}{2}}} \quad (5.36)$$

It appears that in the equilibrium profiles this coefficient is exactly equal to 1 everywhere. This implies that

$$\theta = -cQ + d , \quad (5.37)$$

which gives

$$\theta = cq \quad \text{and} \quad \frac{\partial \theta}{\partial x_i} = -c \frac{\partial Q}{\partial x_i} , \quad (5.38)$$

where c and d are constants. If (5.37), (5.38) and the definition of the K 's (5.26)-(5.27) are used the condition $K_h = K_w$ is fulfilled exactly.

An alternative is to define a Bowen ratio (β^*) in terms of the second order model flux profile relations (5.28)-(5.30)

$$\beta^* = \rho c_p \overline{w\theta} / \rho \lambda \overline{wq} . \quad (5.39)$$

Substitution from (5.28)-(5.30) yields

$$\beta^* = \frac{c_p}{\lambda} \left(\frac{\partial \theta}{\partial z} - \frac{g}{T_r} \frac{\overline{\theta\theta}}{w^2} v \right) / \left(\frac{\partial Q}{\partial z} - \frac{g}{T_r} \frac{\overline{q\theta}}{w^2} v \right) . \quad (5.40)$$

This expression is compared with the standard Bowen ratio formula

$$\beta = \frac{c_p}{\lambda} \frac{\partial \theta / \partial z}{\partial Q / \partial z} .$$

If the condition

$$\frac{\partial \theta / \partial z}{\partial Q / \partial z} = \frac{\overline{\theta\theta} v}{\overline{q\theta} v} , \quad (5.41)$$

is fulfilled, the expressions for β^* and β are identical. Note that in Eq. (5.40) we still assumed that the turbulent transport terms in Eqs. (5.28)-(5.30) can be ignored. But this is not a necessary assumption for Eq. (5.38) to apply. When the correlation coefficient $r_{\theta q}$ is considered again and (5.37) and (5.38) are used in Eq. (5.41) this condition is fulfilled exactly. Even if the turbulent transport terms in Eqs. (5.28)-(5.30) and hence also in Eq. (5.40) are retained we would still find that β^* and β are identical. Thus if the equilibrium distributions as generated by Part I of the model are substituted in (5.41) an equivalence of both sides is obtained. As a result of this equivalence, the fluxes of latent and sensible heat calculated with the standard Bowen ratio method (5.16) are almost identical (within 5%) with the respective turbulent heat fluxes as generated by the model (see Fig. 5.3). The small

difference of 5% follows, again, from the linear decrease of $\overline{w\theta}$ and \overline{wq} with height to zero at the PBL height. As β is calculated by the ratio of these fluxes, the Bowen ratio method is not sensitive to this linear decrease and consequently yields heat fluxes which are essentially constant with height.

Finally, Figs. 5.2, 5.3 and 5.4 also contain the fluxes calculated with Eqs. (5.28)-(5.30). These fluxes do not deviate significantly from the fluxes generated by the model, by virtue of the good approximation of neglecting the very small turbulent transport terms.

5.5 CONCLUSIONS

In this chapter flux-profile relations were derived from (i) the "standard" or K-theory approach and (ii) the modeled transport equations of the turbulent fluxes in homogeneous situations. For both methods a number of approximations must be made. For the first class of relations (K-theory) these approximations are: (i) the profiles of U , θ and Q above homogeneous terrain exhibit the well-known Monin-Obukhov similarity behaviour expressed by Eqs. (5.4)-(5.6) (the Aerodynamic and Combination methods) (ii) $K_h = K_w$ or rather $\phi_h = \phi_w$ (the Bowen ratio method). For the second class of relations (2nd order model) there is only one major approximation viz. (iii) the turbulent transport terms in the second moment equations may be neglected.

Of course there are other assumptions which are equally important e.g. the exact modeling expression for the pressure terms in the equations of the second moments, the assumption that $k = 0.41$ etc. The discussion of these assumptions lies beyond the scope of the present study.

From the comparison of the two classes of methods, performed in section 5.4, the following conclusions can be made. (i) The present model is not capable of reproducing the distribution

of the dimensionless gradients ϕ_m , ϕ_h and ϕ_w exactly. This is attributed to the discrepancy of the model equations where the upper boundary conditions are substituted. As a consequence, the results of the application of the aerodynamic method on the equilibrium profiles of U , θ and Q show a large deviation from the fluxes calculated with the model. This deviation increases with $|z/L|$. (ii) An analytical treatment of the flux-profile relations from the second order transport equations (Appendix 5) in order to arrive at the standard flux-profile relations is only partly successful as the factor $\bar{w}^2 \tau / c_{13}$ in Eqs. (5.28) - (5.30) cannot be rewritten in terms of the first moments (U , θ and Q). (iii) The assumption $K_h = K_w$ is valid, by virtue of $r_{\theta q}$, defined in (5.36), equals 1. Thus in the equilibrium profiles the application of the Bowen ratio method is allowed. It produces heat fluxes which are consistent with the heat fluxes generated by the model. (iv) The application of the Bowen ratio technique on the turbulent fluxes $\bar{w}\theta$ and $\bar{w}q$ from Eqs. (5.29) and (5.30) leads to an excellent agreement (within 5%) with the heat fluxes calculated with the "standard" Bowen ratio method. There is also a good agreement with the turbulent heat fluxes as calculated with the model.

Regarding (i) one must realize that in the example given in section 5.4 the instability was very strong ($L = -22$ m). This, rather extreme, value of the Obukhov length was chosen deliberately to bring out the deviations of the two classes of methods more clearly. For moderate to weak unstable situations these deviations are smaller. The disadvantage of this choice of the Obukhov length is that it is questionable if the present and forthcoming analysis may be extended far beyond $|z/L| = 1$. Keeping this remark in mind, we may conclude that the analysis of the standard flux-profile methods in inhomogeneous situations with the aid of the second order model equations, which will be made in the next chapter, is very good feasible.

6 Flux-profile methods above inhomogeneous terrain

6.1 INTRODUCTION

From the moment the air flows over a different surface a relaxation towards a new equilibrium will occur (Ch. 2). Obviously, the indiscriminate use of standard flux-profile methods in these circumstances may lead to results which are unrealistic or which contain large errors. Confining the measurement heights to the IAL will probably exclude these problems as good as possible. However, it cannot be taken for granted that every flux-profile method will yield correct results there. In this chapter an analysis is made of the errors in the standard flux-profile methods when they are applied within the IAL.

In this chapter only one single change in surface conditions will be considered. Blom and Wartena (1969) showed that in the case of multiple changes in surface conditions a new IBL, and also a new IAL, will start to develop after every discontinuity of the surface. Thus the influence of former transitions in surface conditions will primarily influence the ASL above the newly developing IAL. This observation can also be inferred from the analysis of the double change in surface conditions in Ch. 4.

This chapter will mainly concentrate on two standard flux-profile methods: the Bowen ratio method (section 6.2) and the Aerodynamic method (section 6.3). The remaining combination method (see 5.2) merely combines, as the name indicates, some of the assumptions of both methods and no new viewpoints are expected from its analysis. From the two methods just mentioned, the Bowen ratio method will get the major part of the attention, for it has proven to be very consistent with both the second order flux-profile relations and with the fluxes as generated with the model in the homogeneous case (Ch. 5). The deviation of the results of the Aerodynamic method from the equilibrium model fluxes, primarily for large $|z/L|$ values, complicates the

application of this method in non-homogeneous conditions.

The fluxes of both standard methods after the change in surface conditions ($x > 0$) will be compared with the fluxes calculated with the model. The deviations encountered in this comparison will be analyzed by considering the distribution of the terms in the turbulent flux equations as well as some other important parameters of the ASL.

6.2 THE BOWEN RATIO METHOD

In the case which will be considered here, the air flows from a smooth, hot and dry terrain to a rougher, cooler humid surface. The upstream terrain is characterized by: $z_{01} = 0.02$ m, $\theta_{01} = 44.5^\circ\text{C}$, $Q_{01} = 4.6 \cdot 10^{-3}$, $H_{01} = 336$ W/m² and $\lambda E_{01} = 19$ W/m², resulting in $L = -23$ m. The parameters solely characterising the downstream surface are: $z_{02} = 0.06$ m and $R_{02} = 100\%$, all other parameters are a function of the downstream distance (x).

From the situation described above, the latent heat flux is expected to increase substantially just after the surface change, while the sensible heat flux must decrease accordingly. The dashed lines in Figs. 6.1 and 6.2 for the sensible and latent heat flux respectively, represent the profiles of these quantities for various distances downstream of the surface change. These fluxes, calculated with the model, will be indicated by λE_{MD} and H_{MD} . As the air progresses over the humid surface its absolute humidity increases and the vapour pressure deficit and the latent heat flux near the surface decrease. For the sensible heat flux the opposite occurs. The drawn lines in Figs. 6.1 and 6.2 represent the profiles of the sensible and latent heat flux calculated with the Bowen ratio method, using the local profiles of θ and Q . These fluxes will be denoted λE_{BR} and H_{BR} .

Several characteristic features can now be observed. For $z > d$ the profiles of θ and Q must, by definition, be unaffected by

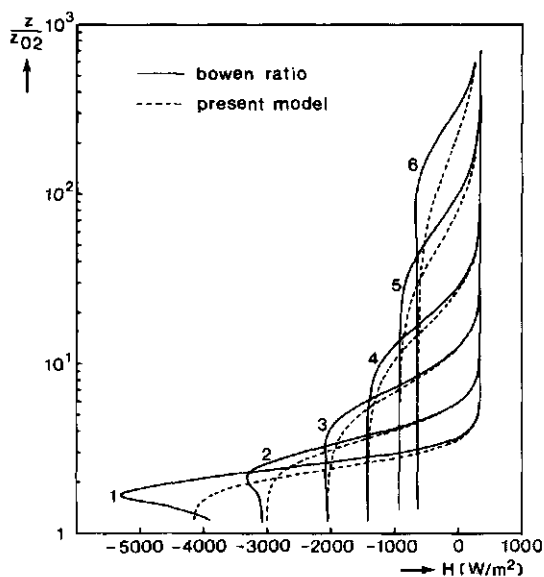


Figure 6.1

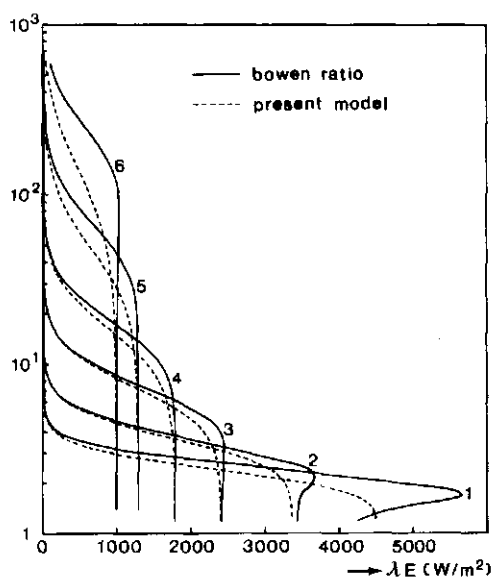


Figure 6.2

Figure 6.1 Profiles of the sensible heat flux density after a change in surface conditions for various distances downstream ($z_{02} = 0.06$ m).

Curve	x/z_{02}	x (m)
1	1.7	0.1
2	8.5	0.5
3	$4.3 \cdot 10^1$	2.5
4	$1.7 \cdot 10^2$	10.0
5	$8.5 \cdot 10^2$	50.0
6	$3.4 \cdot 10^3$	200.0

Figure 6.2 Profiles of the latent heat flux density after a change in surface conditions for various distances downstream (curve numbers correspond to those of Fig. 6.1).

the new surface. In this region the fluxes determined with the Bowen ratio method are equal to the respective fluxes calculated with the model. Only some small deviations, discussed in chapter 5, can be observed. For $z \ll \delta$ the same observation can be made. In this region the ASL is in local equilibrium with the new surface and hence, the Bowen ratio method yields fluxes that deviate less than a few percent from the fluxes calculated with the model. Between these two layers the discrepancies are confined.

Because the heat fluxes attain extremely high values just after the surface change, Figs. 6.1 and 6.2 tend to obscure the relative differences between the fluxes of the two methods. Fig. 6.3 therefore presents the ratio of the latent heat fluxes of the model and the Bowen ratio method ($\lambda E_{BR}/\lambda E_{MD}$). For the sensible heat flux this ratio can attain unrealistic values, because both H_{BR} and H_{MD} change sign at different heights. The ratio H_{BR}/H_{MD}

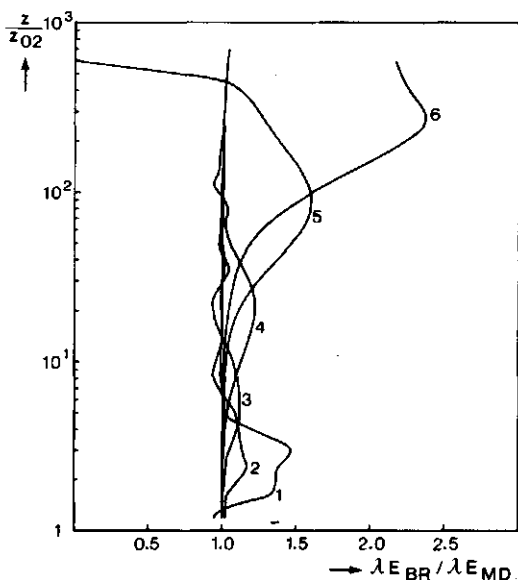


Figure 6.3 Ratio of the latent heat flux calculated with the Bowen ratio method (λE_{BR}) and the latent heat flux generated with the model (λE_{MD}) for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

will not be considered. From Fig. 6.3 it can be seen that the Bowen ratio method in the present case yields latent heat fluxes within the IAL which are larger than the latent heat fluxes calculated with the model. The error, however, nowhere exceeds 50%. For distances just after the surface change (curves 1 - 3) this error decreases with downstream distance, as expected. However, the error increases again further downstream (curves 4 - 6). This increase of the error can be explained as follows. As the IBL height increases with downstream distance, the profiles of θ and Q are affected at increasing heights. At these heights the gradients of θ and Q are very small and even a minute perturbation will cause a relatively large error in the flux-profile

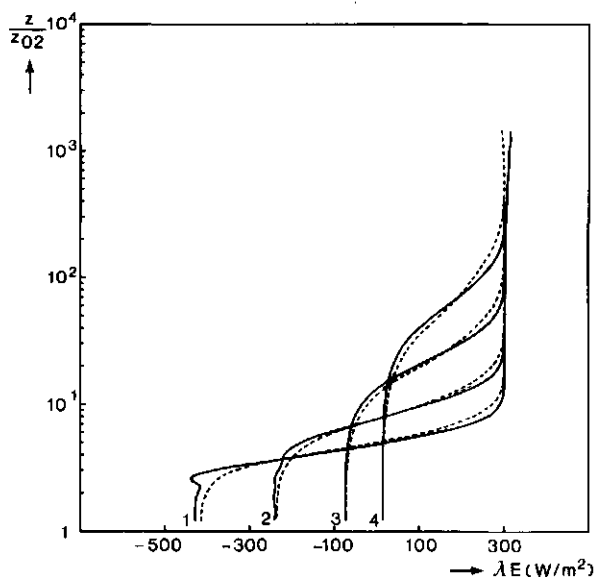


Figure 6.4 Profiles of the latent heat flux density after a change in surface conditions for various distances downstream.

Curve	x (m)
1	0.25
2	1.0
3	5.0
4	20.0

method. In the last two curves (5 and 6), where $d > z_{\max}$ (upper grid level), the upper boundary conditions contaminate the solution in the upper IBL. We disregard this region from now on.

When a change takes place from a cool, humid terrain to a warm and dry surface the Bowen ratio method also yields heat fluxes which are larger than the heat fluxes produced by the model. This can be observed in Fig. 6.4 for λE .

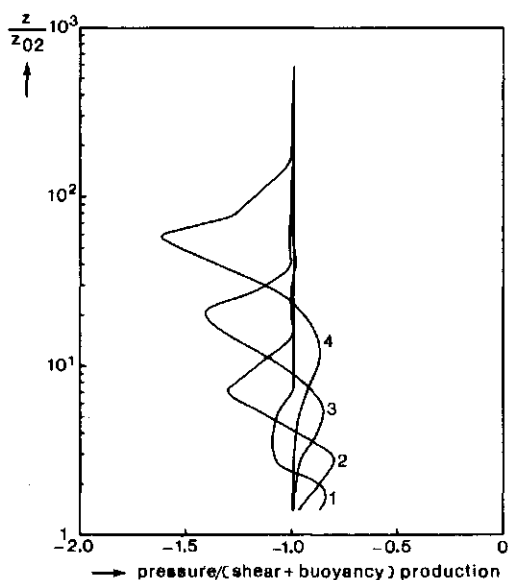
In order to analyse the causes of the errors in the Bowen ratio method, the equation for the second moments must be considered. The reason for this is that we demonstrated in the previous chapter that the method in which the Bowen ratio (β^*) was defined with the mean turbulent heat fluxes ($\overline{w\theta}$ and \overline{wq}) from the model equations (5.40) yields results which are identical with the results of the standard Bowen ratio method (5.16). This agreement occurred if one condition was fulfilled viz. $r_{\theta q} = 1$, which also implied the condition (5.41). Application of the complete model equations for \overline{wq} and $\overline{w\theta}$ in $\beta^* = \frac{c_p}{\lambda} \frac{\overline{w\theta}}{\overline{wq}}$ will naturally yield latent and sensible heat fluxes which are exactly equal to λE_{MD} and H_{MD} . After the change in surface conditions, however, two effects will cause the inequality of β and β^* . First, $r_{\theta q}$ no longer equals 1. And, second, the equations for $\overline{w\theta}$ and \overline{wq} contain two new terms for the inhomogeneous part of the flow. The second effect will be considered first.

The modeled \overline{wq} equation for $x > 0$ reads:

$$\underbrace{U \frac{\partial \overline{wq}}{\partial x}}_I + \underbrace{W \frac{\partial \overline{wq}}{\partial z}}_{II} + \underbrace{\overline{w^2} \frac{\partial Q}{\partial z} - \frac{g}{T} \overline{q\theta} \overline{v}}_{III} - a_t \frac{\partial}{\partial z} (\overline{w^2 \tau} \frac{\partial \overline{wq}}{\partial z}) = - d \frac{\overline{wq}}{\tau} \quad (6.1)$$

A similar equation holds for $\overline{w\theta}$. Two advection terms (I and II) are added in the inhomogeneous situation. Note that after the surface change the turbulent transport term (III) no longer can be neglected forthwith. An assessment of the relative importance of the advection and turbulent transport terms together can be obtained by considering the ratio of the pressure term and the

sum of the production terms (shear and buoyancy). The distribution of this ratio is given in Fig. 6.5a.

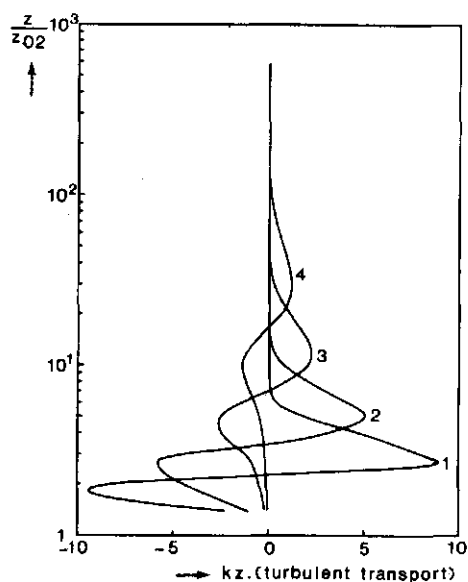


[2]

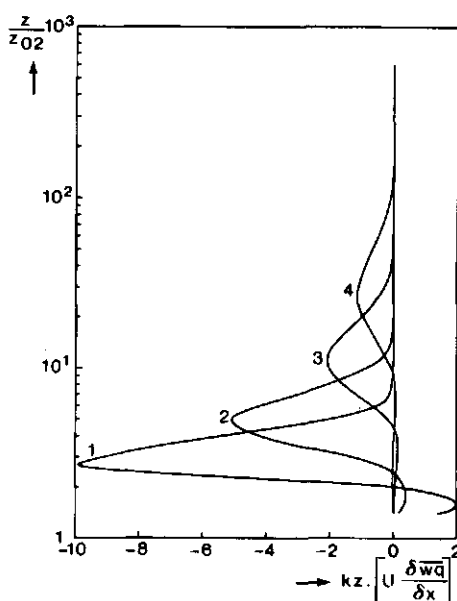
Figure 6.5a Ratio of the pressure term of the \overline{wq} equation (Eq. 6.1) and the sum of the buoyancy- and shear production terms of the same equation, for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

Fig. 6.5a shows that just downstream of $x = 0$ there is no region adjacent to the surface where the production terms and the pressure covariance term balance each other (curves 1,2). Approximately from curve 3 onwards ($x = 2.5$ m.) such a region does exist. Note that as the IBL increases in height downstream, the deviation of the ratio from the equilibrium value increases in the upper part of the IBL. The reason, again, is the relatively large sensitivity of the ratio for small perturbations in a region where every term of the equation has become very small.

Calculation of the terms I, II and III learns that the vertical advection term $W \partial \overline{wq} / \partial z$ (II) is much smaller than either of the other two terms. It amounts to approximately 15% of either term



[b]



[c]

Figure 6.5b Profiles of the turbulent transport term of the \overline{wq} -equation (eq. 6.1) for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

Figure 6.5c Profiles of the horizontal advection term of the \overline{wq} -equation (eq. 6.1) for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

I or III. This does not imply that this vertical advection term is always insignificant. Figs. 6.5b and 6.5c show that the turbulent transport term (III) and the horizontal advection term (I) nearly cancel each other. Hence the vertical advection term (II), in which the roughness change information is contained, may be very important after all.

The effect of the "inhomogeneity term" i.e. the sum of the advection terms and the turbulent transport term in the \overline{wq} (and $\overline{w\theta}$) equations on the latent heat flux is presented in Figs. 6.6 and 6.7. In Figure 6.6 the dashed curves represent the model

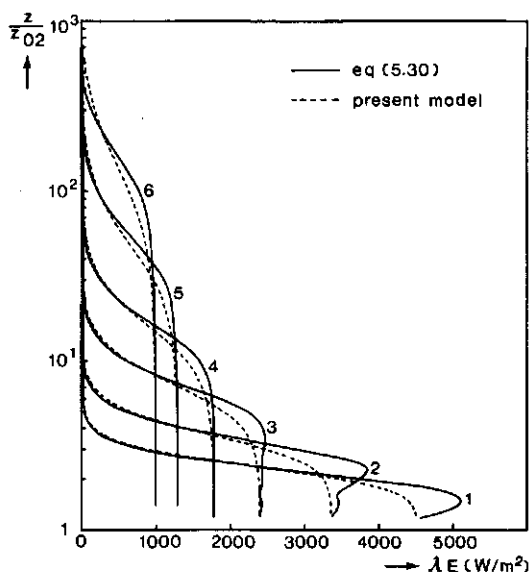


Figure 6.6 Profiles of the latent heat flux density after a change in surface conditions, for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

flux λE_{MD} , i.e. application of the full equation (6.1). The drawn curves in Fig. 6.6 represent the latent heat fluxes obtained by applying eq. (6.1) without the inhomogeneity term (I + II + III). This flux will be denoted by λE_{HM} . The ratio $\lambda E_{HM}/\lambda E_{MD}$ is given in Fig. 6.7.

If Figure 6.6 is compared with Fig. 6.2 it can be seen that the errors in λE_{BR} for the greater part can be attributed to the effects of the inhomogeneity term in the \overline{wq} (and $\overline{w\theta}$) model equations. This observation only applies, however, within the IAL. Outside this layer, but well inside the IBL, the neglect of the "inhomogeneity term" causes the latent heat flux λE_{HM} to

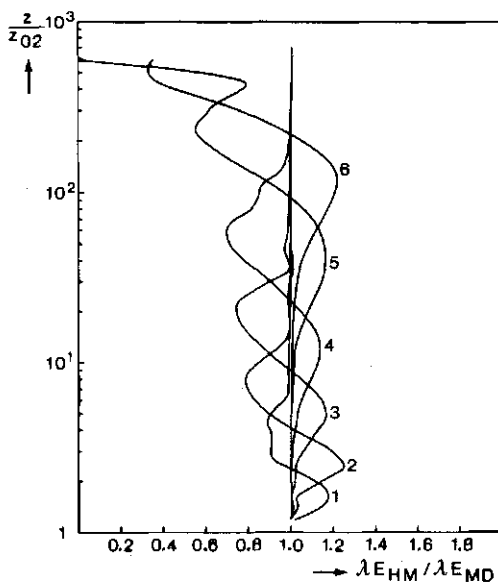


Figure 6.7 Ratio of the latent heat flux calculated with eq. (5.30) (λE_{HM}) and the latent heat flux generated with the model (λE_{MD}) for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

be smaller than λE_{MD} . This underestimation does not occur when the standard Bowen ratio method is applied. As Fig. 6.5a indicates, the contribution of the inhomogeneity term to the \overline{wq} -equation in this part of the flow is reversed, hence it must be expected that $\lambda E_{HM} < \lambda E_{MD}$. Therefore we conclude that there is another effect which counteracts the effects of the reversed contribution of the inhomogeneity term to eq. (6.1).

The agreement within the IAL of λE_{BR} and λE_{HM} is demonstrated in Fig. 6.8. This figure presents the ratio $\lambda E_{BR}/\lambda E_{HM}$. Calculations show that within the IAL both fluxes agree within 3%. From this figure it is immediately obvious, once more, that outside the IAL the two methods strongly disagree.

Although the region above the IAL is of no direct concern for the present study, it is still interesting to discuss the cause of the above mentioned deviation. In chapter 5 it was shown that

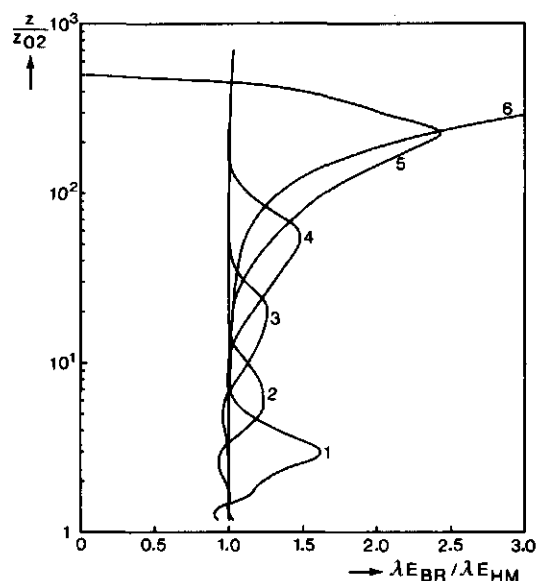


Figure 6.8 Ratio of the latent heat flux calculated with the Bowen ratio method (λE_{BR}) and the latent heat flux calculated with eq. (5.30) (λE_{HM}) for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

$\lambda E_{BR} = \lambda E_{HM}$ and $H_{BR} = H_{HM}$ if condition (5.41) is fulfilled. In Fig. 6.9 the ratio $[(\partial\theta/\partial z)/(\partial Q/\partial z)]/(\overline{\theta\theta}_v/\overline{q\theta}_v)$ is given for the same downstream distances as the curves in Fig. 6.8. Comparison of Figs. 6.8 and 6.9 learns that, especially for larger distances downstream (curves 3 and 4), the heights where $\lambda E_{BR}/\lambda E_{HM}$ shows a maximum coincide with the heights where the ratio of Fig. 6.9 has a minimum. This coincidence of heights is not present for small distances (curves 1 and 2). Hence, if distances just downstream of the surface change ($x < 1.m$) are disregarded, the disagreement of λE_{BR} and λE_{HM} above the IAL can be attributed to the differences in the transport of temperature and humidity of which the ratio $[(\partial\theta/\partial z)/(\partial Q/\partial z)]/(\overline{\theta\theta}_v/\overline{q\theta}_v)$ is a measure.

Finally, one is usually interested in the value of the fluxes at the surface (or at $z = z_0$). The Bowen ratio method inside the IAL yields results which agree better with the surface flux than

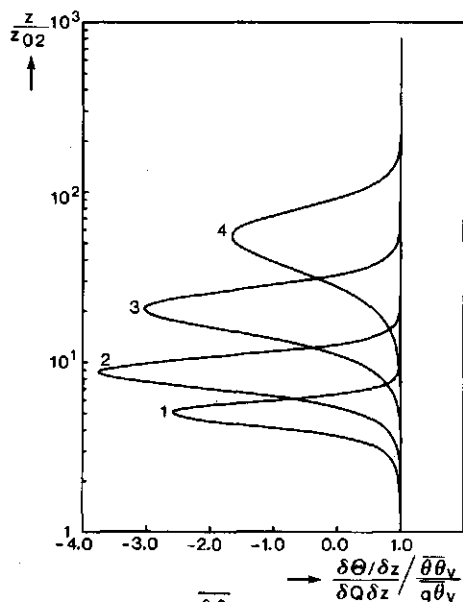


Figure 6.9 Ratio $\left(\frac{\partial \theta / \partial z}{\partial Q / \partial z}\right) / \frac{\bar{\theta} \bar{v}}{\bar{q} \bar{\theta}_v}$ for various distances downstream

(Curve numbers correspond to those of Fig. 6.1).

the model flux. This can be inferred from Figs. 6.1 and 6.2. Thus as far as the surface fluxes are concerned, the Bowen ratio method may even be applied above the IAL. As the accuracy of present flux estimation techniques is approximately 10% (chapter 5) a height (h_{BR}) can be defined below which the deviation of λE_{BR} and H_{BR} is less than 10% of the respective surface fluxes. The ratio h_{BR}/δ is approximately constant with downstream distance and amounts to 1.5.

Summarizing, in this section it was found that the Bowen ratio method, applied above a wet, cool surface downstream of a dry, hot terrain yields sensible- and latent heat fluxes in the IAL of which the absolute values are larger than the respective heat fluxes calculated with the model. If Bowen ratio measurements are made within the IAL, the deviation of H_{BR} and λE_{BR} from the fluxes calculated with the model (H_{MD} and λE_{MD}) does not exceed 10%, except for $x < 1.0$ m. This deviation within the IAL can mainly be attributed to the neglect of the advection terms and the turbulent transport term in the \overline{wq} and $\overline{w\theta}$ equations. Outside

the IAL this neglect would result in latent heat fluxes which are up to 40% smaller than the model fluxes. In that part of the ASL, however, the differences in transport of water vapour and heat compensate for this underestimation.

6.3 THE AERODYNAMIC METHOD

Referring to the remarks made in sections 5.5 and 6.1 about the applicability of the Aerodynamic method, a quantitative discussion of this method cannot be made. Hence only a short qualitative review of the results of this method above non-homogeneous terrain is presented.

First of all, note that the Monin-Obukhov similarity theory formally cannot be used for $z < 10 h_0$ (Tennekes, 1973). Here h_0 stands for the average height of the roughness elements and is related to z_0 by $z_0 \approx 0.14 h_0$. However, in the present model the calculations which use the Monin-Obukhov theory are performed down to $z = z_0$. We commented on this issue in section 4.1.3. For analytical purposes, this violation of the Monin-Obukhov theory has no serious consequences. Hence the discussion in this section may be performed. For practical purposes, however, only the region $z/z_{02} > 100$ must be considered. In the present situation $z = 100 z_{02}$ just about coincides with the height of the region above which the fluxes resulting from the aerodynamic method applied to the equilibrium profiles start to deviate considerably from the model fluxes (see Fig. 5.2). The qualitative results of the following, however, may be transposed to situations where those regions do not overlap.

Figure 6.10 shows the latent heat flux from the application of the aerodynamic method to the same situation as in the preceding section. Fig. 6.10 may be compared with Fig. 6.2 (note the difference in the scaling of the horizontal axes). From this comparison it is clear that the latent heat fluxes which result from the application of the aerodynamic method (λE_{AE}) show a

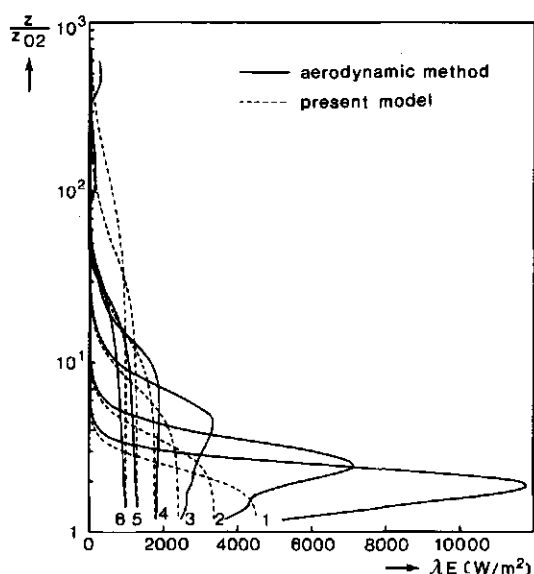


Figure 6.10 Profiles of the latent heat flux density after a change in surface conditions for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

much larger deviation from the model fluxes λE_{MD} than the fluxes λE_{BR} . Apparently the aerodynamic method has a larger sensibility for surface inhomogeneities than the Bowen ratio method. This stems from the more stringent conditions which the aerodynamic method has to satisfy (see section 5.2). Fig. 6.11 shows the ratio $\lambda E_{AE}/\lambda E_{MD}$. The upper part of this figure ($z/z_{02} > 30$) for curves 1 - 4 complies with Fig. 5.3: It shows the incapability of the present model to reproduce the theoretical $\phi_m(z/L)$ curves exactly. The lower part of this figure shows that the aerodynamic method yields latent heat fluxes within the IAL, which are larger than the model fluxes λE_{MD} . The magnitude of the sensible heat flux is also overestimated by the aerodynamic method.

Comparison of Figs. 6.11 and 6.3 learns that the deviations of λE_{AE} from λE_{MD} are roughly 4 times as large as the deviations of λE_{BR} from λE_{MD} . It also shows that the aerodynamic method yields latent heat fluxes which are considerably smaller than

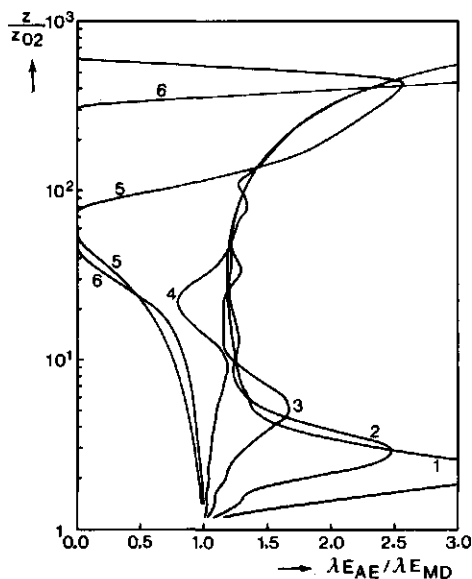


Figure 6.11 Ratio of the latent heat flux calculated with the aerodynamic method (λE_{AE}) and the latent heat flux generated with the model (λE_{MD}) for various distances downstream. (Curve numbers correspond to those of Fig. 6.1).

λE_{MD} for larger distances downstream (curves 4 and 5) in the upper part of the IBL. This is partly attributed to the deterioration of the model results caused by the upper boundary conditions discussed hereafter.

For very great distances downstream ($x/z_{02} \approx 2000$) the fluxes from the aerodynamic method within the IAL, however, do not converge towards the fluxes H_{MD} and λE_{MD} . This behaviour, which was not present in the Bowen ratio results, is attributed to a gradual deterioration of the quality of the model results as the calculation is performed beyond $x = x_{max}$. At this distance downstream the IBL height d equals $z = z_{max}$ (the upper calculation level). Especially the shear stress profiles are affected by the errors which result from the application of the upper boundary conditions for $x > x_{max}$. These errors gradually diffuse downwards. The Bowen ratio method, by nature, is less susceptible to perturbations in the shear stress profile. But, indi-

rectly, through the coupling of all the differential equations, also the Bowen ratio method is slightly in error in the upper part of the IBL (the last curve in Figs. 6.1 and 6.2). This is of no concern as yet.

7 Summary and conclusions

In Chapter 1 the goals of the present study were presented.

These goals are

- (i) the estimation and analysis of the errors introduced in the standard flux determination methods when they are applied above non-homogeneous terrain
- (ii) providing simple techniques for estimating these errors, using a minimum number of data concerning sensor location, surrounding terrain(s) etc.

These goals suggested that a direct treatment of the flux-profile methods above a non-homogeneous terrain, with the aid of a second order model, was feasible. In the course of this study, however, it turned out that a direct treatment was not immediately possible. First of all, the performance of the model had to be analyzed in order to tackle the posed problem. This analysis was needed for the correct interpretation and appreciation of the results of the flux-profile methods above non-homogeneous terrain. Hence, the conclusions in this final chapter can be separated in two groups: (i) those which concern the model used in the present study, and (ii) those which concern the application of the model in order to study the flux-profile methods.

In Chapter 2 the first simplifications were introduced. These simplifications referred to the initial conditions as well as the upper boundary conditions of the numerical model. E.g. the present model is restricted to the atmospheric surface layer, which in its initial state is supposed to be in equilibrium with the surface. Also, the upper boundary value of every variable (except W) remains fixed after the change in surface conditions. This led to the conclusion that the model is only applicable to a limited downstream distance (x_{\max}). As soon as the flow perturbations, induced by the new surface, reach the highest grid level (z_{\max}) the fixed upper boundary values start to contaminate the solution. The deterioration of the solution subsequent-

ly diffuses downwards as the downstream distance increases beyond x_{\max} . To avoid this problem the model should be extended to comprise the whole of the atmospheric boundary layer. This has already been accomplished (e.g. Wyngaard, 1975).

In Chapter 3 we concluded that second order models are superior to first order models, mainly when the second moments (fluxes and other (co)variances) are considered. However, the disadvantage of second order models is the difficulty related to the modeling of the third order terms and the pressure terms. Sometimes, mean strain and buoyancy terms have to be incorporated in the approximation of the 3rd order and pressure terms, in order to get physically realistic results. At present, there is no general agreement on the modeling of these terms. Usually the "engineering approach" is used, where model constants are tuned to obtain the desired results. The third order models which have recently appeared, merely shift the problem to the modeling of the fourth order terms. Obviously, much has still to be learned here.

The above conclusions hold for any 2nd order model. The present model was examined more closely in Chapter 4. It was found that flaws in the upper boundary conditions may be responsible for the deviation of the dimensionless gradients of U , θ and Q from the expected universal ϕ -curves in diabatic equilibrium conditions. Efforts to overcome these imperfections were only partly successful. A comparison of the budgets of several 2nd moment equations with experimental results showed that the present model reproduces the equilibrium equation budgets quite accurately. A more serious problem was encountered in the statement of the lower boundary conditions downstream of the step change in surface characteristics. It was found that the condition of a constant surface relative humidity after the surface transition causes an unphysical solution for the change from a cool and humid terrain to a hot and dry one. A possible solution for this problem has been presented. Another problem, possibly related to the lower boundary conditions, emerged when the experimental

results of e.g. Lang et al. (1983) became available. It appears that the model in its present state is not able to reproduce the inequality of K_h and K_w in the IAL, found in the experiment. This problem remains to be solved. In the same chapter the model was modified and extended in order to be able to treat rough-to-smooth transitions. It was then possible to demonstrate that, when the ASL is subjected to a temporary change of the lower boundary conditions, the solution will eventually approach its original equilibrium.

The above results necessitated the analysis of the results of the application of the standard flux-profile methods on the equilibrium (or initial) profiles, generated with the initialisation part (part I) of the model. Chapter 5 is devoted to this subject. In Chapter 5 the so-called "second order flux-profile relations" were derived from the modeled 2nd order equations. This was done in order to be able to relate (in Chapter 6) the distribution of every term of the modeled 2nd order equations to the errors produced when applying the standard flux-profile relations in inhomogeneous conditions. We demonstrated that the 2nd order flux-profile relations are identical with the standard flux-profile relations in homogeneous and neutral conditions. When the ASL above homogeneous terrain has a diabatic stratification, the interpretation of the structure of the 2nd order flux-profile relations is rather difficult. It was shown that buoyancy effects enter these relations in two ways:

- (i) through the generation of an additional term in the relations, and
- (ii) through the modification of the factor in the relation which could be identified with the eddy diffusivity in neutral conditions.

It was found that the aerodynamic method yields fluxes which deviate considerably from the fluxes generated with the model. This is attributed to the disagreement between the dimensionless profile of the wind shear ($\partial U / \partial z$) and the expected ϕ_m -curve. This was discussed above and in Chapter 4. The Bowen ratio method is not sensible for the exact definition of ϕ_h and ϕ_w as long as

these functions are equal. In Chapter 5 it was found that the dimensionless equilibrium profiles $\partial\theta/\partial z$ and $\partial Q/\partial z$ are exactly equal. Hence, the application of the Bowen ratio method to the equilibrium profiles yields sensible- and latent heat fluxes which agree very well (within 5%) with the fluxes generated with the model.

In Chapter 6, both the Bowen ratio method and the aerodynamic method were applied to the relaxation profiles of U , θ and Q after the surface change. Both methods yield heat fluxes in the IAL which are larger than the heat fluxes calculated with the model (λE_{MD} , H_{MD}). The difference of the Bowen ratio heat fluxes (λE_{BR} , H_{BR}) and the model fluxes within the IAL is less than 10% if the region just downstream of the surface change ($x < 1$ m.) is disregarded. This difference decreases as the downstream distance increases. The analysis of the various terms of the \overline{wq} -equation downstream of the surface change, showed that the two advection terms ($U \frac{\partial \overline{wq}}{\partial x}$ and $W \frac{\partial \overline{wq}}{\partial z}$) and the modeled turbulent transport term are responsible for the differences mentioned. Of these three terms the horizontal advection term and the turbulent transport term nearly cancel each other. Hence, the relatively small vertical advection term is important in this respect. Above the IAL the difference in transport of water vapour and heat partly compensates the deviation caused by the three above mentioned terms. The results of the Bowen ratio method also indicate that the determination of the value of the heat fluxes at the surface is quite accurate (within 10%) even when this method is applied just above the IAL ($z \leq 1.5 \delta$).

As was mentioned earlier (Chapter 4) the present model is not capable to reproduce the ϕ_m -curve exactly. Because the application of the aerodynamic method critically depends on the shape of this curve, the analysis of the aerodynamic method is rather complicated and uncertain. Nevertheless, it was found that the difference of the heat fluxes produced by the aerodynamic method (λE_{AE} , H_{AE}) and the model fluxes within the IAL is larger than the difference of λE_{BR} (H_{BR}) and λE_{MD} (H_{MD}). This is attributed

to the more stringent conditions which the aerodynamic method has to satisfy. At $x = x_{\max}$ the IBL reaches the upper grid level. Beyond x_{\max} the upper boundary conditions contaminate the solution. Especially the aerodynamic method is susceptible to these errors, and a proper analysis cannot be made for these large downstream distances.

The final conclusion is that the present state of the second order model used in this study is subject to improvement, both in the modeling of the pressure terms and in the lower boundary conditions. Until a substantial improvement has been achieved, the main merit of this (and similar) models lies primarily in the qualitative prediction of the structure of the ASL after a change in surface conditions. Used in this way, it is an excellent tool for the understanding of the various processes which take place above an inhomogeneous terrain. This possibility of the application of a 2nd order model has been explored in this thesis for one specific purpose: the analysis of flux-profile relations in inhomogeneous conditions. The application of this model for quantitative purposes turned out to be quite hazardous. This is the sole reason why the second goal of this study has not been achieved. This application awaits the progress in the modeling of the higher order- and pressure terms. The problems concerned with the definition of the lower boundary conditions must be approached experimentally. The performance of an experiment, designed for the precise measurement of the processes which take place near the lower boundary, logically is the next step to the ultimate solution of this problem.

Appendix 1

DERIVATION OF GOVERNING EQUATIONS

A1.1 Conservation of mass equation

With the assumptions mentioned in Chapter 2 it was shown that the mass-conservation equation (2.2) transforms in

$$\frac{\partial \tilde{u}_i}{\partial x_i} = 0, \quad (\text{A1.1})$$

which, after applying Reynolds' decomposition yields

$$\frac{\partial U_i}{\partial x_i} = 0, \quad (\text{A1.2})$$

and

$$\frac{\partial u_i}{\partial x_i} = 0 \quad (\text{A1.3})$$

A1.2 The momentum equation

We start with the equations of motion (2.22)

$$\frac{\partial \tilde{u}_i}{\partial t} + \tilde{u}_j \frac{\partial \tilde{u}_i}{\partial x_j} = - \frac{1}{\rho_r} \frac{\partial \tilde{P}}{\partial x_i} + g_i \frac{\tilde{\theta}}{T_r} + \nu \frac{\partial^2 \tilde{u}_i}{\partial x_j \partial x_j} - 2\Omega_j \varepsilon_{ijk} \tilde{u}_k, \quad (\text{A1.4})$$

and introduce the Reynolds' decomposition (2.24a) into (A1.4)

$$\begin{aligned}
& \frac{\partial U_i}{\partial t} + \frac{\partial u_i}{\partial t} + U_j \frac{\partial U_i}{\partial x_j} + U_j \frac{\partial u_i}{\partial x_j} + u_j \frac{\partial U_i}{\partial x_j} + u_j \frac{\partial u_i}{\partial x_j} = \\
& - \frac{1}{\rho_r} \left(\frac{\partial p}{\partial x_i} + \frac{\partial p}{\partial x_i} \right) + g_i \left(\frac{\theta_v}{T_r} + \frac{\theta_v}{T_r} \right) + \nu \frac{\partial^2 U_i}{\partial x_j \partial x_j} + \nu \frac{\partial^2 u_i}{\partial x_j \partial x_j} + \\
& - 2\Omega_j \epsilon_{ijk} U_k - 2\Omega_j \epsilon_{ijk} u_k .
\end{aligned} \tag{A1.5}$$

Averaging (A1.5) and using (A1.3) the equation for the mean field is obtained

$$\begin{aligned}
& \frac{\partial U_i}{\partial t} + U_j \frac{\partial U_i}{\partial x_j} = - \frac{1}{\rho_r} \frac{\partial p}{\partial x_i} + g_i \frac{\theta_v}{T_r} - 2\Omega_j \epsilon_{ijk} U_k + \nu \frac{\partial^2 U_i}{\partial x_k \partial x_k} \\
& - \frac{\partial}{\partial x_j} (\overline{u_i u_j}) .
\end{aligned} \tag{A1.6}$$

Subtracting (A1.6) from (A1.5) yields the equation for the turbulent field

$$\begin{aligned}
& \frac{\partial u_i}{\partial t} + \frac{\partial}{\partial x_j} u_j U_i + U_j u_i + u_j u_i - \overline{u_i u_j} = - \frac{1}{\rho_r} \frac{\partial p}{\partial x_i} + g_i \frac{\theta_v}{T_r} + \\
& \nu \frac{\partial^2 u_i}{\partial x_k \partial x_k} - 2\Omega_j \epsilon_{ijk} u_k ,
\end{aligned} \tag{A1.7}$$

where (A1.2) and (A1.3) were used several times. Equation (A1.7) is needed for the derivation of the transport equations of $\overline{u_i u_j}$, $\overline{u_i \theta}$ and $\overline{u_i q}$ which will be done in section A1.4.

A1.3 The entropy equation

According to e.g. Lumley and Panofsky (1964) the following relation holds for the entropy (S)

$$ds = \left(\frac{\partial S}{\partial T} \right)_p dT + \left(\frac{\partial S}{\partial p} \right)_T dp = \frac{c_p}{T} dT - \left(\frac{\partial V}{\partial T} \right)_p dp = \frac{c_p}{T} dT - R \frac{dp}{p} . \tag{A1.8}$$

Substituting this into (2.3) and using (A1.1) we arrive at

$$\rho \frac{c}{T} \frac{\partial T}{\partial t} + \rho \frac{c}{T} u_j \frac{\partial T}{\partial x_j} - \frac{R}{P} \left(\frac{\partial P}{\partial t} + u_j \frac{\partial P}{\partial x_j} \right) = \kappa_H \frac{\partial^2 T}{\partial x_k \partial x_k}.$$

Starting with the decomposition (2.5), putting $\partial \tilde{P} / \partial t = 0$ and $\tilde{u}_j \frac{\partial \tilde{P}}{\partial x_j} = 0$, using $|\tilde{T}/T_r| \ll 1$, $|\tilde{P}/P_r| \ll 1$ and finally

dividing by ρc_p we obtain

$$\frac{\partial \tilde{T}}{\partial t} + \tilde{u}_j \frac{\partial \tilde{T}}{\partial x_j} + \tilde{u}_j \frac{\partial T_r}{\partial x_j} - \frac{R T_r}{c_p P_r} \tilde{u}_j \frac{\partial P_r}{\partial x_j} = \kappa_H \frac{\partial^2 \tilde{T}}{\partial x_k \partial x_k}. \quad (\text{A1.9})$$

If for $\partial P_r / \partial x_j$ and $\partial T_r / \partial x_j$ the relations (2.7) and (2.9) are substituted respectively and if also the equation of state (2.10) is applied, the third and fourth term in (A1.9) cancel each other. What remains is

$$\frac{\partial \tilde{T}}{\partial t} + \tilde{u}_j \frac{\partial \tilde{T}}{\partial x_j} = \kappa_H \frac{\partial^2 \tilde{T}}{\partial x_k \partial x_k},$$

which to a good approximation may be written as

$$\frac{\partial \tilde{\theta}}{\partial t} + \tilde{u}_j \frac{\partial \tilde{\theta}}{\partial x_j} = \kappa_H \frac{\partial^2 \tilde{\theta}}{\partial x_k \partial x_k}. \quad (\text{A1.10})$$

The next step, again, is the application of Reynolds' convention (2.24) to equation (A1.10). Analogous to the derivation in section A1.2 we obtain

$$\frac{\partial \theta}{\partial t} + U_j \frac{\partial \theta}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_j \theta}) + \kappa_H \frac{\partial^2 \theta}{\partial x_k \partial x_k}, \quad (\text{A1.11})$$

and

$$\frac{\partial \theta}{\partial t} + \frac{\partial}{\partial x_j} U_j \theta + u_j \theta + u_j \theta - \overline{u_j \theta} = \kappa_H \frac{\partial^2 \theta}{\partial x_k \partial x_k}, \quad (\text{A1.12})$$

for the mean and turbulent field respectively.

A1.4 The specific humidity equation

Without derivation the equations corresponding to (A1.11) and (A1.12) for the specific humidity are presented. These read for the mean and turbulent field respectively

$$\frac{\partial Q}{\partial t} + U_j \frac{\partial Q}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_j q}) + \kappa_v \frac{\partial^2 Q}{\partial x_k \partial x_k}, \quad (A1.13)$$

$$\frac{\partial q}{\partial t} + \frac{\partial}{\partial x_j} U_j q + u_j q + u_j Q - \overline{u_j q} = \kappa_v \frac{\partial^2 q}{\partial x_k \partial x_k}. \quad (A1.14)$$

A1.5 Derivation of the second moment equations

A1.5.1 The Reynolds' stress equation

Multiplying (A1.7) for u_i with u_k , adding (A1.7) for u_k multiplied with u_i and averaging the result yields

$$\begin{aligned} \frac{\partial}{\partial t} \overline{u_i u_k} + (\overline{u_j u_k} \frac{\partial \overline{u_i}}{\partial x_j} + \overline{u_j u_i} \frac{\partial \overline{u_k}}{\partial x_j}) + U_j \frac{\partial}{\partial x_j} \overline{u_i u_k} = \\ - (\overline{u_j u_k} \frac{\partial \overline{u_i}}{\partial x_j} + \overline{u_j u_i} \frac{\partial \overline{u_k}}{\partial x_j}) - \frac{1}{\rho_r} (\overline{u_k} \frac{\partial \overline{p}}{\partial x_i} + \overline{u_i} \frac{\partial \overline{p}}{\partial x_k}) + \\ + \frac{1}{T_r} (g_i \overline{u_k \theta_v} + g_k \overline{u_i \theta_v}) + (\overline{v u_k} \frac{\partial^2 \overline{u_i}}{\partial x_j \partial x_j} + \overline{v u_i} \frac{\partial^2 \overline{u_k}}{\partial x_j \partial x_j}) + \\ - 2\Omega_j (\epsilon_{ijl} \overline{u_l u_k} + \epsilon_{kjl} \overline{u_l u_i}), \end{aligned} \quad (A1.15)$$

where terms like $\overline{u_k \frac{\partial}{\partial x_j} \overline{u_i u_j}}$ and $\overline{u_i \frac{\partial}{\partial x_j} \overline{u_k u_j}}$ are equal to zero and can be omitted. Moreover, (A1.2) and (A1.3) were used several times.

The viscous terms in (A1.15) can be transformed as follows

$$\overline{v u_k} \frac{\partial^2 \overline{u_i}}{\partial x_j \partial x_j} + \overline{v u_i} \frac{\partial^2 \overline{u_k}}{\partial x_j \partial x_j} = \nu \frac{\partial}{\partial x_j} (\overline{u_k} \frac{\partial \overline{u_i}}{\partial x_j}) - \nu (\overline{\frac{\partial u_k}{\partial x_j} \frac{\partial u_i}{\partial x_j}} +$$

$$\begin{aligned} \nu \frac{\partial}{\partial x_j} \overline{\left(u_i \frac{\partial u_k}{\partial x_j} \right)} - \nu \overline{\left(\frac{\partial u_i}{\partial x_j} \frac{\partial u_k}{\partial x_j} \right)} &= \nu \frac{\partial}{\partial x_j} \overline{\left(\frac{\partial u_i u_k}{\partial x_j} \right)} - \nu \frac{\partial}{\partial x_j} \overline{\left(u_i \frac{\partial u_k}{\partial x_j} \right)} + \\ &+ \nu \frac{\partial}{\partial x_j} \overline{\left(u_i \frac{\partial u_k}{\partial x_j} \right)} - 2\nu \overline{\frac{\partial u_i}{\partial x_j} \frac{\partial u_k}{\partial x_j}}. \end{aligned}$$

This yields

$$\nu u_k \frac{\partial^2 u_i}{\partial x_j \partial x_j} + \nu u_i \frac{\partial^2 u_k}{\partial x_j \partial x_j} = \nu \frac{\partial^2 \overline{u_i u_k}}{\partial x_j \partial x_j} - 2\nu \overline{\frac{\partial u_i}{\partial x_j} \frac{\partial u_k}{\partial x_j}}. \quad (\text{A1.16})$$

The first term on the right hand side of (A1.16) represents molecular diffusion of $\overline{u_i u_k}$ and can be neglected in high Re-number flows (Wyngaard, 1982; Hinze, 1959). The second term on the right hand side of (A1.16) represents molecular destruction and cannot be ignored. If isotropy of the small scale structure of turbulence is assumed then

$$2\nu \overline{\frac{\partial u_i}{\partial x_j} \frac{\partial u_k}{\partial x_j}} = -\frac{2}{3} \nu \overline{\left(\frac{\partial u_i}{\partial x_j} \right)^2} \delta_{ik} = -\frac{2}{3} \overline{\epsilon} \delta_{ik}. \quad (\text{A1.17})$$

This implies that molecular destruction only affects the normal components of the Reynolds' stress tensor i.e. the variances $\overline{u_i u_i}$.

With this transformation of the viscous terms, (A1.15) reads

$$\begin{aligned} \frac{\partial}{\partial t} \overline{u_i u_k} + \overline{\left(u_j u_k \frac{\partial u_i}{\partial x_j} + u_j u_i \frac{\partial u_k}{\partial x_j} \right)} + U_j \frac{\partial}{\partial x_j} \overline{u_i u_k} = \\ - \frac{\partial}{\partial x_j} \overline{\left(u_i u_j u_k \right)} - \frac{1}{\rho_r} \overline{\left(u_k \frac{\partial p}{\partial x_i} + u_i \frac{\partial p}{\partial x_k} \right)} + \frac{1}{T_r} \overline{\left(g_i u_k \theta_v \right)} \\ + g_k \overline{u_i \theta_v} - \frac{2}{3} \overline{\epsilon} \delta_{ik} - 2\eta_j (\epsilon_{ijl} \overline{u_l u_k} + \epsilon_{kjl} \overline{u_l u_i}), \quad (\text{A1.18}) \end{aligned}$$

where (A1.3) has been used once again to obtain the third

order term $\frac{\partial}{\partial x_j} (\overline{u_j u_i u_k})$

A1.5.2 The turbulent flux equations

The result of multiplying (A1.7) with θ , (A1.12) with u_i , adding and averaging is

$$\begin{aligned} \frac{\partial}{\partial t} \overline{u_i \theta} + u_j \frac{\partial}{\partial x_j} \overline{u_i \theta} + (\overline{u_i u_j} \frac{\partial \theta}{\partial x_j} + \overline{u_j \theta} \frac{\partial u_i}{\partial x_j}) = - \frac{\partial}{\partial x_j} (\overline{u_i u_j \theta}) - \\ \frac{1}{\rho_r} \overline{\theta \frac{\partial p}{\partial x_i}} + g_i \overline{\frac{\theta \theta}{T_r} v} + v \overline{\theta \frac{\partial^2 u_i}{\partial x_j \partial x_j}} + \kappa_H \overline{u_i \frac{\partial^2 \theta}{\partial x_j \partial x_j}} - \\ 2\Omega_j \epsilon_{ijk} \overline{\theta u_k} . \end{aligned} \quad (A1.19)$$

The molecular terms can be written as

$$v \overline{\theta \frac{\partial^2 u_i}{\partial x_j \partial x_j}} = v \overline{\frac{\partial^2 \theta u_i}{\partial x_j \partial x_j}} - v \overline{\frac{\partial}{\partial x_j} (u_i \frac{\partial \theta}{\partial x_j})} - v \overline{\frac{\partial \theta}{\partial x_j} \frac{\partial u_i}{\partial x_j}} , \quad (A1.20)$$

and

$$\kappa_H \overline{u_i \frac{\partial^2 \theta}{\partial x_j \partial x_j}} = \kappa_H \overline{\frac{\partial^2 u_i \theta}{\partial x_j \partial x_j}} - \kappa_H \overline{\frac{\partial}{\partial x_j} (\theta \frac{\partial u_i}{\partial x_j})} - \kappa_H \overline{\frac{\partial \theta}{\partial x_j} \frac{\partial u_i}{\partial x_j}} . \quad (A.21)$$

The first term on the right hand side of (A1.20) and (A1.21) represents molecular diffusion of $\overline{u_i \theta}$ and can be neglected in flows with a high Re and Pe number. According to Wyngaard (1982) the second term in both equations is negligible also. Hence only the third term remains. However, if isotropy of the smallest scales is assumed, these terms, because of their odd number of indices, must also equal zero. This means that the molecular terms in (A1.19) disappear altogether. Thus

$$\frac{\partial}{\partial t} \overline{u_i \theta} + u_j \frac{\partial}{\partial x_j} \overline{u_i \theta} + (\overline{u_i u_j} \frac{\partial \theta}{\partial x_j} + \overline{u_j \theta} \frac{\partial u_i}{\partial x_j}) =$$

$$- \frac{\partial}{\partial x_j} (\overline{u_i u_j \theta}) - \frac{1}{\rho_r} \overline{\theta \frac{\partial p}{\partial x_i}} + g_i \frac{\overline{\theta \theta}}{T_r} - 2 \Omega_j \epsilon_{ijk} \overline{u_k \theta}, \quad (A1.22)$$

The derivation of the humidity flux transport equation is analogous to the derivation given above.

A1.5.3 The other (co)variance equations

The derivation of the $\overline{\theta^2}$, $\overline{q^2}$ and $\overline{q\theta}$ equations is straightforward. For the $\overline{\theta^2}$ equation we multiply (A1.12) with θ and average, this yields

$$\frac{1}{2} \frac{\partial}{\partial t} \overline{\theta^2} + \frac{1}{2} U_j \frac{\partial \overline{\theta^2}}{\partial x_j} + \frac{1}{2} \frac{\partial}{\partial x_j} (\overline{u_j \theta^2}) + \overline{u_j \theta} \frac{\partial \theta}{\partial x_j} = \kappa_H \overline{\theta \frac{\partial^2 \theta}{\partial x_k \partial x_k}}, \quad (A1.23)$$

where the molecular term must be transformed. To this end we write

$$\begin{aligned} \kappa_H \overline{\theta \frac{\partial^2 \theta}{\partial x_k \partial x_k}} &= \kappa_H \frac{\partial}{\partial x_k} \left(\overline{\theta \frac{\partial \theta}{\partial x_k}} \right) - \kappa_H \overline{\left(\frac{\partial \theta}{\partial x_k} \right)^2} = \\ &= \frac{1}{2} \kappa_H \frac{\partial \overline{\theta^2}}{\partial x_k \partial x_k} - \kappa_H \overline{\left(\frac{\partial \theta}{\partial x_k} \right)^2}. \end{aligned}$$

The first term on the right hand side represents molecular diffusion again and can be neglected. The second term cannot be neglected, even when isotropy of the smallest scales is assumed. Thus

$$\kappa_H \overline{\theta \frac{\partial^2 \theta}{\partial x_k \partial x_k}} = - \kappa_H \overline{\left(\frac{\partial \theta}{\partial x_k} \right)^2} = - \overline{\epsilon_\theta}. \quad (A1.24)$$

Equation (A1.23) then reads

$$\frac{\partial}{\partial t} \overline{\theta^2} + U_j \frac{\partial \overline{\theta^2}}{\partial x_j} + 2 \overline{u_j \theta} \frac{\partial \theta}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_j \theta^2}) - 2 \overline{\epsilon_\theta}. \quad (A1.25)$$

For the $\overline{q^2}$ and $\overline{q\theta}$ equation we find accordingly

$$\frac{\partial}{\partial t} \overline{q^2} + u_j \frac{\partial \overline{q^2}}{\partial x_j} + 2 \overline{u_j q} \frac{\partial Q}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_j q^2}) - 2 \overline{\varepsilon_q}, \quad (\text{A1.26})$$

and

$$\frac{\partial}{\partial t} q\theta + u_j \frac{\partial \overline{q\theta}}{\partial x_j} + \overline{u_j q} \frac{\partial \theta}{\partial x_j} + \overline{u_j \theta} \frac{\partial Q}{\partial x_j} = - \frac{\partial}{\partial x_j} (\overline{u_j q\theta}) - \overline{\varepsilon_{q\theta}}, \quad (\text{A1.27})$$

where

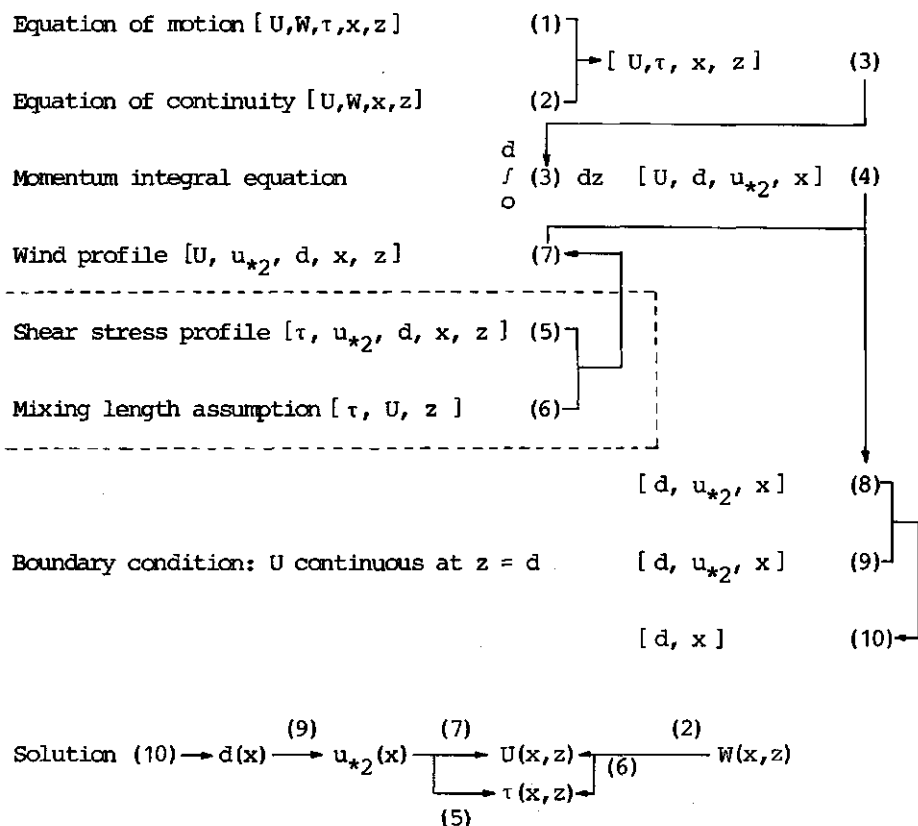
$$\overline{\varepsilon_q} = - \kappa_v \left(\overline{\frac{\partial q}{\partial x_k}} \right)^2, \quad \text{and} \quad \overline{\varepsilon_{q\theta}} = - (\kappa_v + \kappa_H) \overline{\frac{\partial q}{\partial x_k} \frac{\partial \theta}{\partial x_k}}. \quad (\text{A1.28})$$

a,b)

Appendix 2

SOLUTION PROCEDURE FOR INTEGRAL METHODS

If square brackets are used to denote a relation between the dependent and independent variables placed within these brackets, the solution procedure used in integral models can be summarized as follows:



Using the equation of continuity (2), it is possible to eliminate the vertical velocity W from the equation of motion (1), resulting in eq. (3). If this equation is integrated over the depth d of the IBL, the shear stress at $z = 0$ (u_{*2}) and the

IBL height enter the analysis in (4). When a prescribed form of the horizontal velocity U is used (7) we are able to eliminate U as a dependent variable from (4) resulting in (8). Finally, if the boundary condition of a continuous velocity at $z = d$ (9) is used, two equations (9) and (8) with two unknowns (d and u_{*2}) are obtained. Combining these equations will generate a differential equation in d (10) which can be solved either analytically or numerically, giving d as a function of x .

Substituting backwards will result in the other dependent variables as shown in the bottom line diagram.

We did not include the parameters z_{01} , z_{02} and u_{*1} in these diagrams. The part of the diagram behind the dashed line indicates that the prescription of a shear stress profile within the IBL is, as Peterson (1969) notices, a fourth equation, which creates a system with more equations than unknowns. But as Taylor (1969) notices, the mixing length hypothesis is necessary to obtain the vertical distribution of the shear stress.

Appendix 3

SUMMARY OF CLOSURE APPROXIMATIONS AND BOUNDARY CONDITIONS

A3.1 Closure approximation and modeled equations

Although a full description of the closure approximations in the present model is given by Rao et al. (1974b) they are summarized in this Appendix for completeness and easy reference. Some closure approximations were modified in the course of this study and these modified expressions are also presented here. As $k = 0.41$ was used instead of $k = 0.35$ (Rao et al., 1974b), the value of various constants in the closure approximations is changed accordingly.

Turbulent transport terms

The turbulent transport terms are approximated by a gradient transport model (see Chapter 4):

$$\overline{Mu_i} = -a_t \frac{\partial M}{\partial x_j} \overline{u_i u_j} \tau \quad (A3.1)$$

where M can be $\overline{\theta u_j}$, $\overline{qu_j}$, $\overline{\theta^2}$, $\overline{q^2}$, $\overline{q\theta}$ or $\overline{\epsilon}$ and $\tau = \overline{u_i u_i} / \overline{\epsilon}$ is a turbulent time scale. The third moment $\overline{u_i u_j u_k}$ in the $\overline{u_i u_k}$ equation is expressed as

$$\overline{(u_i u_j + \frac{2}{3} p \delta_{ij}) u_k} = a_t \frac{\partial (-\overline{u_i u_j})}{\partial x_m} \overline{u_m u_k} \tau \quad (A3.2)$$

where a_t in both equations is a transport constant. In the original model $a_t = 0.15$ was used, except in the $\overline{\epsilon}$ -equation, where the surface layer constraints require $a_t = (4-4a)/16.6$ (see Wyngaard et al., 1974a).

Pressure covariance terms

Rao et al. (1974b) modeled the pressure covariance terms in Eqs. (4.1)-(4.3) respectively by

$$\begin{aligned}
& - (\overline{u_k \frac{\partial p}{\partial x_i}} + \overline{u_i \frac{\partial p}{\partial x_k}}) + 2 \frac{\partial}{\partial x_j} (\overline{u_j p}) \frac{\delta_{ik}}{3} = \\
& - (\overline{u_i u_k} - \overline{u_i u_i} \frac{\delta_{ik}}{3}) \frac{c_{ik}}{\tau}
\end{aligned} \tag{A3.3}$$

$$- \overline{\theta \frac{\partial p}{\partial x_i}} = - d_i \frac{\overline{\theta u_i}}{\tau} \tag{A3.4}$$

$$- \overline{q \frac{\partial p}{\partial x_i}} = - d_i \frac{\overline{q u_i}}{\tau} \tag{A3.5}$$

where the constants $c_{ii} = 6.7$, $c_{ik} (i \neq k) = 13.1$, $d_1 = 5.0$ and $d_3 = 13.1$. In these modeling approximations no mean strain and buoyancy effects have been incorporated. The extension to account for these effects is given by Wyngaard (1975) and was added in a later version of the model in this study, hence

$$- (\overline{u_k \frac{\partial p}{\partial x_i}} + \overline{u_i \frac{\partial p}{\partial x_k}}) + \frac{2}{3} \frac{\partial}{\partial x_j} (\overline{u_j p}) \delta_{ik} = A_{ik} \tag{A3.6}$$

where

$$\begin{aligned}
A_{ik} = & - \frac{C}{\tau} (\overline{u_i u_k} - \overline{u_i u_i} \frac{\delta_{ik}}{3}) + \frac{1}{2} C_1 \overline{u_i u_i} (\frac{\partial U_i}{\partial x_k} + \frac{\partial U_k}{\partial x_i}) + \\
& + C_2 (\overline{u_i u_j} \frac{\partial U_k}{\partial x_j} + \overline{u_k u_j} \frac{\partial U_i}{\partial x_j} - \frac{2}{3} \delta_{ik} \overline{u_m u_j} \frac{\partial U_j}{\partial x_m}) + \\
& + C_3 (\frac{g_i}{T_r} \overline{\theta u_k} + \frac{g_k}{T_r} \overline{\theta u_i} - \frac{2}{3} \frac{g_j}{T_r} \overline{\theta u_j} \delta_{ik})
\end{aligned} \tag{A3.7}$$

The C-term is identical to the right hand side of eq. (A3.3). The C_1 and C_2 terms represent mean strain effects, while the C_3 term parameterizes the buoyancy effects. Applying the lower boundary neutral surface layer limit, these constants are related by:

$$\begin{aligned} C &= 6.67 (1-C_2) \\ C_1 &= 0.23 (1-C_2) \end{aligned} \quad (A3.8)$$

Unfortunately, Wyngaard (1975) only presents values of the various C's for the stable surface layer. For the turbulent heat and humidity flux, the approximation read respectively:

$$\overline{\theta \frac{\partial p}{\partial x_i}} = d_i \frac{\overline{\theta u_i}}{\tau} + a_1 \frac{g_i}{T_r} \overline{\theta \theta_v} + a_2 \frac{\partial U_i}{\partial x_j} \overline{\theta u_j} \quad (A3.9)$$

and

$$\overline{q \frac{\partial p}{\partial x_i}} = d_i \frac{\overline{q u_i}}{\tau} + a_1 \frac{g_i}{T_r} \overline{q \theta_v} + a_2 \frac{\partial U_i}{\partial x_j} \overline{q u_j} \quad (A3.10)$$

where (A3.9) has been slightly modified with respect to the expression given by Wyngaard (1975) to account for humidity.

The a_1 and a_2 terms parameterize buoyancy and mean strain effects respectively. The value of d_i is the same as in Eqs. (A3.4) and (A3.5), while $a_2 = -0.5$ was used as suggested by Launder (1975). The value of a_1 is determined by the gradient Richardson number:

$$\begin{aligned} a_1 &= 0.5 && \text{if } R_i < 0 \\ a_1 &= 0.5 + 1.5 R_i^2 - R_i^3 && \text{if } 0 < R_i < 1 \\ a_1 &= 1.0 && \text{if } R_i > 1 \end{aligned} \quad (A3.11)$$

$$\text{where } Ri = (g/T) (\partial \theta / \partial z) / (\partial U / \partial z)^2$$

Modeled equations

The modeling expressions originally given by Rao et al. (1974a) presented in this Appendix are now applied to Eqs. (4.1)-(4.6) and (4.14). If homogeneity is assumed i.e. $\partial / \partial x = 0$ and, by virtue of the equation of continuity, also $W = 0$, the following equations apply to the various turbulence quantities:

$$2 \overline{uw} \frac{\partial U}{\partial z} = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{u^2}}{\partial z}) - \left[\overline{u^2} - \frac{1}{3} (\overline{u^2} + \overline{v^2} + \overline{w^2}) \right] \frac{c_{11}}{\tau} - \frac{2}{3} \overline{\epsilon} \quad (\text{A3.12})$$

$$0 = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{v^2}}{\partial z}) - \left[\overline{v^2} - \frac{1}{3} (\overline{u^2} + \overline{v^2} + \overline{w^2}) \right] \frac{c_{22}}{\tau} - \frac{2}{3} \overline{\epsilon} \quad (\text{A3.13})$$

$$-2 \frac{g}{T_r} \overline{w\theta}_v = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{w^2}}{\partial z}) - \left[\overline{w^2} - \frac{1}{3} (\overline{u^2} + \overline{v^2} + \overline{w^2}) \right] \frac{c_{33}}{\tau} - \frac{2}{3} \overline{\epsilon} \quad (\text{A3.14})$$

$$\overline{w^2} \frac{\partial U}{\partial z} - \frac{g}{T_r} \overline{u\theta}_v = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{uw}}{\partial z}) - \overline{uw} \frac{c}{\tau} \quad (\text{A3.15})$$

$$\overline{w\theta} \frac{\partial U}{\partial z} + \overline{uw} \frac{\partial \theta}{\partial z} = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{u\theta}}{\partial z}) - \overline{u\theta} \frac{d_1}{\tau} \quad (\text{A3.16})$$

$$\overline{w^2} \frac{\partial \theta}{\partial z} - \frac{g}{T_r} \overline{\theta\theta}_v = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{w\theta}}{\partial z}) - \overline{w\theta} \frac{d_3}{\tau} \quad (\text{A3.17})$$

$$\overline{wq} \frac{\partial U}{\partial z} + \overline{uw} \frac{\partial Q}{\partial z} = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{uq}}{\partial z}) - \overline{uq} \frac{d_1}{\tau} \quad (\text{A3.18})$$

$$\overline{w^2} \frac{\partial Q}{\partial z} - \frac{g}{T_r} \overline{q\theta}_v = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{wq}}{\partial z}) - \overline{wq} \frac{d_3}{\tau} \quad (\text{A3.19})$$

$$\overline{wq} \frac{\partial \theta}{\partial z} + \overline{w\theta} \frac{\partial Q}{\partial z} = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{q\theta}}{\partial z}) - \overline{q\theta} \frac{2b}{\tau} \quad (\text{A3.20})$$

$$2 \overline{w\theta} \frac{\partial \theta}{\partial z} = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{\theta^2}}{\partial z}) - \overline{\theta^2} \frac{2b}{\tau} \quad (\text{A3.21})$$

$$2 \overline{wq} \frac{\partial Q}{\partial z} = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{q^2}}{\partial z}) - \overline{q^2} \frac{2b}{\tau} \quad (\text{A3.22})$$

$$\frac{-4 \overline{\epsilon}}{\overline{u^2} + \overline{v^2} + \overline{w^2}} (\overline{\epsilon} - aP) = -a_t \frac{\partial}{\partial z} (\overline{w^2} \tau \frac{\partial \overline{\epsilon}}{\partial z}) \quad (\text{A3.23})$$

A3.2 Summary of boundary conditions

The lower boundary conditions used in the model of Rao et al. (1974b) were based on the equilibrium flux-profile relations defined by the Kansas experiment results (Businger et al., 1971). In the present study modified expressions are used in accordance with $k = 0.41$ instead of $k = 0.35$. Thus:

at $z = z_{0i}$ ($x < 0$, upstream conditions: $i = 1$)

$$U = W = 0, \quad \theta = \theta_0, \quad Q = Q_0,$$

$$\frac{kz_{0i}}{u_{*i}} \frac{\partial U}{\partial z} = \frac{kz_{0i}}{T_{*i}} \frac{\partial \theta}{\partial z} = \frac{kz_{0i}}{q_{*i}} \frac{\partial Q}{\partial z} = 1,$$

$$\overline{uw}/u_{*i}^2 = -1.0, \quad \overline{u^2}/u_{*i}^2 = 4.0, \quad \overline{v^2}/u_{*i}^2 = \overline{w^2}/u_{*i}^2 = 1.75$$

$$\overline{w\theta}/u_{*i}T_{*i} = \overline{wq}/u_{*i}q_{*i} = -1.0, \quad \overline{u\theta}/u_{*i}T_{*i} = \overline{uq}/u_{*i}q_{*i} = 4.0,$$

$$\overline{\theta^2}/T_{*i}^2 = \overline{q^2}/q_{*i}^2 = \overline{q\theta}/q_{*i}T_{*i} = 4.0, \quad \frac{kz_0}{u_{*i}^3} \overline{\epsilon} = 1.0 \quad (A3.24)$$

In part II of the model ($x > 0$, downstream conditions: $i = 2$) the value of $u_{*2}(x)$, $T_{*2}(x)$ and $q_{*2}(x)$ is determined by $U(x, z)$, $\theta(x, z)$ and $Q(x, z_2)$ at the lowest two levels (z_1 and z_2) where the lowest level z_1 coincides with the surface roughness height z_{02} . The value of $U(x, z_2)$, $\theta(x, z_2)$ and $Q(x, z_2)$ is calculated by the model using the finite difference computation scheme. But at $z = z_1 = z_{02}$ only U is known, as by definition $U(x, z_1) = 0$. This allows the computation of $u_{*2}(x)$ by means of (5.4). The boundary value of the temperature and specific humidity must somehow be obtained by inferring equations relating these quantities to others already known. Rao et al (1974b) used the energy balance equation

$$H_{02}(x) + \lambda E_{02}(x) = R_n - G \quad (A3.25)$$

with a predetermined constant value of $R_n - G$. That the available energy is constant throughout the flow, (either for $x < 0$ and $x > 0$), seems questionable but appears to be a reasonable assumption. Experimental data (Lang, et al., 1983) indicate that this is, at least within some 10% a good approximation. Using the equilibrium flux-profile relations, (A3.25) can be expressed in terms of mean quantities

$$\left(\frac{\partial \theta}{\partial z}\right)_{z_{02}} + \frac{\lambda}{c_p} \left(\frac{\partial Q}{\partial z}\right)_{z_{02}} = \frac{u_{*1} (T_{*1} + \frac{\lambda}{c_p} q_{*1})}{kz_{02} u_{*2}(x)} \quad (A3.26)$$

where $u_{*2}(x) = kz_{02} \left(\frac{\partial U}{\partial z}\right)_{z_{02}}$ and the gradients of Q and θ are approximated by finite differences:

$$\left(\frac{\partial Q}{\partial z}\right)_{z_{02}} = \frac{Q(x, z_2) - Q(x, z_1)}{z_2 - z_1},$$

and

$$\left(\frac{\partial \theta}{\partial z}\right)_{z_{02}} = \frac{\theta(x, z_2) - \theta(x, z_1)}{z_2 - z_1}.$$

A second equation is necessary to solve (A3.26) for $\theta(x, z_1)$ and $Q(x, z_1)$. Rao et al. postulated that

$$Q(x, z_1) = R_{02} \cdot Q_s(\theta(x, z_1)) \quad (A3.27)$$

where R_{02} is the surface relative humidity, which is assumed to be constant for $x > 0$. $Q_s(\theta(x, z_1))$ is the saturation specific humidity for the temperature at $z = z_1 = z_{02}$. Solving (A3.26) and (A3.27) for $\theta(x, z_1)$ and $Q(x, z_1)$ enables the calculation of $T_{*2}(x)$ and $q_{*2}(x)$ which, together with $u_{*2}(x)$ determine the lower

boundary condition for every variable by means of (A3.24). Instead of using (A3.27), an alternative method to close (A3.26) was suggested by McNaughton (1984) and is discussed in Appendix 4.

The upper boundary conditions in Part I ($x < 0$, upstream situation) are also determined by the existing flux-profile relations and the predetermined height of the atmospheric boundary layer height (z_I).

At $z = z_{\max}$ ($x < 0$, upstream conditions: $i = 1$)

$$\frac{k z_{\max}}{u_{*i}} \frac{\partial U}{\partial z} = \phi_m \left(\frac{z_{\max}}{L} \right), \quad \frac{k z_{\max}}{T_{*i}} \cdot \frac{\partial \theta}{\partial z} = \phi_h \left(\frac{z_{\max}}{L} \right),$$

$$\frac{k z_{\max}}{q_{*i}} \frac{\partial Q}{\partial z} = \phi_w \left(\frac{z_{\max}}{L} \right)$$

$$\overline{u^2}/u_{*i}^2 = 0.2(w_*^2/u_{*i}^2) + 4,$$

$$\overline{v^2}/u_{*i}^2 = 0.2(w_*^2/u_{*i}^2) + 1.75,$$

$$\overline{w^2}/u_{*i}^2 = 2 \left(- \frac{z_{\max}}{L} \right)^{2/3} + 1.75$$

$$\overline{w\theta}/u_{*i} T_{*i} = \overline{wq}/u_{*i} q_{*i} = \overline{uw}/u_{*i}^2 = \frac{z_{\max}}{z_I} - 1$$

$$\overline{u\theta}/u_{*i} T_{*i} = \overline{uq}/u_{*i} q_{*i} = 4 \phi_m \phi_h$$

$$\overline{\theta^2}/T_{*i}^2 = \overline{q^2}/q_{*i}^2 = \overline{q\theta}/q_{*i} T_{*i} = 4 \left(1 - 8.3 \frac{z_{\max}}{L} \right)^{-2/3}$$

$$\overline{\epsilon} = - \overline{uw} \frac{\partial U}{\partial z} + \frac{g}{T_r} \overline{w\theta} \quad (A3.28)$$

In part II ($x > 0$, downstream conditions) the upper boundary value of every variable, except W , remains fixed, i.e. stationary conditions aloft are assumed. Gradients are supposed to be very small hence the vertical windspeed W does not substantially affect the upper boundary values. This implies that calculations cannot be extended beyond the value of x where the IBL coincides with the highest calculation level.

Appendix 4

RECONSIDERATION OF THE LOWER BOUNDARY CONDITION

In order to avoid the, rather unrealistic, assumption that R_{02} has a constant value for $x > 0$, and thus discarding Eq. (A3.27) McNaughton (1984) suggested the following procedure.

Use the Penman-Monteith equation to calculate the latent heat flux density

$$\lambda E = \frac{\left(\frac{s}{s+\gamma}\right) (R_n - G) + \frac{\rho c_p}{s+\gamma} \cdot \frac{1}{r_a} [Q_s(\theta(x, z_2)) - Q(x, z_2)]}{1 + \left(\frac{\gamma}{s+\gamma}\right) r_s/r_a} \quad (\text{A4.1})$$

where r_a is the aerodynamic resistance,

r_s is the surface resistance,

s is the slope of the saturation specific humidity curve

and $\gamma = c_p/\lambda$.

Multiplying this equation by $\left(\frac{s+\gamma}{\gamma}\right) r_a$ yields

$$\lambda E = \frac{\left(\frac{s}{\gamma}\right) (R_n - G) r_a + \rho \lambda [Q_s(\theta(x, z_2)) - Q(x, z_2)]}{(1 + s/\gamma) r_a + r_s} \quad (\text{A4.2})$$

The aerodynamic resistance r_a is obtained from the wind profile by

$$r_a = U(x, z_2) / u_*^2(x) \quad (\text{A4.3})$$

This leaves r_s undetermined, and hence a new independent parameter is introduced.

The latent heat flux density at each step downstream is calculated by solving Eqs. (A4.2) and (A4.3), using an empirical function for $Q_s(\theta)$. The energy balance (A3.25) provides the accompanying sensible heat flux density. The value of $Q(x, z_1)$ and $\theta(x, z_1)$ is obtained by using

$$H = \rho c_p (\theta(x, z_1) - \theta(x, z_2)) / r_a \quad (\text{A4.4})$$

and

$$\lambda E = \rho \lambda (Q(x, z_1) - Q(x, z_2)) / r_a \quad (\text{A4.5})$$

In daytime situations when $R_n - G$ is positive, all variables in (A4.2) are positive and hence a negative value of the latent heat flux density λE is precluded.

The boundary conditions (A4.2)-(A4.5) were inserted into the present model and the same upstream situation as discussed in Chapter 4 was used (Figs. 4.10 and 4.11). The value of the surface resistance r_s for the upstream terrain was calculated from the equilibrium profiles and amounted to 37 s/m. As the air moves over a warmer, dryer and rougher surface downstream, the latent heat flux and the aerodynamic resistance (r_a) must decrease. According to (A4.2) the value of r_s must be greater than 37 s/m. For this experiment r_s was set to 5000 s/m.

The results, as far as the temperature and the absolute humidity are concerned, are presented in Figs. (A4.1) and (A4.2). Fig. (A4.1) clearly shows that the gradient of the absolute humidity is negative everywhere, hence the latent heat flux is directed upwards throughout the flow. The negative (i.e. downwards) latent heat flux in Fig. (4.10) releases latent heat at the surface which is physically unrealistic. Because of this additional input of energy at the surface, the surface temperature must increase substantially. This is visible in Fig. (4.11). In the present situ-

ation no latent heat is released at the surface and hence the increase of the surface temperature after $x = 0$ will be much smaller.

Though the problem of the negative latent heat flux has been effectively eliminated by the procedure outlined in this appendix, one problem still remains. Instead of using the assumption of a constant relative humidity at the surface it is now assumed that the surface resistance r_s remains unchanged. Although, on observing Eq. (A4.2), the solution is not very sensitive for this assumption, it remains in doubt. Furthermore, the application of equation (A4.2) implies that the surface is at saturation point. In this example the Penman-Monteith equation is used above a very dry surface, this seems contradictory.

Finally, for a change from a dry, warm surface to a cool and humid one, the problem of a downwards latent heat flux will not arise. In that case the sensible heat flux will be directed downwards, this is not an uncommon feature.

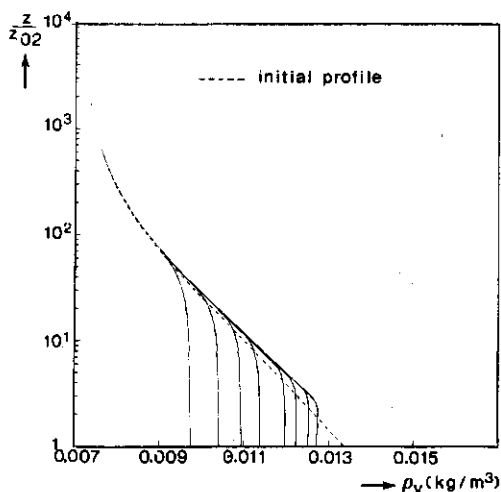


Figure A4.1 Absolute humidity profiles after a change in surface conditions for various distances downstream.

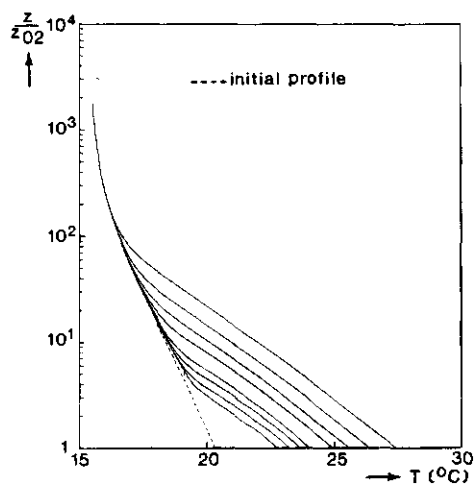


Figure A4.2 Temperature profiles after a change in surface conditions for various distances downstream.

Appendix 5

TRANSFORMATION OF THE SECOND ORDER FLUX-PROFILE RELATIONS

We want to transform the flux profile relations, derived from the modeled transport equations of the second moments, into relations resembling the "standard" flux-profile relations (5.22)-(5.24). The additional equations for $\overline{u\theta}$, $\overline{\theta^2}$ and $\overline{q\theta}$ (A3.16), (A3.21) and (A3.20) are then needed.

The following two assumptions are made:

(i) the buoyancy terms $\frac{g}{T_r} \frac{\overline{m\theta}}{w^2} v$ in (5.28)-(5.30) can be approximated by $\frac{g}{T_r} \frac{\overline{m\theta}}{w^2}$ ($m = u, \theta$ or q). Thus the influence of water vapour on the buoyancy term will be neglected.

(ii) the turbulent transport term in the equation for $\overline{u\theta}$, $\overline{\theta^2}$ and $\overline{q\theta}$ is negligibly small. This can be inferred from Figs. 4.3 and 4.4 for $\overline{\theta^2}$ and $\overline{u\theta}$, respectively. Calculations show that this also applies for the $\overline{q\theta}$ equation.

Hence we start with the following equations:

$$\overline{uw} = - \frac{\overline{w^2} \tau}{c_{13}} \left(\frac{\partial U}{\partial z} - \frac{g}{T_r} \frac{\overline{u\theta}}{w^2} \right), \quad (A5.1)$$

$$\overline{w\theta} = - \frac{\overline{w^2} \tau}{d_3} \left(\frac{\partial \theta}{\partial z} - \frac{g}{T_r} \frac{\overline{\theta^2}}{w^2} \right), \quad (A5.2)$$

$$\overline{wq} = - \frac{\overline{w^2} \tau}{d_3} \left(\frac{\partial Q}{\partial z} - \frac{g}{T_r} \frac{\overline{q\theta}}{w^2} \right), \quad (A5.3)$$

$$\overline{u\theta} = - \frac{\tau}{d_1} \left(\overline{w\theta} \frac{\partial U}{\partial z} + \overline{uw} \frac{\partial \theta}{\partial z} \right), \quad (A5.4)$$

$$\overline{\theta^2} = - \frac{\tau}{b} \left(\overline{w\theta} \frac{\partial \theta}{\partial z} \right), \quad (A5.5)$$

$$\overline{q\theta} = -\frac{\tau}{2b}(\overline{wq} \frac{\partial \theta}{\partial z} + \overline{w\theta} \frac{\partial q}{\partial z}), \quad (\text{A5.6})$$

where $c_{13} = d_3 = c$.

Substitute $\overline{\theta^2}$ (A5.5) into (A5.2), this results in:

$$\overline{w\theta} = -\frac{\overline{w^2}\tau}{c} \frac{\partial \theta}{\partial z} / (1 + \frac{\tau^2}{bc} \frac{g}{T_r} \frac{\partial \theta}{\partial z}). \quad (\text{A5.7})$$

Substitute $\overline{u\theta}$ (A5.4) and subsequently $\overline{w\theta}$ (A5.7) into (A5.1). This yields after rearranging:

$$\overline{uw} = -\frac{\overline{w^2}\tau}{c} \frac{\partial u}{\partial z} \frac{1 - P_1/(1 + P_2)}{1 + P_1}, \quad (\text{A5.8})$$

$$\text{where } P_1 = \frac{\tau^2}{d_1 c} \frac{g}{T_r} \frac{\partial \theta}{\partial z},$$

$$\text{and } P_2 = \frac{\tau^2}{bc} \frac{g}{T_r} \frac{\partial \theta}{\partial z}.$$

Analogously we will find for \overline{wq} after substituting $\overline{q\theta}$ (A5.6) and $\overline{w\theta}$ (A5.7):

$$\overline{wq} = -\frac{\overline{w^2}\tau}{c} \frac{\partial q}{\partial z} \frac{1 - P_3/(1 + P_2)}{1 + P_3}, \quad (\text{A5.9})$$

$$\text{where } P_3 = \frac{\tau^2}{2bc} \frac{g}{T_r} \frac{\partial \theta}{\partial z}$$

Equations (A5.8) and (A5.9) can be reduced to simpler forms if $P_i \ll 1$ ($i = 1, 2, 3$) is assumed. We then obtain:

$$\overline{uw} = -\frac{\overline{w^2}\tau}{c} \frac{\partial u}{\partial z} / (1 + \frac{2\tau^2}{d_1 c} \frac{g}{T_r} \frac{\partial \theta}{\partial z}), \quad (\text{A5.10})$$

and

$$\overline{wq} = - \frac{\overline{w^2} \tau}{c} \frac{\partial \theta}{\partial z} / (1 + \frac{\tau^2}{bc} \frac{g}{T_r} \frac{\partial \theta}{\partial z}) \quad (\text{A5.11})$$

Note the similarity of the expressions for the fluxes of sensible- and latent heat (A5.7) and (A5.11). This is in accordance with eqs. (5.23) and (5.24) and the assumption (5.13). In the next step only the buoyancy correction factors in (5.22) and (A5.10) are considered. Hence it is assumed that the influence of buoyancy on $\overline{w^2} \tau / c$ is small. Then we are left to show that:

$$\phi_m = 1 + \frac{2\tau^2}{cd_1} \frac{g}{T_r} \frac{\partial \theta}{\partial z} \quad (\text{A5.12})$$

Using eqs. (5.11) and (5.7), the left hand side of the above equation reads

$$\phi_m = (1 - 16 \frac{z}{L})^{-\frac{1}{4}} \approx 1 + 4 \frac{z}{L} = 1 + 4 z \frac{g}{T_r} k \frac{T_*}{u_*^2}$$

which after applying eq. (5.5) yields

$$\phi_m = 1 + 4 k^2 z^2 \frac{g}{T_r} \frac{1}{u_*^2} \frac{\partial \theta}{\partial z} \quad (\text{A5.13})$$

Finally, it is assumed that $\tau \approx \kappa z / u_*$. If eqs. (A5.13) and (A5.12) are substituted we obtain

$$1 + 4 \frac{k^2 z^2}{u_*^2} \frac{g}{T_r} \frac{\partial \theta}{\partial z} = 1 + \frac{2}{cd_1} \frac{\kappa^2 z^2}{u_*^2} \frac{g}{T_r} \frac{\partial \theta}{\partial z} \quad (\text{A5.14})$$

The assumption of the distribution of τ is of course not rigorous, as part of the buoyancy effect is still contained within the $\overline{w^2} \tau / c$ term. Attempts to solve also for $\overline{e^2}$ and $\overline{\epsilon}$ are frustrated by the fact that the turbulent transport terms of $\overline{u^2}$, $\overline{v^2}$ and $\overline{w^2}$ are not negligible, compared to the production and dissipation terms. This renders the whole set of equations analytically untractable.

Samenvatting

De bepaling van de verticale fluxdichtheden van impuls, voelbare- en latente warmte aan het aardoppervlak is van groot belang in diverse wetenschappen zoals bijvoorbeeld meteorologie, landbouwkunde en hydrologie. De fluxdichtheid van een grootheid wordt gedefinieerd als het transport van die grootheid per eenheid van oppervlak en per tijdseenheid.

Een directe meting van een fluxdichtheid aan het aardoppervlak is vaak lastig uit te voeren, daarom neemt men gewoonlijk z'n toevlucht tot metingen in de atmosfeer en in de bodem. Een veel toegepaste techniek is die welke gebruik maakt van zogenaamde "flux-profiel relaties". Deze relaties koppelen de verticale fluxdichtheid van bijvoorbeeld de voelbare warmte aan een verticaal temperatuurprofiel. Bij zo'n methode kan dan volstaan worden met een relatief eenvoudige meting van de temperatuur op 2 of meerdere hoogten. Echter, bij het gebruik van flux-profiel relaties gaat men er vanuit dat aan een aantal voorwaarden is voldaan. De belangrijkste voorwaarde is wel dat de fluxdichtheid onafhankelijk is van de hoogte, hetgeen alleen opgaat boven een uitgestrekt, homogeen terrein. Daar aan deze eis in de praktijk meestal niet (voldoende) voldaan is, rijst de vraag in hoeverre dit de meetresultaten beïnvloedt. Het doel van dit onderzoek is dan ook na te gaan hoe groot de fouten zijn die men maakt bij het gebruik van de standaard flux-profiel relaties boven een niet-homogeen terrein, en tevens na te gaan hoe die fouten samenhangen met de structuur van de atmosfeer boven een inhomogeen terrein.

In hoofdstuk 1 wordt als inleiding de bovenstaande vraagstelling gepresenteerd. Er wordt in aangegeven dat het onderzoek uitgevoerd zal worden door gebruik te maken van een numeriek 2e orde model. De numerieke aanpak in het algemeen wordt vergeleken met de traditionele keuze: theoretisch of experimenteel.

Een nadere uitwerking van de probleemstelling wordt gegeven in hoofdstuk 2. Hierin wordt aangegeven dat de studie betrekking heeft op het onderste deel van de atmosfeer: de atmosferische oppervlakte laag (ASL) en op welke manier de ASL zich ruwweg aanpast aan een terreinovergang. Het doel van dit onderzoek wordt daarna in dit kader geplaatst. In het tweede deel van hoofdstuk 2 worden de voor het probleem relevante vergelijkingen behandeld.

In hoofdstuk 3 wordt een indeling in diverse klassen gemaakt van de bestaande modellen die een beschrijving geven van het gedrag van de atmosferische oppervlaktelaag na een terreinovergang. De voor- en nadelen van de diverse modelklassen worden besproken. Het blijkt dat voor het huidige onderzoek de numerieke modellen met een tweede orde sluiting, kortweg 2e orde modellen, de meeste mogelijkheden bieden en dat van dit soort modellen tevens de beste resultaten verwacht mogen worden.

Het 2e orde model van Rao, Wyngaard en Coté (1974b) dat gebruikt zal worden, wordt in hoofdstuk 4 nader bekeken. Dit model bestaat uit een gedeelte (deel 1) dat de beginprofielen berekent van de diverse grootheden zoals windsnelheid, temperatuur, luchtvochtigheid enz. (initialisatie). Het andere gedeelte (deel 2) van dit model berekent daarna het gedrag (= de verdeling) van de diverse grootheden na een terreinovergang.

De beginprofielen en de balansen van diverse vergelijkingen, berekend met deel 1 worden vergeleken met experimenteel gevonden waarden en enige universele functies. Hieruit blijkt dat de randvoorwaarden aan de bovengrens van het model problemen opleveren in een niet-neutrale atmosfeer. Tevens wordt duidelijk dat de modellering (= benadering) van met name de druktermen in de diverse vergelijkingen te wensen overlaat. Een en ander resulteert in beginprofielen, bijvoorbeeld van de verticale gradiënt van de windsnelheid ($\partial U / \partial z$), die niet exact overeenkomen met de verwachte universele functies. In deel 2 van het model wordt als randvoorwaarde aan de ondergrens van het model o.a.

verondersteld dat de relatieve vochtigheid (R_{02}) constant is. Deze aanname levert een niet-realistische oplossing bij een overgang van een nat en koel terrein, naar een droog en warm terrein. Een methode om dit te verbeteren wordt aangegeven. Het uitvoerig bespreken van deze zwakke punten wil natuurlijk niet zeggen dat het model daarom onbruikbaar is. Een positief punt is dat de oplossing stabiel blijkt te zijn bij een tijdelijke verstoring van de evenwichtssituatie: alle variabelen keren weer terug naar hun oorspronkelijke toestand.

In hoofdstuk 5 wordt een samenvatting gegeven van de bestaande flux-profiel methoden. De aannamen die aan elk van die methoden ten grondslag liggen worden kort besproken. In de homogene (evenwichts)situatie worden deze methoden vergeleken met elkaar en met de zogenaamde "2e orde flux-profiel relaties". Dit laatste zijn flux-profiel relaties, welke afgeleid worden uit de transportvergelijkingen van het model. Het blijkt dat in een neutrale atmosfeer de fluxen, bepaald met beide soorten relaties, vrijwel identiek zijn. In een niet-neutrale atmosfeer is de interpretatie van de 2e orde relaties minder eenvoudig. Het blijkt dat stabiliteitseffecten op 2 manieren de 2e orde flux-profiel relaties beïnvloeden: via een extra term in de relatie en via de verandering van de zogenaamde "2e orde eddy diffusie constanten". Op 2 manieren wordt vervolgens nagegaan in hoeverre de 2e orde relaties consistent zijn met de standaard flux-profiel relaties. Het blijkt dat fluxen, bepaald met de Bowen verhouding methode zeer goed (binnen 5%) overeenkomen met de fluxen bepaald met de 2e orde relaties. Daarentegen blijkt dat de fluxen bepaald met de aerodynamische methode een veel slechtere overeenkomst vertonen. Dit laatste is een gevolg van de tekortkoming van het model, besproken in hoofdstuk 4.

De Bowen verhouding methode en de aerodynamische methode worden in hoofdstuk 6 toegepast op de profielen van windsnelheid, temperatuur en luchtvochtigheid welke berekend zijn met deel 2 van het model. In deze inhomogene situatie blijkt dat de Bowen verhouding methode warmtefluxen geeft die in absolute waarde

groter zijn dan de fluxen berekend met het model. Tenminste, dit is het geval in de zogenaamde aangepaste grenslaag (IAL), waarin de modelfluxen vrijwel onafhankelijk van de hoogte zijn. Het blijkt dat het verschil voornamelijk veroorzaakt wordt door de bijdragen van de advectionstermen en de turbulente transporttermen in de modelvergelijkingen. Met deze effecten wordt in de Bowen verhouding methode per definitie geen rekening gehouden. De verschillen zijn echter gering ($< 10\%$) indien het gebied vlak achter de terreinovergang (binnen 2 m.) buiten beschouwing wordt gelaten. Een bijzonder gunstige bijkomstigheid is overigens dat de Bowen verhouding methode zelfs een betere schatting van de warmtefluxen aan het aardoppervlak geeft dan de volledige modelvergelijkingen. Ver boven de IAL zijn de fluxen, bepaald met de Bowen verhouding methode, juist kleiner (in absolute zin) dan de fluxen bepaald met het model. Een mogelijke verklaring hiervoor wordt kort besproken.

De analyse van de aerodynamische methode, tenslotte, is zoals in hoofdstuk 5 werd opgemerkt, minder goed mogelijk door de afwijkingen in de beginprofielen. Hierdoor is de beschouwing van de resultaten van de aerodynamische methode beperkt en slechts kwalitatief. Het blijkt wel dat deze methode aanzienlijk grotere fouten geeft dan de Bowen verhouding methode. Dit wordt toegeschreven aan de grotere gevoeligheid van de aerodynamische methode voor de juiste hoogteverdeling van de diverse dimensieloze gradiënten.

In hoofdstuk 7, tenslotte, worden de resultaten van dit onderzoek samengevat, en wordt kort aangegeven wat in het algemeen de toepassingsmogelijkheden van het gebruikte model zijn.

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Curriculum vitae

Leonardus Johannes Maria Kroon werd op 5 januari 1952 geboren te Rotterdam. In deze stad doorliep hij de lagere- en middelbare school. In 1970 behaalde hij het diploma HBS-B aan het St. Franciscuscollege, en schreef hij zich in aan de Rijksuniversiteit te Utrecht. Het kandidaatsexamen natuurkunde, studierichting N3, legde hij af in 1973. Het doctoraaldiploma Geofysica, hoofdrichting Meteorologie, werd behaald in 1976 met als afstudeeronderwerp "Ladingsscheiding in onweerswolken" (prof. dr. F.H. Schmidt en dr. J.C.H. van der Hage). Tevens werd de eerstegraads onderwijsbevoegdheid in de wis- en natuurkunde behaald. Hierna volgde een periode tot 1980 waarin hij naast een aanstelling als wetenschappelijk medewerker werkzaam was als leraar natuurkunde in het middelbaar onderwijs. De wetenschappelijke werkzaamheden werden verricht op het Instituut voor Meteorologie en Fysische Oceanografie (IMOU) van de RUU en hadden betrekking op het verdampingsonderzoek en de ontwikkeling van geventileerde psychrometers. De leraarsfunctie werd achtereenvolgens vervuld bij SG. Spieringshoek te Schiedam (0.5 jaar), Het Nieuwe Lyceum te Bilthoven (1 jaar) en Het Nieuwe Lyceum te Hilversum (6 jaar). Door een zwak, maar overigens perfect getimed, moment van de toenmalige minister van defensie Vredeling, werd de mogelijkheid geschapen, en gebruikt, om legaal aan de "vaderlandsplicht" te ontsnappen. In 1980 volgde een aanstelling als promotieassistent bij de Vakgroep Natuur- en Weerkunde van de Landbouwhogeschool te Wageningen. Naast deze aanstelling bleef hij echter als leraar verbonden aan het middelbaar onderwijs, totdat hij in 1984 volledig in dienst trad bij de laatstgenoemde vakgroep als universitair docent.